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HEAVY LIFT HELICOPTER - CARGO HANDLING ATC PROGRAM. VOLUME II. FABRICATION OF TEST HARDWARE AND FIXTURES (INTEGRATED TEST RIG)

Joseph Shefrin, et al

Boeing Vertol Company

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Army Air Mobility Research and Development Laboratory

December 1974

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This report formally document	s the efforts	and results of the					
cargo handling system segment	of he Heavy	Lift Helicopter (HLH)					
Advanced Technology Component	(ATC) develo	pment program.					
The suppose of the HTH/ATC NE	ti mininina						
The purpose of the HLH/ATC was	is to minimize	technical, cost and					
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development, test and evaluation (RDTE) and production programs. This was achieved by design, fabrication, and testing of specific ATC hardware in three critical air vehicle subsystems:

- a. Rotor/Drive System
- b. Flight Control System
- c. Cargo Handling System

This report covers only the cargo handling system and consists of three volumes:

- Volume I Detail Design, Structural and Weights Analysis, and Static and Dynamic Load Analysis
- Volume II Fabrication of Test Hardware and Fixtures (Integrated Test Rig)
- Volume III Results of Tests, Inspections and Evaluations

Volume II contains the design criteria, physical description, stress analysis, fabrication and supporting data for the Integrated Test Rig which was used to conduct system testing of the full-scale cargo-handling ATC hardware.

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INTRODUCTION

Final design ATC hardware was based on subsystem design development testing. Verification of the cargo handling system ATC design goals, accomplished in advance of an aircraft installation, required a test fixture with which system performance could be demonstrated.

Since no suitable evaluation fixture existed, the ATC program included preparation of such a fixture. This fixture, the integrated test rig (ITR), was designed to permit: operation of the ATC developed hardware through all its functions; verification of performance and reliability characteristics, failure modes and effects, and maintenance characteristics; demonstration of 1800 hoisting cycles using both suspension modes at design load and speed and demonstration of maximum static load.

Although the ITR was erected on Boeing Vertol property, its design provided for dismantlement and recrection at another site. To support use at another location, appendixes to this document include foundation loads, descriptive drawings, air supply operating instructions, and an instrumentation calibration procedure.

GENERAL DESCRIPTION OF INTEGRATED TEST RIG (ITR)

MAJOR ELEMENTS OF TEST FIXTURE

The integrated test rig is shown in Figure 1. It consists of two steel I-beam towers, 14x14 feet square, supporting a pair of horizontal internally braced 30-inch-deep wide-flange I-beams with a 40-foot span and a 70-foot vertical clearance. Lateral outriggers buttress the towers.

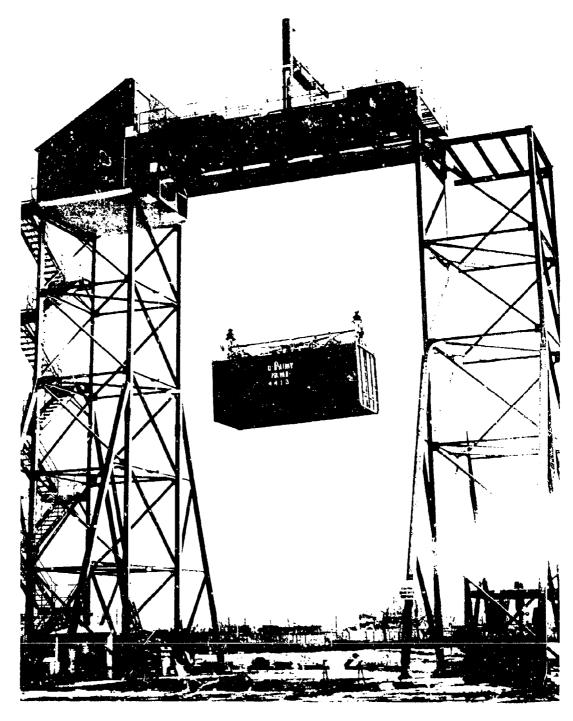
The major elements of the ITR are:

- Main structure consisting of towers and overhead section.
- 2. Footings and foundation with provisions for three ground tiedown points ("dead men").
- Pneumatic power generation (hoist drive air supply) including fuel supply.
- 4. _tility power and communication.
- 5. Control room and enclosure.
- 6. Instrumentation and data recording system.
- 7. Stairway to control room and work platform.
- 8. Provisions for utility hoist.
- 9. Site improvements including access roads.

FULL-SCALE STATIC SIMULATION - HLH INSTALLATION

The test rig simulates a full-scale installation of the cargo handling system (CHS) in the Heavy Lift Helicopter (HLH), Figures 2 and 3, in the following respects:

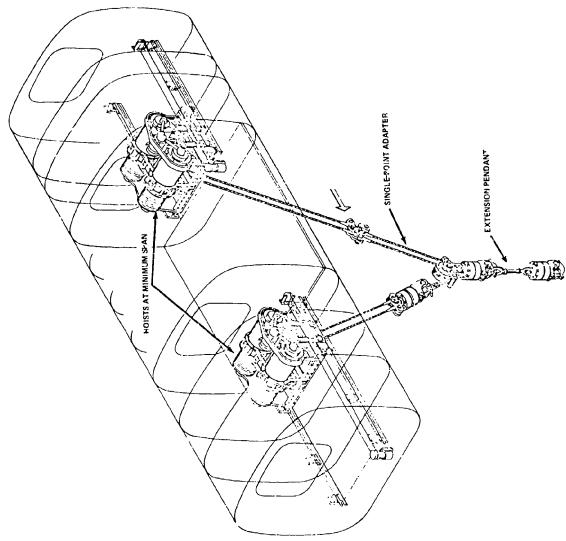
1. Two steel frames simulate the a/c mounting structure for the hoists and signal conductor reels. These are removable to permit complete ground buildup of individual hoists and associated assemblies including the wiring to form a "hoist module". The assembled "modules" are raised with the overhead utility hoist and installed between the main overhead and auxiliary horizontal I-beams. Each hoist module also mounts the span positioning equipment and has space provisions for hoist separations of 16, 22 and 26 feet.



中海出土公 安全人

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Figure 1. HLH Cargo Handling System Integrated Test Rig Hoisting - 29-Ton Load.



Sing c-Point Mode - Hoist Installation in HLH. Figure 2.

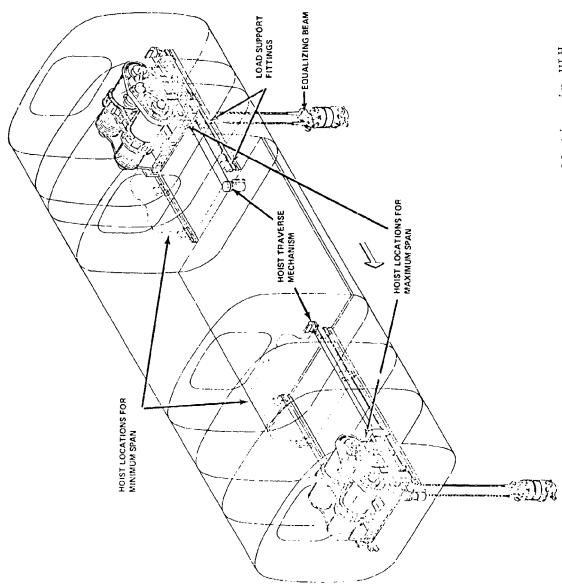


Figure 3. Dual-Point Mode - Hoist Installation in HLH.

- Power, control, geometry and physical spacing of hoist assemblies.
- 3. Fuselage level attitude.
- 4. Relative location, type of hoist controls and displays, and layout of the rear-facing, loadcontrolling-crewman (LCC) station.
- 5. A 50-foot cable deployment with full-scale, rigged loads.

RIG LOCATION

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The ITR is located at the Boeing Vertol Company facility, Ridley Park, Pennsylvania.

DESIGN REQUIREMENTS AND CRITERIA

HARDWARE COMPATIBILITY

The test rig design shall be compatible with the following full-scale cargo handling system hardware to be installed and operated in the test rig:

Hoist and tension member assembly Hoist drive unit Load isolators
Span positioning system
Signal conductor reels
Controls and displays systems
Coupling
Single-point adapters
Cable cutters
Pneumatic distribution system

and the following GFE (non-CHS) auxiliary equipment:

8' x 8' x 20' MILVAN Container handling d∈vice

SUSPENSION AND HOIST SPAN ARRANGEMENT

Poth single-point and two-point suspension arrangements shall be provided. Hoist span positioning space and functional provisions shall be incorporated for both 16- and 26-foot suspension spans.

DESIGN LOADS

Steady Vertical Loads

The individual hoists must support vertical loads and accommodate a 60/40 load split based on the two-point suspension. The loads to be supported are:

System design operating load = 28.0 tons Hoist design operating load (28×0.6) = 16.8 tons System maximum static load (28×2.5 g) = 70.0 tons Hoist maximum static load (28×2.5 g $\times0.6$) = 42.0 tons

Failure case (single hoist, any span position) = 70.0 tons

Transient Vertical Loads

Some transient vertical loads may be encountered under differential hoisting (load levelling). The normal load tilt (multipoint mode) will not exceed +15°. Load release transients are insignificant (loads above 1,000 lb cannot be released).

Side, Overturning, and End Loads

The hoists are "single-point payout", therefore, tension member loads shall always act through the longitudinal (fore-aft) centerline of the test rig. No side, end, or overturning loads are anticipated due to testing except as may be encountered from load swinging (±5°, est.) or wind. Load motion shall not result in the loads striking the rig structure.

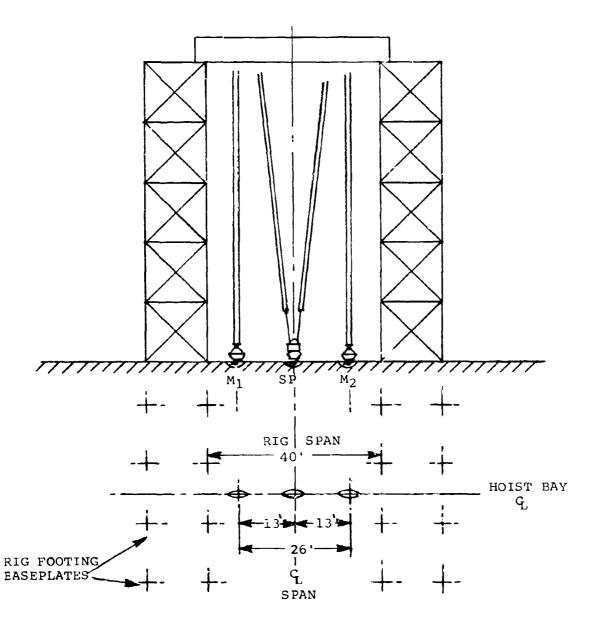
Maximum Static Load Provisions

Tiedown provisions in the ITR foundation are required for application of the maximum design static load to either individual or both hoists as shown in Figure 4.

The loading member, ring or bar, with a maximum 2-1/2 inch diameter, is to be compatible with CHS coupling. Space must be provided for coupling attachment/release.

FAILURE CRITERIA

Failure simulation involving severe load transients or high load release tests was not a requirement of the ATC program.



LOAD CAPACITIES: M_1 , $M_2 = 42$ Tons SP = 70 Tons

Figure 4. Multi-Point and Single-Point Static Loading Schematic.

Rig survival - no collapse - was provided in the design in the event of a sling or other failure under maximum static load. The vertical columns are the main support for the control room and are independent of the column struts in case of strut removal by impact of the test load (due to a suspension failure).

LOAD SIZE AND ACCESS

The standard test load size is an 8' x 8' x 20' container. A 40-foot clear span between overhead supports is required to provide vehicular clearance around the load for load placement and removal. Direct access is required along the fore-aft centerline and between the supports and superstructure of one tower to accommodate a MILVAN on a flatbed trailer.

LOAD LIFT

The test rig is to have a clear height of 70 feat.

HOISTING SPEED

Design hoisting and lowering speed (with load) is 60 fpm. Design payout speed (without load) is 120 fpm.

UTILITIES AND COMMUNICATION

The following provisions shall be made for utilities:

- 1. Equipment hoist 6,000 lb capacity
- 2. Intercom LCC station to ground and test areas
- 3. Storage area
- 4. Cooling water for PPG heat exchanger
- 5. Work platforms around hoists
- 6. Lightning protection
- 7. Aircraft warning signals
- 8. Weather protection for cargo system and PPG
- 9. Electrical requirements for the cargo system, pneumatic power generator, related instrumentation, general utility and equipment hoist, and the cargo handling device shall be as follows:

28V	DC	Direct current
5V	AC	Single-phase, 400 Hz
26V	AC	Single-phase, 400 Hz
115V	AC	Single-phase, 60 Hz, regulated
120/208V	AC	Three-phase, 60 Hz
120/280V	AC	Three-phase, 400 Hz

Compliance with MIL-STD-704 is required.

CONTROL ROOM

The control room shall house the LCC station, with a full view of the cargo in the ground and hoisted positions, the controls for the PPG and all related instrumentation. The floor loading criteria of 125-150 psf (light industrial) shall be used.

SAFETY CONSIDERATIONS

In addition to ITR failure criteria, the structure and location of operating equipment and utilities shall meet OSHA* requirements. The personnel stairway shall be remote from the test load.

*Williams - Stieger Industrial Safety, Cct. 1970.

TRANSPORTABILITY CONSIDERATIONS

The rig design shall incorporate provisions for dismantlement and recrection at another site.

PNEUMATIC FOWER GENERATOR

A pneumatic power generator (PPG), located at ground level, and air distribution system shall be used to supply air to the hoists at a pressure ratio of 4.3 and a temperature of 450°F. Operating controls, pressure regulation and safety devices are required.

DATA REQUIREMENTS

Recordings are to be made of the following parameters (as delineated in Table I):

Ambient conditions - temperature and pressure Supply air - temperature and pressure Air turbine operation - temperature, pressure and speed Control inputs and functions Tension member - load, angle, payout length Cargo system noise

DETAIL DESIGN

DESIGN CONSIDERATIONS

Selection of the ITR configuration was based on the following design considerations:

Control room requirement

Access to hoist test location Simulation of HLH LCC station Drinking water, but no lavatory or toilet facilities

TABLE I. ITR - T	TEST DATA AND	INSTRUMENTATION] [REQUIREMENTS	TENTS.		
Data Requested	Unit	Max. Range	No. of Regü. Trans- Accracy ducers	No. of Trans- ducers	Visual Oscil Monitor	Visual i	Hemo W.B. Scope FM.
PNEUMATIC POWER GENERATOR Supply Air Temp. Supply Air Pressure	°F PSIA	09-0	·: +	<i>ਜ</i> ਜ	××		
HOIST DRIVE Turbine Inlet Pressure Turbine Inlet Temp. Turbine Exhaust Temp. Turbine Motor RPM	PSIG °F °F +VDC	0-50 32-500 32-500 5	サー 十 + + か か か ぬ	4421	××××		
HOIST CABLE DATA Cable Tension Cable Angle-Longitudinal Cable Angle-Lateral Cable Payout Hoist RPM(Load Velocity)	ib DEG 30° DEG FT RPM	0-30000 30°Fwd/40°Aft 30 100 25	···+ +	40000		****	
CABLE LENGTH INCIDATOR Hoist Position (Fwd) Hoist Position (Aft)	FWD/MID/AFT FWD/MID/AFT	N/N N/A		00		××	
HOIST CONTROL Command Velocity	+VDC	ιń	+5%	2	×		
CARGO COUPLING Mech. Release Signal	+VDC	28	+2%	2			×
NEAR-& FAR-FIELD NOISE WIND VELOCITY AMBIENT AIR TEMPERATURE	db MPH °C	140 60 -10+40		ਜਿਸ ਜ		× ×	×

Span/overhead structure

Vehicle clearance for manipulating load Variable cargo hoist span Center mount for davit

3. Safety

OSHA standards Stairway remote from test load - single egress location

4. Outriggers

Wind loading requirement Access to test load through base of tower

5. Transportability

Possible use of ITR at another site required inclusion of splices and bolted joints at significant locations

6. Footings and foundation

Detail design based on known local rock formations determined by site borings.

CONFIGURATION DESCRIPTION

Because of height requirements of the towers and the relative low design loads, a trussed frame configuration was selected. Requirements for a control room made the selections of a 14foot-square tower desirable. Easy installation of stairways on this type of tower was also a factor in the selection. Outriggers were used on each leg of the tower in the inboardoutboard plane for two reasons:

- Wind loading, and
- Need to remove secondary bracing in lower portions of tower to permit a loaded truck to drive through the tower.

Outriggers in the fore-aft plane were not required because the criteria used were not factors in this plane. Also, horizontal components of test loads are reacted by four frames in the fore-aft plane, while in the inboard-outboard direction only the two inner frames will react the horizontal component of loading.

Highlights of the ITR design including layouts of the test fixture and equipment are discussed in the following text.

MATERIAL DESIGN CRITERIA

Structure

Structural steel - Commercial A36 (A1SC Specification)

Allowable 22,000 psi Yield 36,000 psi Ultimate 60,000 psi

Welding practice - AISC

Bolts - ASTM A325

Design loads - per requirement section.

Environmental factors - Wind - 30 psf (loaded)

50 psf (unloaded)

Snow - NE local standard

Safety factor - 1.5 (on maximum/failure load using material allowable.)

Footings and Foundation

Concrete - 3,000 psi compressive strength at

28 days - reinforced, continuous pour

Reinforcing steel - per ASTM A61

Anchor bolts - ASTM Spec A325

Footings - 3,000 psi concrete on bedrock

Anchor beams - to mate with Crosby-Laughlin round

pin shackles

TRADE-OFFS/COST REDUCTION

An alternative A-frame configuration for the structure was considered. It was rejected in favor of the selected design since higher column stiffness requirements would have resulted in a heavier design. Also, bolted and riveted construction was eliminated as more costly than the welded design used. Use of a 60-foot span to accommodate a 40-foot load length was also considered. This option was not implemented due to an estimated cost increase of approximately 20%.

MEMBER SIZING

41.

	Size	Type	Tp/lf.f
Main beams	30×14	WE	172
Main vertical supports (towe	rs) 8x8	WF	40
Tower bracing	(2) 4x3-1/2	Angles	9.1

TOWER STRUCTURES

Transportability

To facilitate disassembly and recrection, each of the tower columns included splices which consisted of a combination of welding and bolting at the joints, bolted connections at each of the four diagonal struts in the adjacent bay, and bolted joints at the outrigger struts.

Using this construction configuration, each tower could be dismantled into two discrete sections and the supporting outriggers could be disassembled for shipping without the necessity for welding cuts.

Stairways

Stairways were enclosed by 42-inch guardrails on both sides with a 32-inch handrail being incorporated on the inside guardrail. The handrail was located 3 inches into the 30-inch stairway.

The stairway was located entirely outside the outboard tower columns for safety considerations. This removed it as far as possible from the test load in the event of any failure in the suspension system.

Utility Connection to Tower Column

An outboard column was used as the terminating point for the utility connection from an adjacent building. The connection was made between the 28-foot and 42-foot levels with an attachment force of approximately 1,000 lb.

Warning Beacons

Beacon type lights normally required atop the structure for the protection of low-flying aircraft were not required for the site selected because of higher structures in the adjacent area.

OVERHEAD STRUCTURE

Overhead Beam Location Holes

Sales Sales Control of the Control

The tower/beam assembly was predrilled to simplify assembly. The tower/beam holes are close fitting. The hoist module attachment was also predrilled.

Slots were not provided for temperature changes as estimated changes were small.

Camber was not specified in the overhead beams, since only 0.7 inch beam deflection was anticipated at the 70-ton static load.

Work Platform

Grip strut, an expanded-metal, nonslip surface, was used for the open work platforms around the hoists. A uniform floor loading of 235 psf was used.

Davit Assembly

The davit assembly or jib crane reach was 12 feet. It had a 6,000-lb hoisting capability. This satisfied the need for lifting the hoist modules and other equipment from the ground, over the side, or through the hoist location to the overhead structure.

The hoist cable reach was 89 feet. Maximum hoisting rate was 40 fpm. Guardrails surrounding the hoist module locations had sections which were removable for hoist installation and removal by the over-the-side method.

HOIST ASSEMBLY MODULES

The hoist module consisted of a simple welded framework and lift points including the hoist main support fittings and hoist positioning tracks. The function of the module was to facilitate preinstallation and checkout of each hoist assembly on the ground (similar to engine quick prep-pack-ages). The module weight was minimized to keep the davit and utility hoist size down. Web material was 1-inch and flanges were .75-inch plate.

Each hoist was removable from its module. Both modules were identical and provided a hoist span positioning travel of 60 inches.

Hoist Shelters

A portable shelter (tarpaulin) was provided for each hoist test location. Each section was removable during hoist testing.

CONTROL-ROOM OPERATOR'S STATION

The control-room operator's station eye level duplicated the HLH LCC position as shown on Figure 5.

The control room floor area was approximately 130 sq ft. A solid, precast-concrete, nonslip floor was used with a design

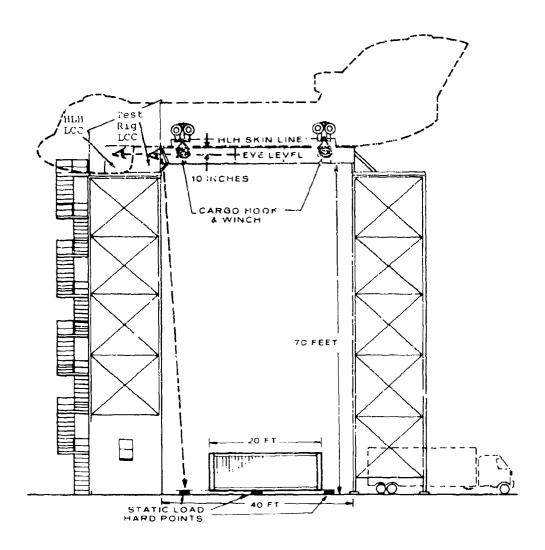


Figure 5. Test Rig, Showing Load-Controlling Crewman's View of Cargo.

floor loading of 172 psf.

LCC load viewing windows, looking aft and downward, were constructed of high-impact-strength polycarbonate (Lexan). A removable protective bar was installed across the main window. Side observation windows were of thermopane.

Except for doors and windows, the enclosure was fully insulated. Heat was furnished by a 3-kw thermostatically controlled space unit. A 5,000-Btu window-mounting airconditioner was provided for summer operation.

SITE LOCATION AND FOUNDATION

The test rig site location was selected to make maximum use of local rock foundations. To minimize service roads, the test rig was oriented adjacent to existing roadways.

The service road included a "Y" with one branch through the base of one tower and the other parallel to the overhead structure for truck pickup or deposit of a load within the reach of the jib crane.

The service road also provided for servicing of the fuel tank located 100 feet from the PPG unit.

Underground static load points in the concrete surface between the towers, along the hoist centerline, provided a 70-ton loading capability at the center point and 42 tons at each of the multi-point loading points. Slots were provided for entry of the cargo hooks, and covers were used to allow the surface traffic over these points.

The working surface around the test rig was asphalt covered, for vehicular traffic and surface drainage.

STRESS ANALYSIS

A stress analysis was performed to substantiate the design strength of the towers and overhead structure, using a Boeing Watfor "plain frame" computer program. An analysis of the hoist module frames and foundation reaction loads was also made. Analysis details are provided as Appendix I.

UTILITIES AND COMMUNICATION

Primary electrical service to the ITR was 277/480 volts AC 3Ø 60 Hz supplied from an adjacent building bus via a utility pole. Four distribution panels were provided in the control room: 480-volt, lighting, regulated-power, and 28-VDC.

Raceways and conduit were provided between the control room, the work platform and the PPG location.

Specific power requirements were serviced from the following sources:

115-Volt Regulator - Stabiline EMT 4115

15 KVA

Input: 95-135V Output: 115V 50/60 cy. 10

110-120V adjustable

Long-range amps: 130.0

400-Hz El ctronic Inverter - Sorensen FCD 3P1000

1000 VA - 3 Ø

Input: 208/230V 48 to 65 cy 3 Ø Output volts: ±10% 115/200V

3 Ø 4 wire 115V L-L 3 Ø 3 wire

Output volt reg: +1%Output freq.: 360-440 cy +1%Load range: 0-1000 VA

Distortion unity PF load: (360-400 CPS) 3%

28VDC - Instrumentation

Solid-state power supply - Sorensen MA28-125

Output:

Volts (DC)	Current (AMPS)	Reg.	Metered
18-36	0-125	.2%	Yes

28VDC - PPG Starting System

4-12VDC heavy duty truck batteries

Communication provisions included:

- Telephone linking the control room, plant switchboard and base of tower.
- Powered intercom system including four headsets, two keyed and two noise-cancelling microphones, and five interconnect boxes.

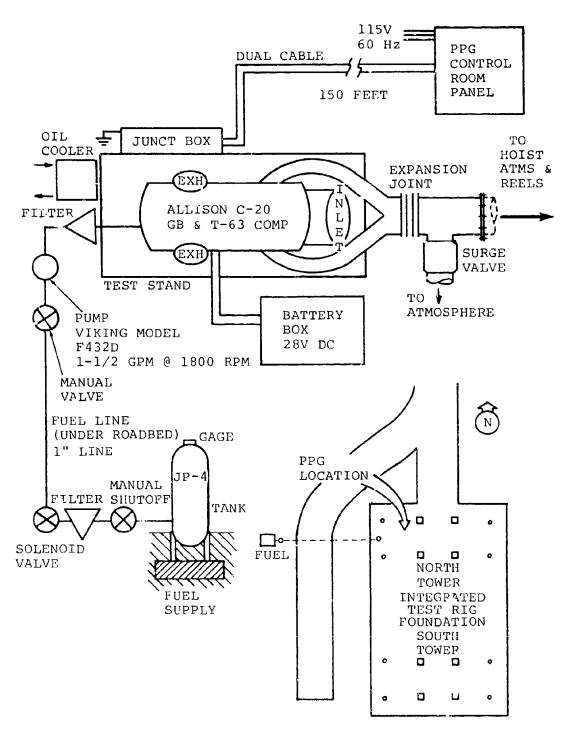
PNEUMATIC POWER GENERATOR (PPG)

Four possible PPG locations were considered: two on the overhead structure and two on the ground level. The two locations at the top of the structure were eliminated due to the additional structure required. In comparing the ground locations - the bay below the control room and the ground bay of the opposite tower - the latter was selected for the following reasons:

- 1. Elimination of:
 - a. fire hazard in vicinity of stairway for operating personnel.
 - b. the need for a fireproof structure around the air supply unit and special ducting, and
 - c. the need for an alternate stairway or secondary means of egress.
- 2. Simplification of PPG fuel system.
- 3. Ease of servicing and isolation from personnel.
- 4. Provision of direct view of PPG from upper-level control room during operation.

The pneumatic power generator unit (air supply) was selfcontained and consisted of the following components described in the system schematic, Figure 6:

- 250-C20 Allison engine rated @ 294 HP at sea level 95°F, serving as a power unit. Additional components include a starter/generator, voltage regulator, 24VDC battery pack, fuel boost pump, oil reservoir, and oil/water heat exchanger.
- 2. A T-63 compressor with a modified T-63 gearbox.
- 3. An interface gearbox (step-up) and power transfer shaft as shown in the overall gearing arrangement, Figure 7.
- 4. A surge control valve designed to prevent compressor surge under all operating ambients from zero to maximum capacity of the compressor. The valve was controlled by a vacuum switch located in the compressor inlet bell mouth.
- 5. A pressure controller compatible with the cargo system pneumatic power distribution system. This valve was controlled by pressure sensed in the header upstream of the hoists.



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Figure 6. Schematic - HLH/ATC Pneumatic Power Generator (PPG) and Generator Location.

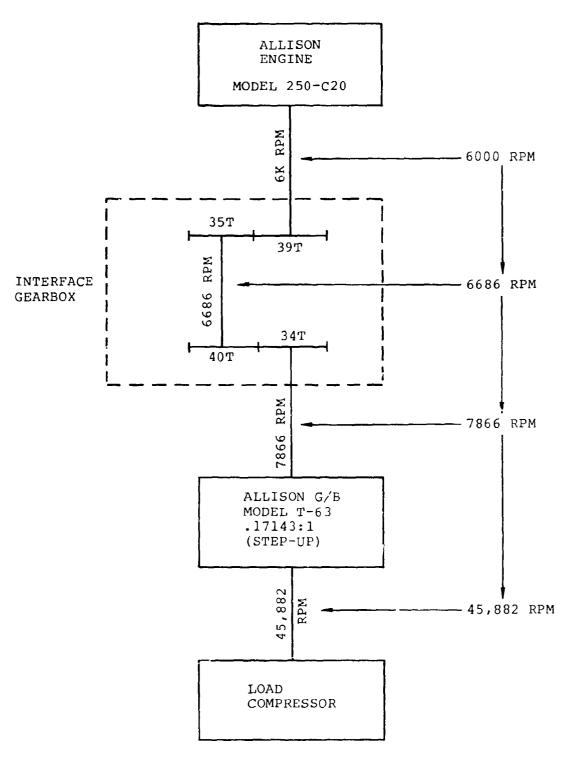


Figure 7. PPG Gearing Arrangement.

- 6. PPG control panel located adjacent to the LCC station, which included engine/compressor gearbox speeds, temperature and pressure instrumentation, fuel and battery charger controls, and emergency shutdown devices.
- 7. A 275-gallon fuel tank with manual and emergency valving with spark-arresting provision.
- Approximate pressure gages and gearbox chip warning sensors and 150-foot control cable.

Air Distribution System

The air distribution system, constructed of Schedule 5 304 scainless steel, was rated for 60 psi at 543°F. The system consisted of:

1.	Main riser	6-inch I D
2.	Main header	6-inch, upstream of both hoists
		3-inch, feeder between hoists
3.	Supply risers	2-1/2-inch flexible ducts
4.	Insulation	3-1/2-inch, with weatherproofed
		alumınum jacket
5.	Dump valve	2-1/2 globe, manually operated
	-	(for start and warmup control)
6.	Drain valve	for removal of line condensation.

A total pressure pickup and a thermocouple (I/C junction) were included in the 6-inch header to measure air supply conditions.

INSTRUMENTATION

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The ITR instrumentation system was specifically designed to record and display the parameters necessary to evaluate the function and performance of the HLH cargo handling system. The parameters recorded and displayed were in addition to the load-controlling crewman station monitoring equipment.

An instrumentation console located in the ITR control room contained the equipment necessary to monitor and record the temperatures, pressures, isolator loads, cable angles and other auxiliary signals.

A listing of the instrumentation equipment used is shown in Table II. For wiring and other detail, refer to the Instrumentation Family Tree ST30861 listed under Design Layouts and shown in Appendix II.

TABLE II. COM	COMPONENT L	LISTING, INSTRUMENTATION	SYSTEM	- HLH CARGO HANDLING	ANDLING SYSTEM.
Equipment	Mode1	Characteristics	Function	Warning	Comments
Temp.Indicator, Love Controls (7ea	100-818	Range 0-600 Deg T/C Type "J" (I/C) Accuracy +1% of scale Recorder output 0-1V	Indicates exhaust temp; supply air temp & hoisting & rev.valve temp.	No Au usi re	Auxiliary output used to drive recording galvo.
Temp.Indicator Love Controls(lea)	Same)	Same	Indicates ATM temp.for operator monitoring	No Oper sele ATM	Operates thru selector switch for ATM temp.
Temp.Selector Switch Thermo- Electric(lea)	33112	No.channels-12	Switches ATM T/C inputs	1	I
Temp.Probe; Thermo-Elec.(lca)	5J2120L	Accuracy ±0.5% Probe Length 6" Type Iron- Constantan	Temp transducer for temp indicators	١ يو	1
Pressure Ind. Dynisco (Sea)	ER466A2	Range 0-100 PSI Accuracy +18 full scale Response time 1.5 sec Aux. output 0-1 Volt	Indicates supply air, hoisting & rev. air pressures	X e s	Auxiliary output used to drive recording galvo, conditions & amplifies pressure transducer signal.
Pressure Transducer (5ea)	103	Range 0-75 PSI, 112.5 PSIA Max.	Press, transducer for press, indicators	- ors	١
Load Isolator Panel	ST30869	Range 0-100KIP Accuracy +0.1% of input	Indicates compression load in each of four load isolators	0 Z	Receives load signal & ref.from cable tension & angle interface electronics

		TABLE II. COM	CONTINUED.		
Equipment	Model	Characteristics	Function	Warning	Comments
Cable Angle Ref. Panel	ST30873	Range 0-100 Deg. Accuracy ±0.1%	Indicates angle of four-hook cables	ON	Same as above
Closed-Circuit TV System Hitachi Lcd.	ST30852	Rango-50 yards Field of view 3ft x 3ft	Used to observe moving parts on cargo handling modules & container position	O N	1
Intercom System	ST30859	No.of stations-4 Type-Headset with mike	Used for communi- cation between the operator & crewmen	1	1
Oscillograph Recorder CEC(3ea)	5-124	No.channels-18 Speeds 0.25,1,4, 16,0- IPS	Used as graphic recording devices	N O	1
Pressure Probe United Sensor	PTC12	Adjustable immersion length Max.operating	Static pressurc transmitter	0 Z	-
	PTC8	Same as above	Total pressure	No	
Galvo CEC(2lea)	7-315	Frequency response 0-60 Hz (+2%)Sensitivity 12.2 UA/IN Impedance 26 Ohms	transmitter (Used as trans- ducers for graphic recording oscillo- graph)	0 Z	See dwg.ST30854A for recorder parameter info.
Galvo CEC(2ea)	7-319	Frequency response 0-350 Hz (+2%)Sensitivity 426 UA/IN Impedance 26 Ohms	Same as above	1	Same as above

		TABLE II. COM	CONCLUDED.		
Equipment	Model	Characteristics	Function	Warning	Comments
Galvo CEC (7ea)	7-351	Frequency response 0-12 Hz (+2%)Sensitivity 2.66 UA/IN Impedance 33 Ohms	Same as above	1	Used to record outputs of temp. indicators-see ST30854A.
Oscillator H-P (lea)	200CDR	Range 0-600 kHz Output 0-40 Volts	Used as time event marker source	ı	,
Amplifier Burr- Brown (6ea)	2088/16	Frequency range 0-10 kHz Gain ADJ 1-1000 CMR ADJ 40-140 DB	Uscd as buffer amplifier-see ST30860	N O N	1
D.C.Null Meter	413A/R	Range ImV-1000 V Accuracy +0.1% of input	Used as an auxil- iary monitor	O N	
Power Supply Kepco	ABC 6.0.75M	Range 0-10V 0-0.75A	Used as a calibra- tion voitage source	o No	
Load Cell Bud (2ea)	T3P2B	Range 0-50K lb Sens 3mV/V	Used to measure hook load	o O N	1
Meter Panel Simpson	ŧ	Range 0-50mA Accuracy +1%	Used to indicate cable speed	O N O N O N O N O N O N O N O N O N O N	Receives cable length signals from hoist interface electronics.
Anemometer-Taylor Instrument Co. Rochester,N.Y.	1	Dual range-zero adjust 0-25 mph 0-100 mph	Used to indicate wind velocity & direction	NO	

The instrumentation system was configured, as shown on Figure 8, in a two-bay console designated rack "A" and rack "B". Rack "A" included the equipment to monitor and record hoisting and reversing air turbine motor (ATM) temperatures, exhaust temperatures, main supply air temperature, hoisting and reversing air pressures, supply pressure, isolator loads, and cable angles. The recording oscillograph, closed-circuit television (CCTV) system switch panel, +15 volt power supply, event oscillator, and intercom station were also located in rack "A". The CCTV monitor, DC null voltmeter, hoist cont ol unit, hoist interface electronics, cable tension and length interface electronics, recording oscillograph and buffer amplifiers were located in rack "B". The data recording format is shown in Table III. Additional equipment added to the system included a recording oscillograph "C" used for recording cable tensions, two null panel meters for observing cable payout speed, an oscillograph control box with event button, a brake release counter, eight chip detectors, and a 12-channel temperature panel with selector switch. During initial checkout, supply pressure and antisurge valve characteristics were investigated. An x-y plotter, B&F conditioner with amplifier, and a dekabox were used to plot supply pressure.

Temperature Instrumentation

Temperature probes were inserted into the air ducts of each hoist drive unit to measure the turbine inlet, exhaust and supply air temperatures. They acted as transducers for the seven temperature indicators in the instrumentation system.

Evaluation of the hoist duty cycle required monitoring of the surface temperatures of the hoist main bevel gear casing, the secondary forward and aft bevel gear casings, the hoist brake casing, the ATM oil pressure line, and the ATM oil return line of each hoisting module. Silversoldered T/C junctions were attached to each of the six surfaces with Caulk grip cement (dental cement) and routed to a 12-channel T/C selector switch via a six-lead thermocouple cable. The selected channel temperature was displayed on a separate indicator. The selector switch and temperature indicator were located on a panel which was positioned in front of the hoist operator.

Pressure Instrumentation

Pressure probes were inserted into the air duct at the same location as the temperature thermocouples in the ATM inlet and were connected by flexible tubing to the pressure transducers.

Figure 8. Instrumentation wack Compole ST30855-1.

ITH INSTRUMENTATION SYSTEM - RECORDED DATA FORMAT.	Recorder B Recorder C	Measurement Chnl.	ng,°F 1 P2-Fwd,hoisting,psig Loca Isolators 2 P4-Aft,hoisting,psig	3 Al-Fwd, turbine speed 1 Fwd hoist, 4 Blank 2 Fwd hoist	5 A2-Fwd, cable payout 3 Aft	., °F 6 A5-Aft, turbine speed 4 Aft hoist,	7 Blank	°F 8 A6-Aft, cable payout	6 4°1;	psia 10	psig 11	sw. 12	.w. 13	psig		sw. 16 -	w. 17 Syn		
TABLE III. ITR INSTRUM	Recorder A	Measurement Chnl. Parameter Chnl.	1 Tl-Fwd,hoisting,°F 1 2 Time/event 2	T2-Fwd,reverse,°F T3-Aft,hoisting.°F	T4-Aft, reverse, °F	T5-rwd, exhaust, °F	Blank	T6-Supply air, °F	T7-Aft, exhaust, °F	Pl-Supply air, psia	P5-Aft, reverse, psig	Aft, torque sw.	Aft, brake sw.	P3-Fwd,reverse,psig	Blank	Fwd, torque sw.	Fwd, brake sw.	Blank	
		บี				_					=		<u>~</u>						

Load Isolator Instrumentation (Cable Tension)

A load cell for axial load sensing was an integral part of each load isolator. The output from the 350-ohm strain gage bridge on the cell was amplified and conditioned by the hoist cable tension and cable length interface electronics.

Cable Tension and Load Instrumentation

The load isolator reference panel located in rack "A" had a five-position rotary selector switch, four calibration adjustment potentiometers and a three-place digital panel meter which displayed the selected axial load.

An auxiliary method of measuring cable tension was employed using two load cells in series with each coupling and a BLH universal percentage indicator.

Cable Angle Instrumentation

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The cable angle sensing mechanism employed two linear transformers (see Figure 9) which produced electrical signals proportional to the longitudinal and/or lateral angles that the cable made with the vertical centerline of the hoist.

The cable angle reference panel had a five-position selector switch, four angle adjustment potentiometer, and a digital panel meter for readout.

Cable Payout Instrumentation

The cable length sensing mechanism (see Figure 10) employed a linear transformer whose electrical output signal was a function of the number of turns of the cable drum from the cable up (stow) position.

Two cable length signals, one for each hoist assembly, were recorded. By relating the cable payout trace deflection to oscillograph paper speed, a cable payout rate or load velocity could be determined.

Hoist Input Command Instrumentation

The hoist operator's control grip thumb switch supplied a command signal to the hoist controller logic. The command signal, conditioned to a 0-5VDC output for 0-100% command and interfaced with other controller network logic, determined the hoisting or lowering rate. In the single hoist mode, each command signal was recorded on rack "B" oscillograph.

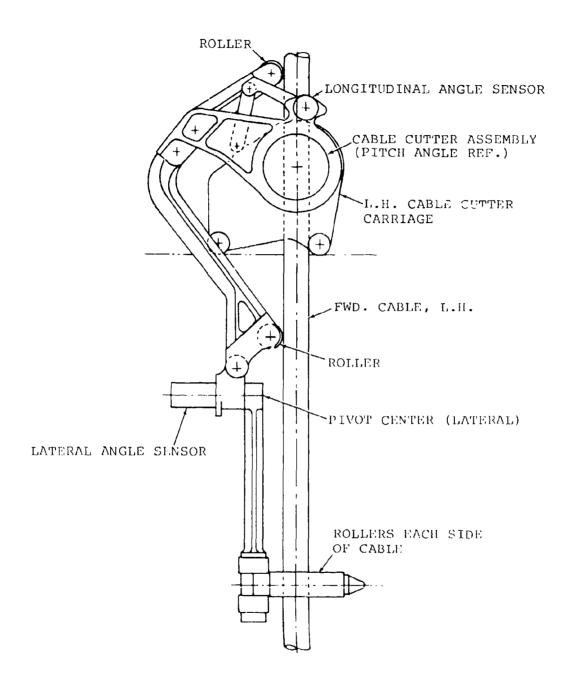


Figure 9. Cable Angle Sensor Mechanism.

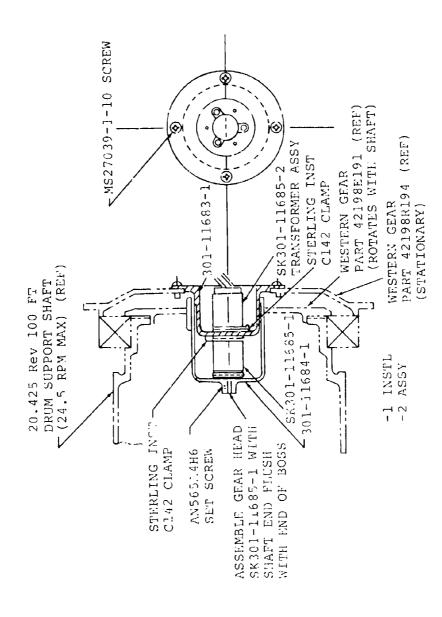


Figure 10. Cable Payout Sensor •

In the sync mode, only the forward hoist command signal was recorded since the aft hoist unit was commanded from the forward grip thumb switch.

Cargo Coupling Instrumentation (Mechanical Release Signal)

The hoist operator's cargo system control panel mechanical release switch was used for recording the command signal to the coupling solenoids and was recorded on rack "B" oscillograph.

Cable Payout Speed Instrumentation

The cable payout was derived from a sensing mechanism employing two magnetic pickups (MPU) which sensed ATM shaft speed. The ATM shaft speed was directly related to cable payout speed by the system gearing as represented in Table IV.

The MPU analog signal was conditioned and recorded on the rack "B" oscillograph.

Cable Speed Instrumentation

The payout speed signal was also fed to two zero center panel meters directly in front of the hoist operator. This provided the operator with a visual indication of cable payout speed and rate change or load velocity.

Power Requirements

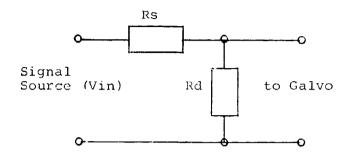
The instrumentation system required 115 VAC $50-60~\mathrm{Hz}$ and 28 VDC power supplies.

Recording Instrumentation

The recording instrumentation consisted of the galvanometer input network, the galvanometer and the recording oscillograph which housed the galvos. The galvo input networks were as shown in Figure 11. The networks were installed on the terminal boards on the junction panel located in rack "A". All the recorded signals from each source were terminated in the same manner with the matching networks located on the rack "A" junction panel.

The recording galvanometers selected for pressure, rpm and modulating valve measurements had a minimum apper frequency response limit of 200 Hz to permit observation of oscillatory, fluctuating or step input signals.

1	ATM SHAFT SPEI SPEED BY SYSTI	ED RELATED TO CA	ABLE PAYOUT
Cable Speed (fpm)	Shaft Speed (rpm)	MPU Frequency (Hz)	Speed Signal (VDC)
120	8,000	5,250	5.00
60	4,000	2,625	2.50
30	2,000	1,313	1.25
15	1,000	656	0.625
7.5	500	328	0.312
3.75	250	164	0.156



Resistor values were derived from the following formula:

$$Rs = \frac{Vin}{D \ Ig} \qquad where$$

Rs = calculated source resistance

Vin = input voltage

D = desired deflection in inches; usually 2.0

Ig = galvo undamped D-C sensitivity in UA or MA/IN.

(see manufacturer's specs)

Rd = required damping resistance, from manufacturer's
 spec sheet

Figure 11. Galvo Input Networks.

Galvanometers with a lower frequency response limit were used for the temperature, cable payout and command signal instrumentation.

Six Burr-Brown amplifiers located in rack "B" were used as buffer amplifiers to isolate those signal sources which did not have sufficient current magnitude to drive the recording galvanometers directly.

A 5-Hz signal was fed to one recording channel on each oscillograph to provide an accurate time/data correlation. The signal was in series with an event switch which served to correlate the start time on each of the three recording oscillographs. The event switch was located at the LCC station, with two recorder "on" lights and two recorder start switches.

Closed-Circuit TV System (CCTV)

A CCTV system was installed to monitor the moving parts on the hoist assembly during system testing. The system consisted of a monitor, a switch panel, and a camera enclosed in an environmental housing. The switch panel was located in rack "A", while the monitor was at the top of rack "B". The camera zoom lens is manually adjustable for the desired field of view.

Intercom System

A four-station intercom system was provided to enable the LCC station to have constant communications with three other stations. They were:

- 1. Ground crewman station
- 2. Rack "B" instrumentation monitor
- 3. Hoist assembly station

Each station was fitted with a dynamic mike and headset combination.

Calibration Procedures

Each system was calibrated daily prior to PPG startup and post-calibrated when the days testing has been accomplished, as described in Appendix IV. Periodic calibrations were in compliance with MIL-C-45662. All instrumentation calibrations were traceable to the National Bureau of Standards.

Container Weighing Instrumentation

The weighing instrumentation for the loaded MILVAN container consisted of two 50,000-lb Baldwin-Lima-Hamilton load cells and a Baldwin-Lima-Hamilton universal percentage indicator. The cargo handling system was in the two-point suspension configuration with one load cell in series with each hook. The SR4 universal percentage indicator was used to measure the individual loads. Connections to the load cells were made using two 70-foot eight-conductor cables from the load cells up to the test rig control room. The measured load using the load cells was within 0.7% of the calculated load.

Maximum Load Test Instrumentation

Three of the load isolator transducers from the hoist units were used for the maximum static load demonstration test. The calibrated transducers were used as the main load-sensing elements during the test.

The isolator load signals which represented the hook load were conditioned and recorded on the CEC recording oscillograph, "C". The conditioned load signal was also displayed for the LCC on a digital voltmeter which indicated the load directly in pounds. The equipment is shown schematically in Figure 12.

DESIGN LAYOUTS

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The integrated test rig is described by the engineering drawings given in Appendix II. Besides the basic ITR structural design, additional subsystems were required for the assembly, installation, functional operation and test of the cargo system. Each of the required design items is identified on the following drawings:

System Test Drawing Tree - SK301-11676 (See Figure 65 in Appendix II)

Hoist installation, including
Hoist module
Span positioning
Control system wiring
Assembly/handling fixtures
LCC station platform
Miscellaneous test fixtures and wiring

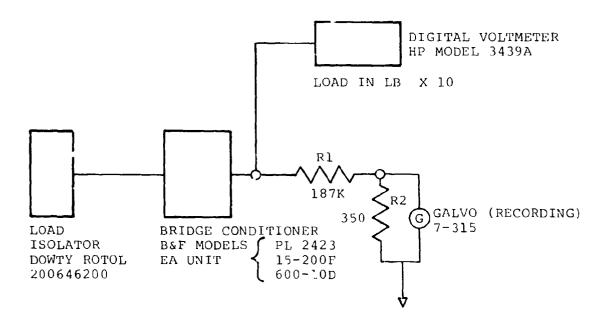


Figure 12. Maximum Load Test Network.

System Wiring - SK301-11694 (See Figure 68 in Appendix II)

A schematic showing the electrical integration of the cargo handling system is shown in Figure 67.

FABRICATION

STRUCTURE PREFABRICATION

The following items were prefabricated and yard assembled prior to delivery to the test site:

- 1. Fore-aft frames
- 2. Control room enclosure
- 3. Hoist modules
- 4. Overhead beams
- 5. Overhead lateral beams and davit supports
- 6. Control room floor
- 7. Stairway sections
- 8. Outriggers and frame bracing
- 9. Footing pads

FOUNDATION

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Excavations exact site location, and footing designs were adjusted to utilize available rock strata. Figures 13 and 14 show the excavation, location of bedrock, and foundation preparation. Figure 15 shows the ground tiedowns provided in the foundation.

The rig was erected with prefabricated fore-aft frames joined at the splices and supported by outriggers. Inboard and outboard frames were assembled between the main columns on alignment pins and welded in place.

The main overhead beams were drilled and bolted on assembly. The lateral beam components and davit supports were then welded in place. Figures 16 through 26 illustrate the assembly method and main features of the structure. The control room enclosure assembly, Figure 27, was hoisted in place and joined to the main column pads. Figure 28 shows the ITR after complete erection, including the air distribution duct. For reassembly at another site, the towers (split into four units at the aplice joints), the overhead structure disassembled to its basic components, the davit, outriggers, and control room shelter can be shipped as individual items. Table V delineates weight breakdown of the ITR structural assemblies.



Figure 13. Excavation for ITR Foundation.

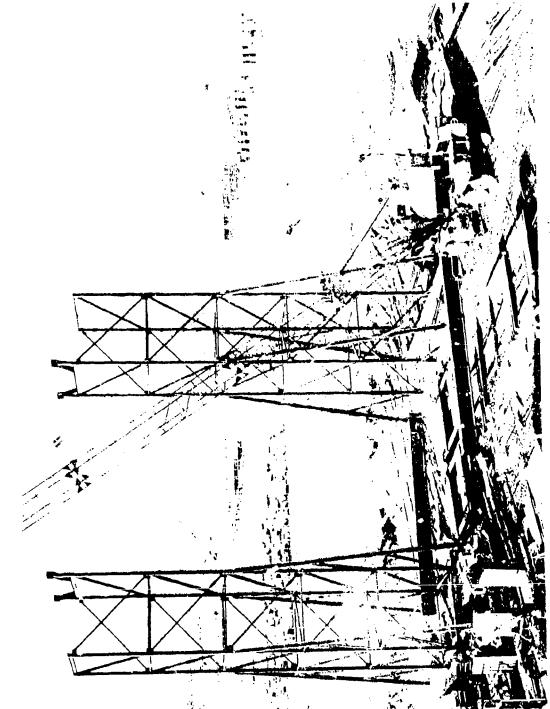


Figure 14. Preparation of Footings and Foundation.

Figure 15. Three Ground Tiedown Points in Foundation.

Component	Rails Frame Struts Modules Davit Mast Davit Matt	
Number	.00.	
Component	Lower Inboard-Outboard Frames Upper Inboard-Outboard Frames Overhead Beams Work Platforms Outrigger Members Stairways and Landings	
a remnii		

kig Components and Raising of Lower Main Frames. Figure 16.



Ξ,

Figure 17. Complete Tower Frames Erection.

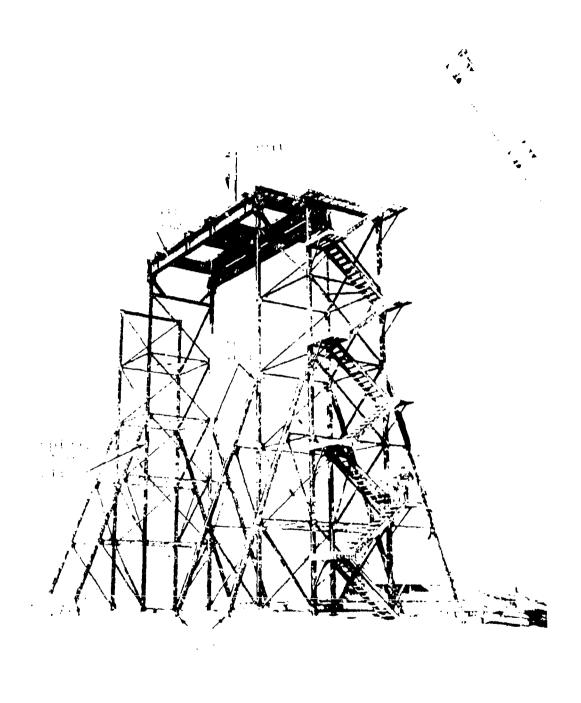


Figure 18. Overhead and Stairway Installation.

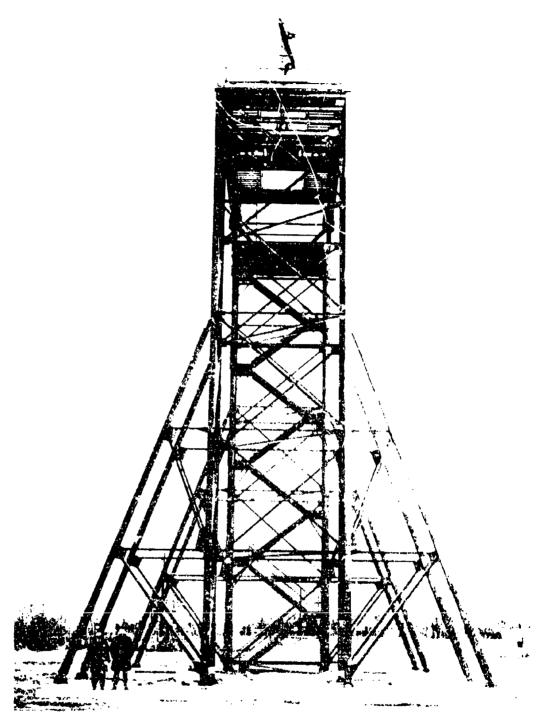


Figure 19. And Now Strain Court,

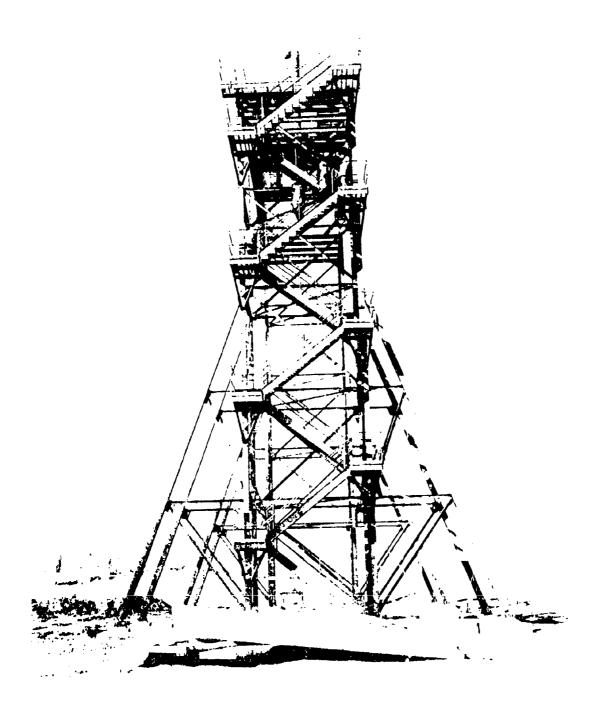
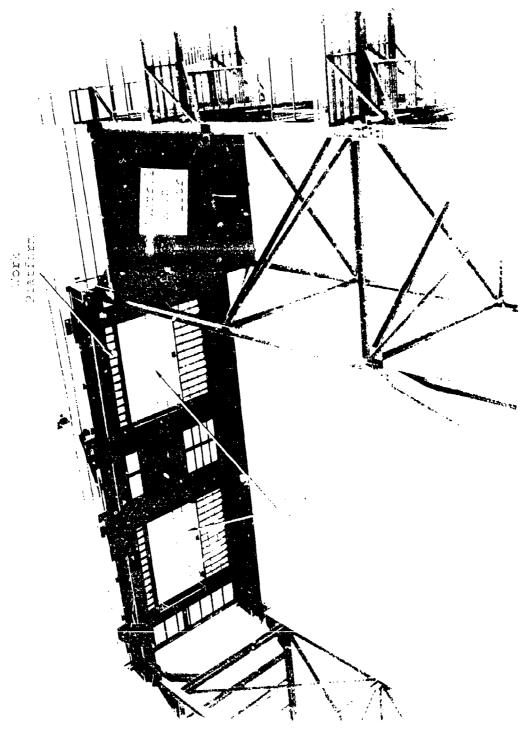


Figure 20. And Wiew of Forward Tower.



Under Tiew of Overnead Structure and Hoist Module Location. Figure 21.

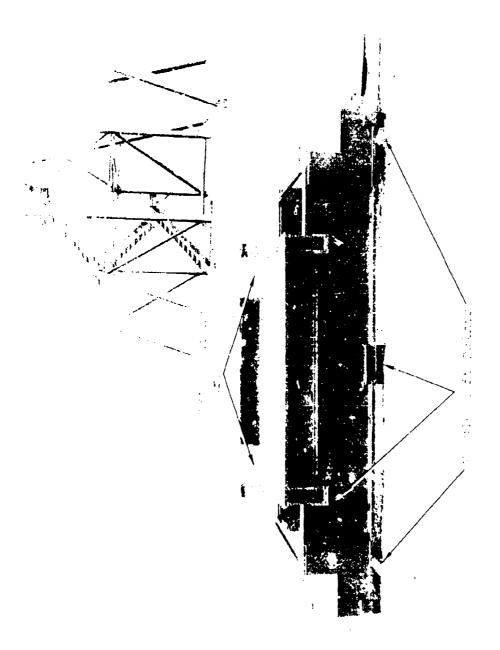


Figure 22. Noist Modules (Inverted).

Figure 23. Control Room Area - Top of Forward Tower.

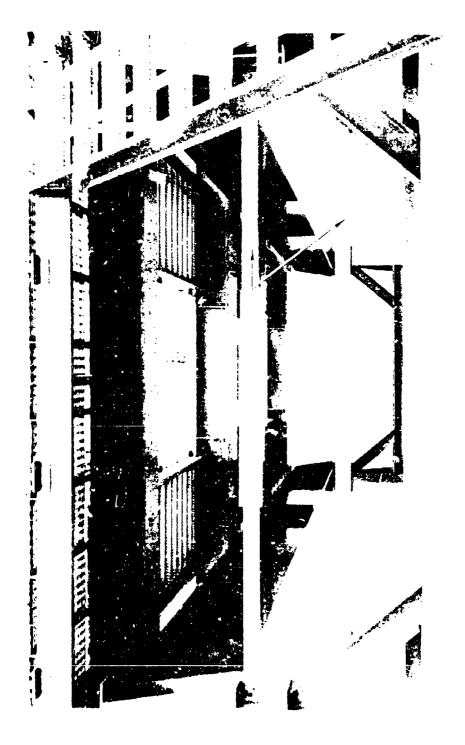


Figure 24. View of Overhead Section From LCC Position; Forward Tower.

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Pigure 25. Work Platform intrazio familia Cortari, Control Room Location.

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Figure 26. Completed ITR Framework on Foundation.

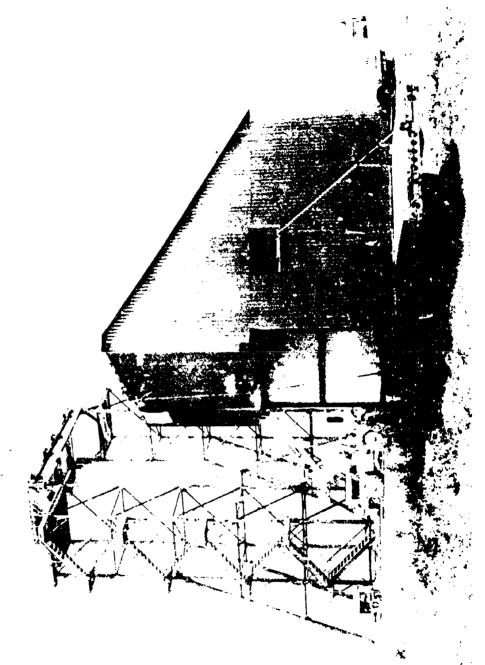


Figure 27. Control Room Enclosure Assembly.



Adrial Tlow Of Complete Structure and Hot Air Distribution Duct.

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数

HOIST MODULE INSTALLATION

Each module was hoisted over the side of the ITR work platform from a truckbed using the overhead davit and a short-reach, adjustable-cg sling, ST40972. The side and module handrails were removed at each module location. Hoisting clearance between the jib crane hook and the work platform kick plates was 9 feet 4 inches. (Note: Safety harnesses were worn by personnel on the ITR platform; hard hats below.) Figure 29 shows both modules enplacements. Each module was secured with four bolts, each torqued in place to AISC requirements. The hoist enclosure handrails were used to support a tarpaulin for protection against the weather.

"Open loop" operation of each hoist was required to accomplish installation of the cargo coupling and attachment of the signal conductor cable. Test instrumentation and lash—up of control room equipment followed.

The following cargo system components were removable from the ITR overhead work platform without requiring operation of the system:

- 1. Hoist drive
- 2. Pneumatic valves and ducting
- 3. Signal conductor reel
- 4. Hoist
- 5. Span actuator
- 6. Rails (with hoist removed)

The following components were removable, but they required operation of the PPG and control system:

- 1. Hoist cables
- 2. Cable cutter assemblies
- 3. Signal conductor disconnect

PPG AND AIR SUPPLY INSTALLATION

The air supply installation consisted of the pneumatic power generator (power unit, load compressor and controls) prepared as a separate package and the air distribution system. The PPG location and air distribution insulated piping are shown in Figures 30 through 35.

INSTRUMENTATION INSTALLATIONS

The LCC station and test rig instrumentation installations are shown in Figures 36 through 41.

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Hoist System Installed in ITR. Figure 29.

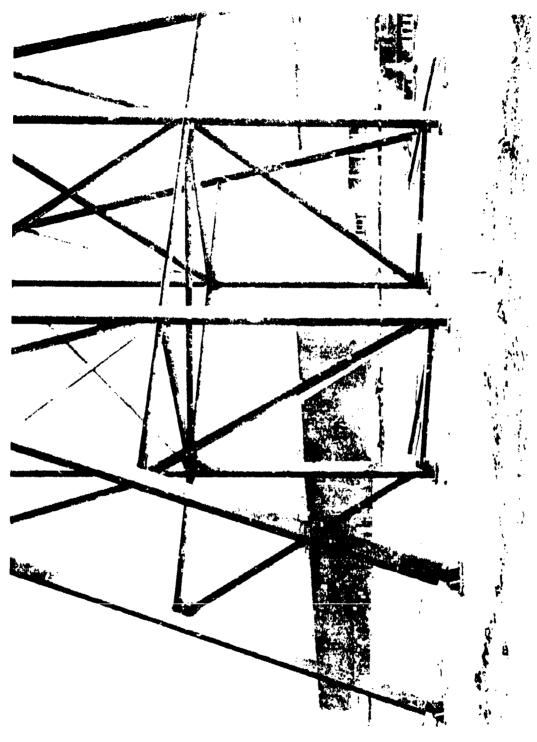


Figure 30. PPG Pad at Aft Tower.

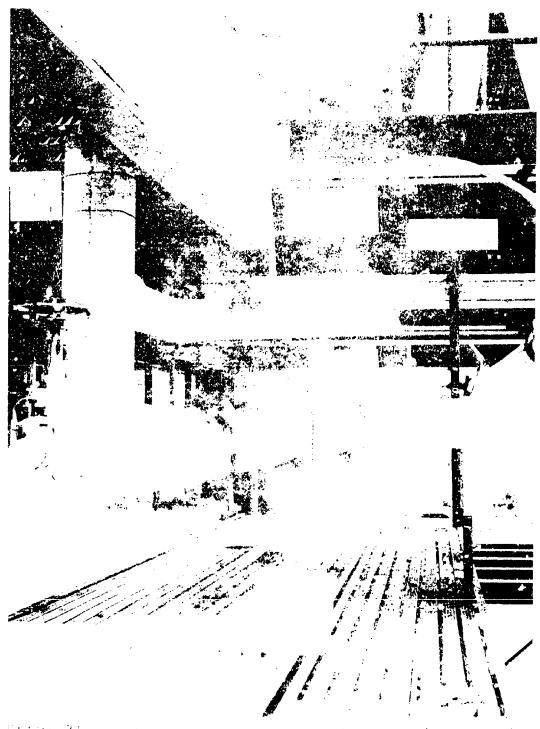


Figure 21. The second supply.

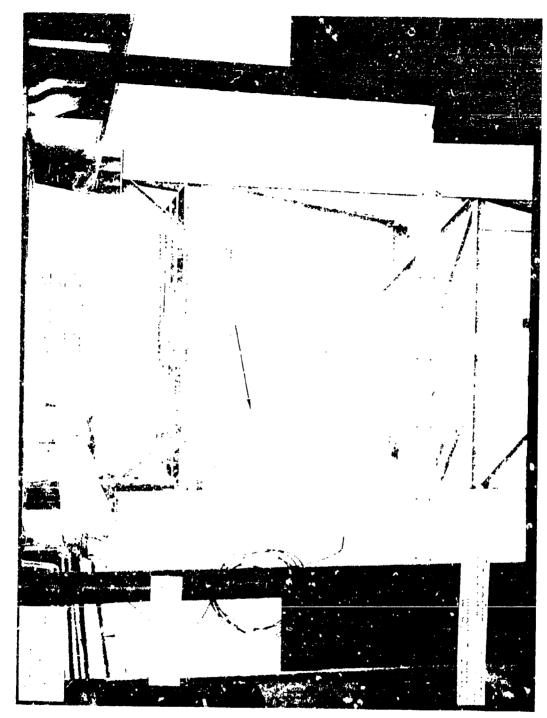


Figure 32. Insulated Air Header and Risor.

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Figure 33. Complete PPG Enclosure and Air Distribution Line.

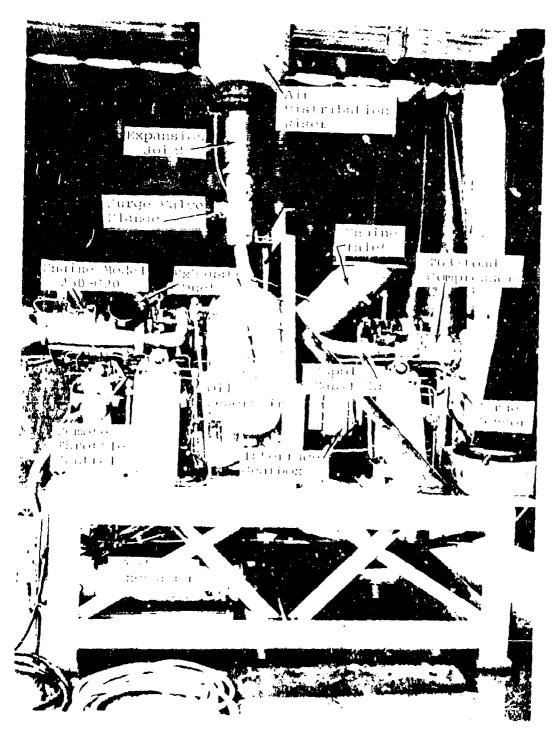
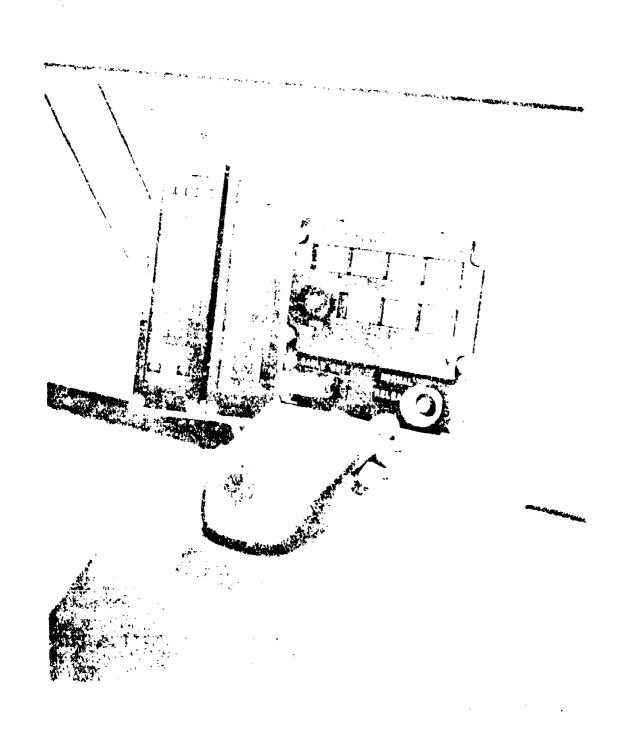


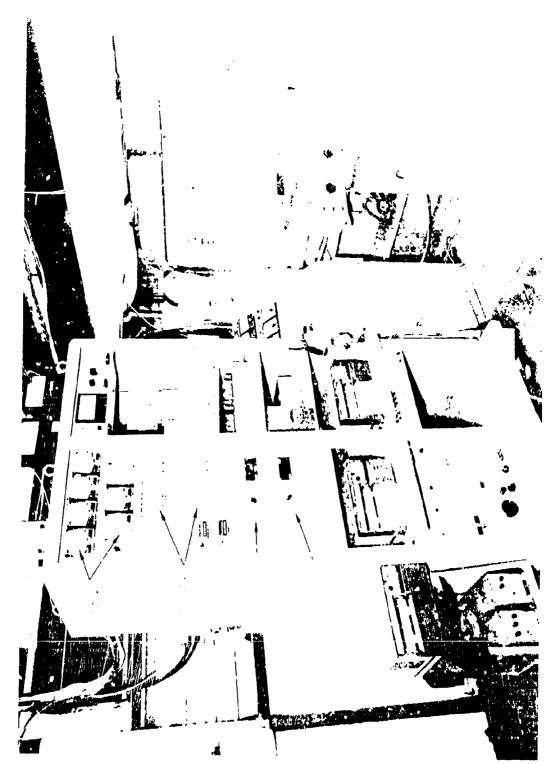
Figure 34. Side View of PPG.



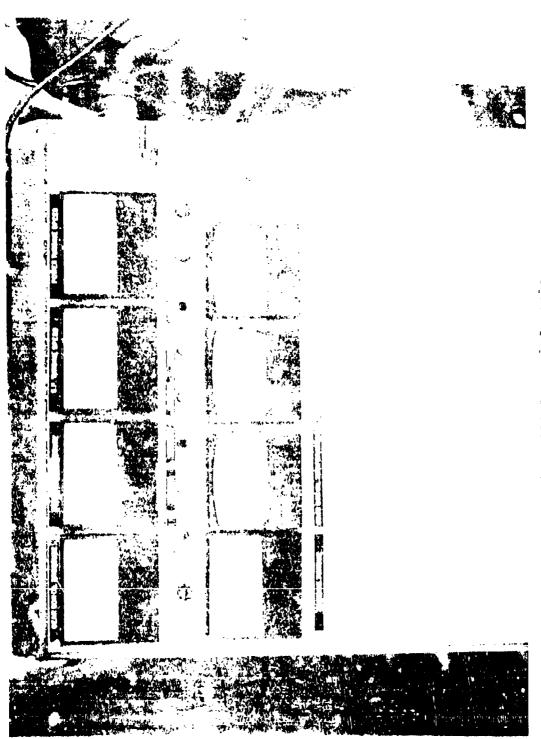
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Signme 37. Instrumentation, 220 and 100 Station.



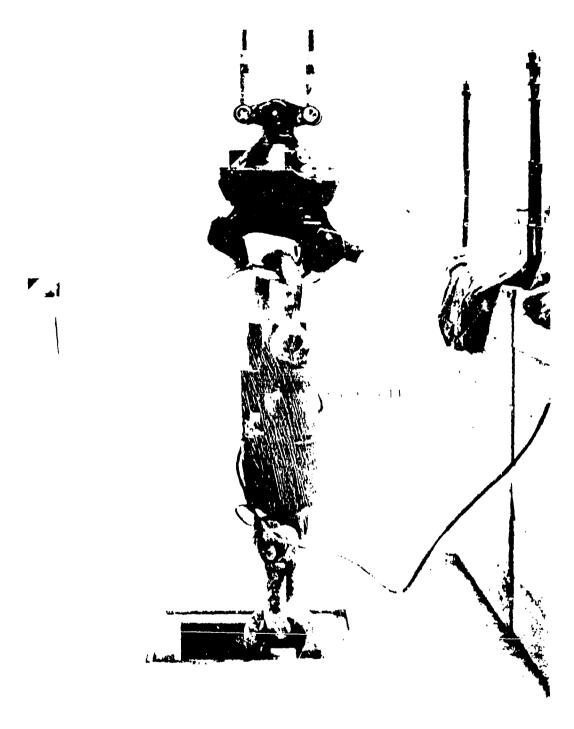
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Figure 41. Status Load Pest Instrumentation.

PROOF LOADING

Handling fixtures and the utility hoist were proof-loaded in accordance with the following schedule:

<u>Fixture</u>	Rated Load, 1b	Proof Load, 1b
Moist sling ST-51273 Moist and module sling ST-40972 Utility hoist-load lifter	2,000 6,000	4,000 12,000
Series 700 and jib crane (davit) model no.990740	6,000	9,000

SPECIMEN ENSTALLATION AND REMOVAL

The cargo system demonstration hardware was initially installed in the individual hoist modules at ground level. Only small hand tools and a sling were required. The modules were elevated on 4-foot workstands to provide access below the assembly. The module buildup started with the span positioning track and drive. The installation of the hoists (with cable) followed, with addition of the equalizer bar, angle and payout sensors, load isolators, hoist drives, pneumatic dueting, and system wiring. Figure 42 shows the hoist/module assembly before emplacement in the ITR.

USE OF IMPEGRATED TEST RIG

The integrated test rig was used to perform the following general categories of cargo handling system tests:

- 1. Functional check of components
- 2. Control system studies (open and closed loop)
- 3. System performance
 - a. No load speed
 - b. Constant load speed
 - c. Stall torque (live loads)
- 4. Static loads to maximum (ground tiedown)
- 5. Individual hoist operation
- 6. Findurance
- 7. Asymmetric load
- 8. PPG regulator valve development
- 9. General operating indoctrination
- 10. Dynamic analysis (high-speed photography)
- 11. Load acquisition demonstrations using a container handling device

The PPG starting and operating procedures used are outlined in Appendix III. The instrumentation calibration procedure used is given in Appendix IV.



このでは、そのでは、「大変を行いた。 とうまとく サースの人の事でなったが、このを持つなるのですとかって、これ、人の人のではないでしょう。この人の人がとなってなる。

TOWNER TO THE MODERN STREET TOWN THREET TO THE

Test operations were performed under the prevailing weather conditions for the Philadelphia area during the months of October through April, which included rain, freezing temperatures, snow, hail, and winds up to 35 miles per hour with higher gusts.

Typical events in the ITR program are illustrated in the following figures.

Figure 43 shows a view of the control room during a typical operation. One man is monitoring the instrumentation, one is monitoring the PPG control panel, and the third is operating the holsts from the simulated LCC station.

Figure 44 shows a view of the cargo couplings from the LCC station. The forward coupling, which is nearest the viewer, is in the stowed position. The aft coupling is in the full-up position.

Figure 45 shows a view over the LCC operator's left shoulder during synchronous hoisting of two separate kirksite block loads.

Figure 46 shows a U.S. Army MILVAN container being hoisted using the GFE helicopter-transported container handling device.

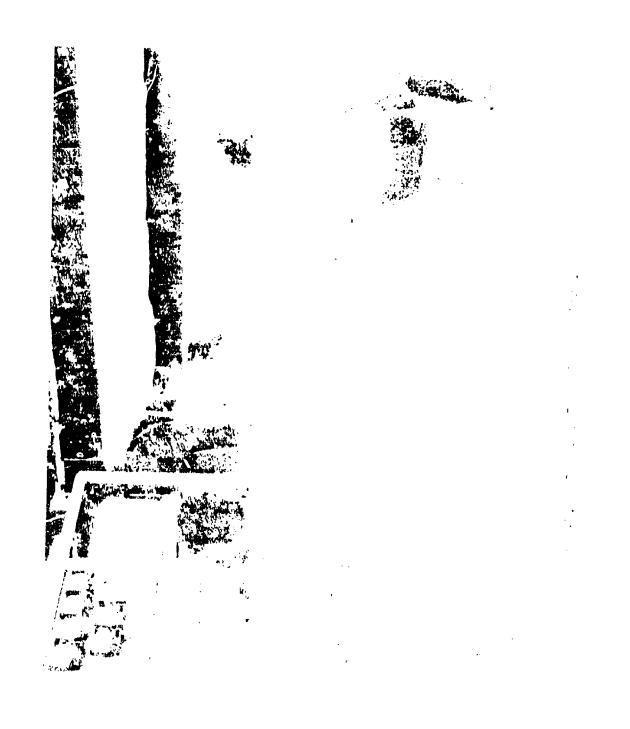
Figure 47 shows a kirksite block load being hoisted using the single-point adapter and sheave system, which is designed to large-load vertical lifts of up to 35 tons.

Figure 48 shows the drive-through capability of the att test rig tower. A MITAVAN container on a standard high-bay flatbed trailer is being positioned for hoisting.

CONCLUSIONS AND RECOMMENDATIONS

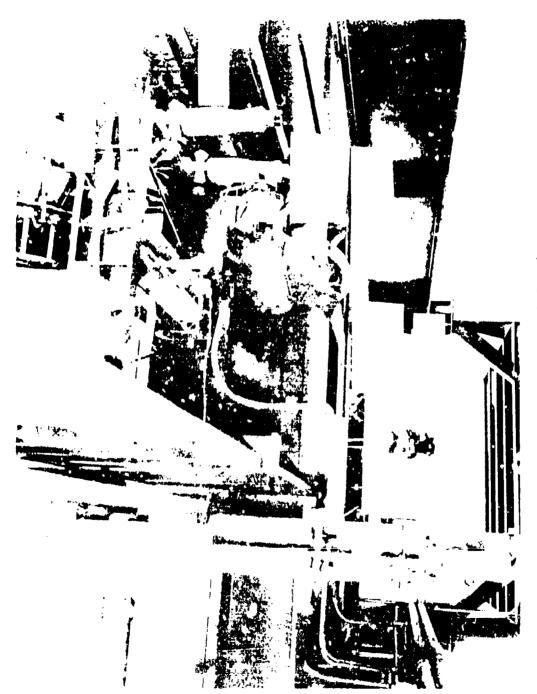
The integrated test rig was a viable fixture for the cargo handling ATC program from the viewpoints of compliance with contractual design objectives, design simplicity, fabrication, erection, system installation, and instrumentation.

Uses of the integrated test rig could be expanded to include maintenance training if work platforms were added beneath the overhead structure to provide access to the hoist assemblies similar to what will exist in the actual helicopter. Improvements are needed in the PPG system; to increase its air supply capacity and equipment reliability.

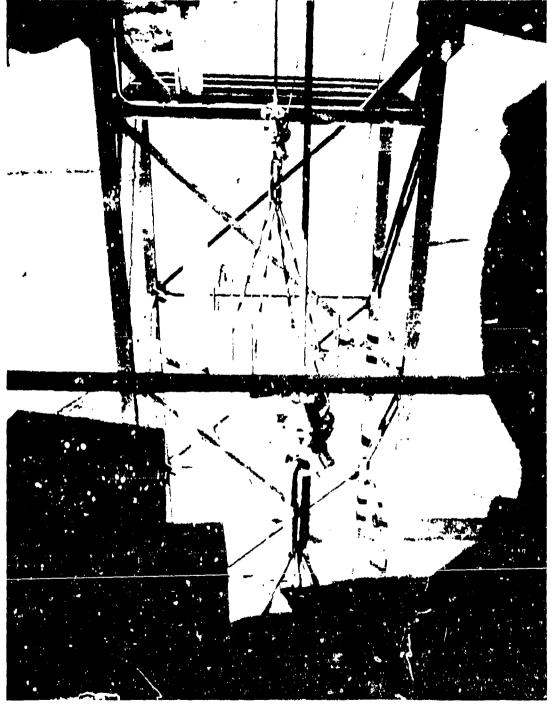


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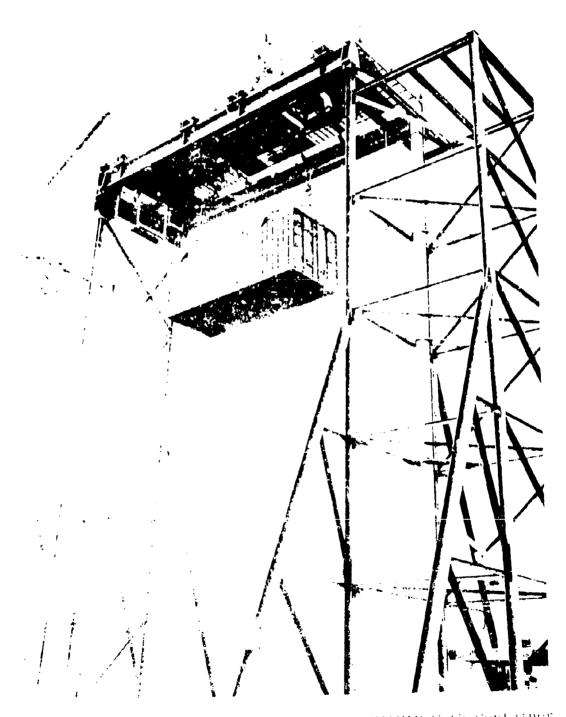
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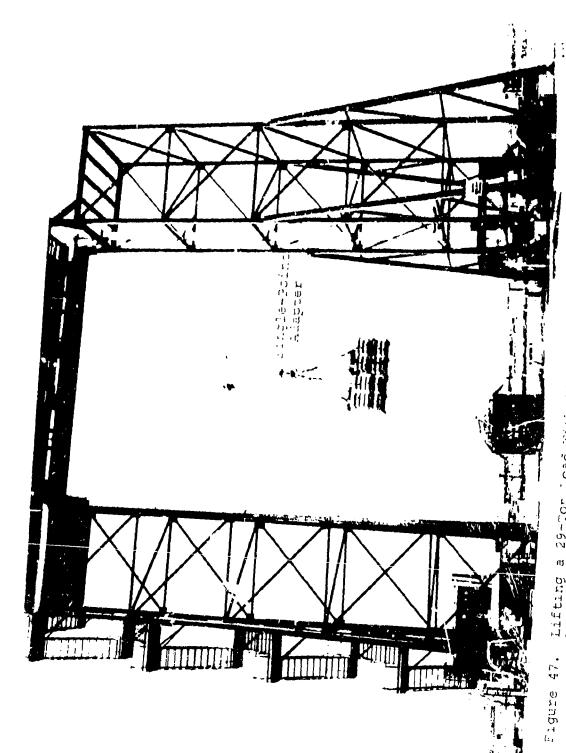


- 1 to View of Two Industrial to the Neur Top of Lift Figure 45. eyele.

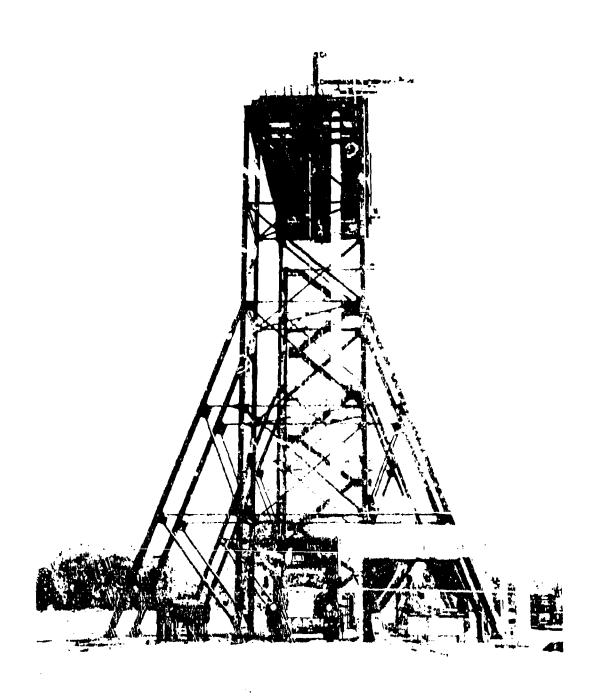


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Figure 16. Problem Matter examples, t. MILWAN with Continuer of Problem Device Device Distance of Poot Life.



Lifting a 29-Ton Load With Single-Point Adapter - Hoist Span V 3 16 Feet.



Diqure 4%. Imforting MINAN Through Buse of Aft Tower.

APPENDIX 1 INTEGRATED TEST RIG STRESS ANALYSIS

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LIST OF SYMBOLS

- A Cross Section Area In.²
- e Distance from Neutral Axis to Extreme Fiber of Beam
- d Depth of Jeam or Girder Inches
- E Modulus of Elasticity of Steel RS1
- e Load Eccentricity Inches
- FA Allowable Stress for Compression Members KS1
- ta Computed Axial Stress KS1
- tb Computed Bending Stress KS1
- FCR Failure Strons KS1
- ty Computed Shear Stress KSI
- 1 Moment of Inertia of Section in.4
- J Polar Moment of Inertia In.4
- L Span Length Inches
- 1 Actual Unbraced Length In.
- M Moment In. KIPS
- M.S. Margin of Safety
- P Applied Load KIPS
- q Shear Flow Lb./ln.
- R Reaction KIPS
- r Governing Radius of Gyration In.
- Ss Section Modulus In. 3
- Torsional Moment In. K
- V Static Shear on Beam KTPS
- Δ Deflection Inches
- Torsional Shear Stress KSI

APPENDIX I INTEGRATED TEST RIG STRESS ANALYSIS

SUMMARY

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This appends is presented to substantiate the strength of towers and overhead steel structure designed for use during testing of cargo hoists of the heavy lift helicopter. It is intended that the data contained herein be used as a basis for establishing the structural integrity of the structure for load changes during HLH testing and/or for tests other than the HDH tests.

A=36 type steel was used throughout since it is common for the steel industry and affords an economical design. American Institute of Steel Construction, Inc., Allowable Stresses and Design Practices, as printed in the sixth edition of their manual, were used. ASTM A325 bolts were used to all connections.

Strength requirements were based upon cargo system loads: 28-ron, design; 70-ton, limit. Rig sarvival - no collapse - is one of the provisions in the design in the event of a sling or other tailure during testing. The towers were designed using 1.5 factors of maximum test load, or tailure loads, with a horizontal load resulting from a 5° component in both cases. All horizontal load components were considered reversible. Stairways, walkw.ys, and work platforms were designed to meet the requirements of the Occupational Safety and Health Standards (Code of Federal Regulations, Title 29, Chapter XVII, Part 1910).

Because of height requirements of the towers and the relative low design loads, a trussed frame configuration was selected. Requirements for a control room in a location near the hoists so that the hoist operator could be located in the same relative position as he would be in the helicopter made the selection of a 14-foot-square tower desirable. Easy installation of stairways on this type of tower was also a factor in the selection of this configuration. Outriggers were used on each leg of the tower in the inboard-outboard plane for two reasons:

- 1. The possibility of an enclosure being erected over the top of the towers which would result in considerable wind loading.
- 2. The requirement that the secondary bracing be removed in the lower portions of the tower to permit a loaded truck to drive through the tower.

Outriggers in the fore-aft plane were not required because the two conditions stated above are not factors in this plane. Also, horizontal components of test loads are reacted by four frames in the fore-aft plane, while in the inboard-outboard direction only the two inner frames will react the horizontal component of loading.

The tower assemblies as delivered by the fabricator were not of the configuration as intended in the original design. This optional construction was analyzed and found to have adequate strength for the requirements of the hoist test tower loads. The reaction loads on the foundation are not as evenly distributed in the optional construction as in the original design; however, since the foundations are resting on bedrock, the optional configuration is sufficient for all test requirements. The configuration as delivered is referred to in the stress analysis as the optional construction configuration.

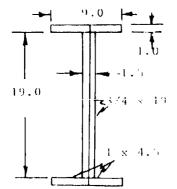
Failure Condition 70 tons on (1) hoist

$$\begin{split} \mathbf{\Sigma} \, \mathbf{MRA} &= \frac{105}{2280} \, \frac{(21.75) + 105}{6280} \, \frac{(59.75) + 2.5}{640.75} \, \frac{(40.75) - P_{\mathrm{R}}(141.5)}{2280 + 6280 + 10^{5} - 141.5} \, \mathbf{R_{\mathrm{R}}} \\ &= \frac{8661}{141.5} - 61.2 \, \mathbf{K} \\ &= \mathbf{R_{\mathrm{A}}} - 61.2 \, - 105 \, - 2.5 \, - 151.3 \, \mathbf{K} \end{split}$$

V 151.3K 45.8

MMAX 81.75 (61.2) 5000 In-E

S Req 14 5000 In R 250 In S 20 801



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- $(2)3/4 \times 19 1 = 428.7(2) = 877.4 \text{ In}^4$
- (4) 1 x 4.5 ! $200 \times 9 = \frac{1800.0 \text{ In}^4}{2657.4 \text{ In}^4}$

 - $T_{\rm D} = 5000$ In E $_{\rm 18.85}$ RGT Bonding Stress $265.4~{\rm In}^3$

Max Test Load 1/2 Farture Condition (a. 9.5 KDI 86

b 9.5 KSI Bending Stress

Cross-Sectional Area at Point of Max Shear

A = (2)11(.75) = 16.50 tn.²

$$f_S = \frac{P}{\Lambda} = \frac{151.3}{16.5} = 9.17$$
 KS1 Shear Stress
M.S. = $\frac{14.5}{9.17} - 1 = .58$

Long. Shear $f_S = \frac{VQ}{Tb}$ at weld (4)3/8 welds

$$Q = 1(10) = 90 \text{ 1n.}^4$$

$$f = \frac{151.3(90)}{2657(1.5)} = 3.42 \text{ KS1 bong. Shear}$$

SK301-11302-1 Torsional Shear

Element	<u>. l</u>	t	bt 3	
Flange	4.125	1.0	4.125	
Flange	4.125	1.0	4.125	
Web	20.0	.75 Σ bt 3	8.43 16.680	
	$J = \frac{2bt^3}{3}$	- <u>16.68</u>	- 5.56	1.97 b

Torsion =
$$(1.97-c)52.5K = 1.35(52.5) = 70.8 \text{ Fig. K}$$

e = .622

Torsional Shear Stress
$$\frac{7 \cdot \text{Tt}}{\text{J}} = \frac{70.8(.75)}{5.56} = 9.55 \text{ KSI}$$

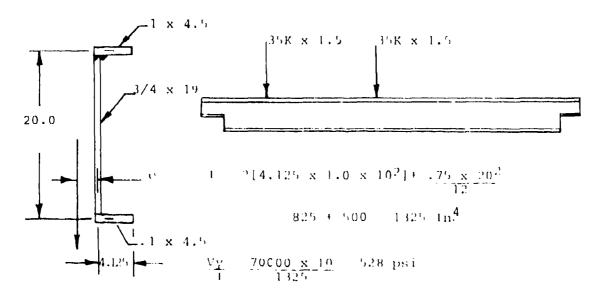
$$P = \frac{70T(1.5)}{4} = 52.5K$$

M.S. - Margin of safety based on allowable design stress (22 KS1 tension, 14.5 KS1 shear)

Bending M.S. =
$$\frac{Fa}{fa} = 1 = \frac{22}{18.85/1.5} = -1 = .75$$
 (F.S.)

$$\frac{\text{Shear}}{\frac{14.5}{9.55/1.5}} -1 = 1.28$$

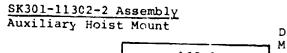
SK301-11302-1 Shear Center

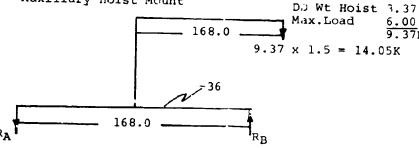


Flange
$$q \approx 528 (1.0 \text{ g})$$
 when $g=0$ $q=0$ g 4.12 $q \approx 2175$ lb/in/at corner

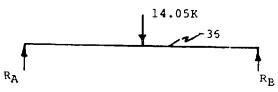
Web
$$q = 2175 + 528(10) \cdot \frac{.75}{.2} = .667y(..75y)$$

 $-2175 + 1982 = 250y^2$
 $-2175(20) + 1982(20).667 = 70,000$
 $-70,000 = -4.12(528)20 = 43500$
 $-622 = -43500 = .622$





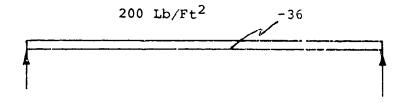
$$-R_A = R_B = \frac{14.05(168)}{168} = 14.08K$$
 (2 beams)
7.64K/beam



$$R_A = R_B = \frac{P}{2} = \frac{14.05}{2} = 7.02K$$

Total Moment =
$$14.05 (168) + 7.02 (84) - 2$$
 beams = $2360 + 592$

$$f_b = \frac{M}{Sx} = \frac{1476}{121.1} = 12.2 \text{ KSI} - 14 \text{ W}^2 78$$

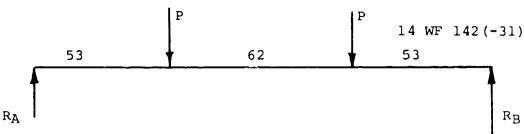


$$M = \frac{w1^2}{8} = \frac{.05(168)^2}{8} = 176 \text{ In K}$$

$$f_b = \frac{M}{Sx} = \frac{176}{121.1} = 1.45 \text{ KSI}$$

Total Stress - Uniform Load + Heist Load
$$f_b = 12.2 + 1.45 = 13.65 \text{ KSI}$$

SK301-11302-3 Hoist Module Support Assembly



Failure Condition

$$P = \frac{151.3}{2} + \frac{DD Wt}{2} =$$

$$= 75.65 + \frac{2.5(100.75)}{141.5} = 75.65 + 1.9 = 77.55$$

$$R_A = R_B = 77.55$$

77.55
V

14 WF 142
A = 41.85 In.²
Sx=226.7 In.³

$$f_{S} = \frac{P}{A} = \frac{77.55}{41.85} = 86 \text{ KSI Shear Stress}$$
 $M_{MAX} = 77.55(53) = 4120 \text{ In. K}$
 $f_{b} = \frac{M}{Sx} = \frac{4120}{226.7} = 18.15 \text{ KSI}$

Max. Test Load =
$$1/2$$
 Failure Condition $f_b = 9.1$ KSI

$$R_{A}=2.4K$$

$$-32 \qquad 6[8.2 \text{ Lb} \quad Sx = 4.3 \text{ In.}^{3}$$

$$M_{MAX} = \frac{w1^{2}}{8} = \frac{33.3(144)^{2}}{8} = 86.3 \text{ In. } K$$

$$f_{b} = \frac{M}{Sx} = \frac{86.3}{4.3} = 20.07 \text{ KSI}$$

$$1.0K$$

$$R_{B}=1/2K$$

$$M_{MAX} = \frac{P7}{4} = \frac{1.000(144)}{4} = 36 \text{ In. } K$$

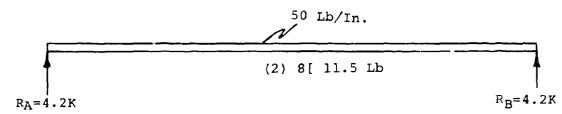
$$f_{b} = \frac{M}{Sx} = \frac{36}{4.3} = 8.38 \text{ KSI}$$

Grip Strut Flooring Capacity (members tack welded together)

Concentrated load 1100 Lb. Uniformly dist. load 353 Lb/Ft²

SK301-11302-4 Work Platform Assembly

Design load 200 Lb/Ft.



$$M_{MAX} = \frac{w^{1/2}}{8} = \frac{.05(168)^2}{8} = 176.2 \text{ In. K}$$

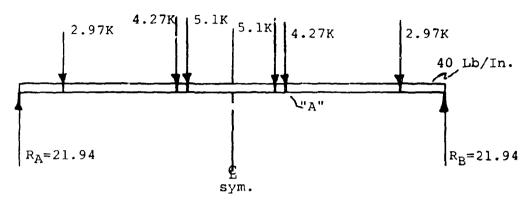
$$f_b = \frac{M}{Sx} = \frac{176.2}{16.2} = 10.88 \text{ KSI}$$

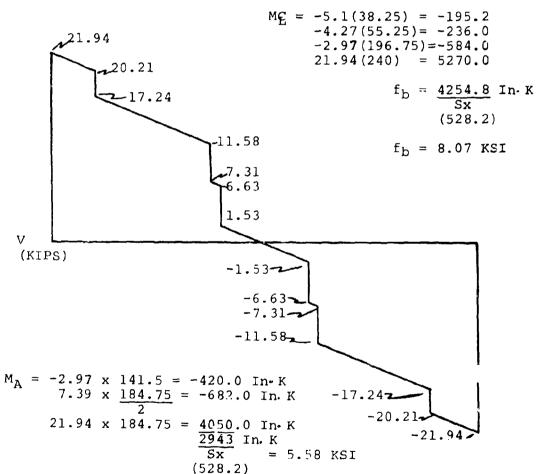
Grip strut on this assembly has a capability of 235 Lb/Ft² of uniformly distributed load and a concentrated load of 633 Lb at the midpoint.

```
SK301-11302-5

Main Fore & Aft Beam Assembly
30WF172
A =50.72 In<sup>2</sup>
Sx=528.2 In.<sup>3</sup>

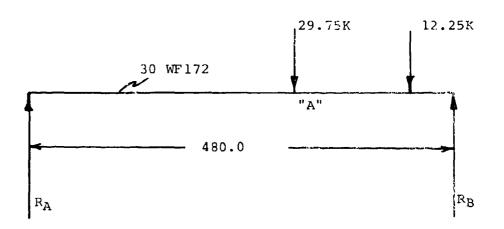
Dead Wt Only
16' position
```





30 WF172

Failure condition
16' hoist position
28-ton load @ 1.5 F.S.



$$\mathbf{X}_{R_A} = 29.75(295.25) + 12.25(436.75) - 30 R_B$$

$$R_B = \frac{8780 + 53.50}{480} = \frac{141.30}{480} = 29.5K$$

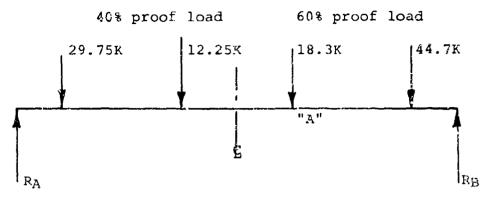
$$R_A = 12.5$$

$$\begin{array}{rcl} \text{MA} &=& -141.5\,(12.25) &+& 184.75\,(29.5) \\ &-1732 &+& 5450 &=& 3718 \text{ In, K} \end{array}$$

$$\begin{array}{rcl} \text{f} &=& \text{MA} \\ &=& \frac{3718}{528.2} &=& 7.04 \text{ KSI Failure load} \\ &&& \frac{5.58}{12.62} \text{ KSI DD Wt} \end{array}$$

30 WF172

Proof load 26' hoist position 70-ton load @ 1.5 F.S. .60-.40 load split

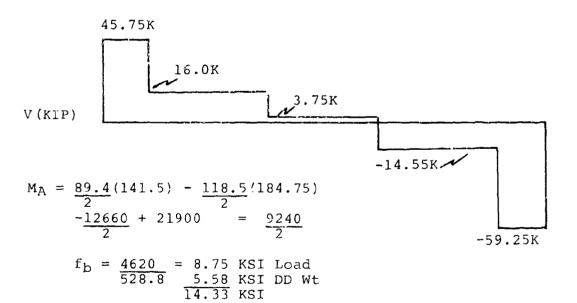


$$\mathbf{Z} M_{RA} = 29.75 (43.25) + 12.25 (184.75) + 18.3 (295.25) + 44.7 (436.75)$$

$$\frac{2575}{2} + \frac{4530}{2} + \frac{10800}{2} + \frac{39000}{2} -480R_{B}$$

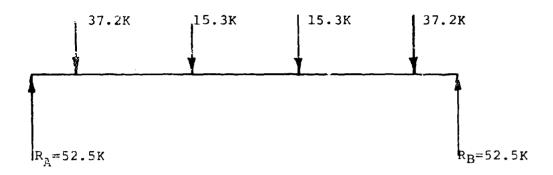
$$R_{B} = \frac{2845.2}{480} = 59.25K$$

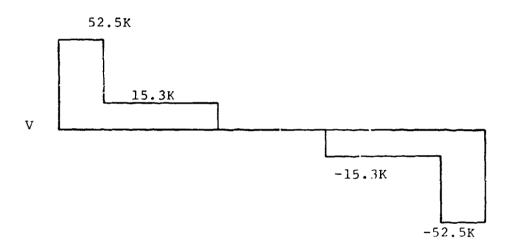
$$R_{A} = 45.75K$$



30 WF172

Proof load 26' hoist position 70-ton load @ 1.5 F.S. .50-.50 load split





$$= 9700 + 52.70 = 4430 \text{ In.K}$$

$$f_{b} = \frac{M_{MAX}}{Sx} = \frac{4430}{528.2} \text{ In.}^{3} = 8.75 \text{ KSI load}$$

$$8.07 \text{ KSI DD Wt}$$

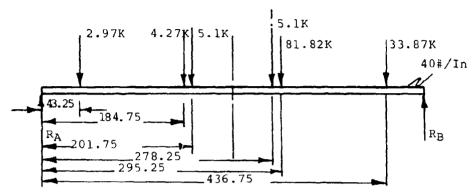
$$16.82 \text{ KSI}$$

 $M_{MAX} = -52.5(184.75) + 37.2(141.5)$

30 WF172 A =50.72 Sx=528.2 In³

SK301-11302-5

Failure Condition at 16' position 70-ton proof load (DD Wt included)



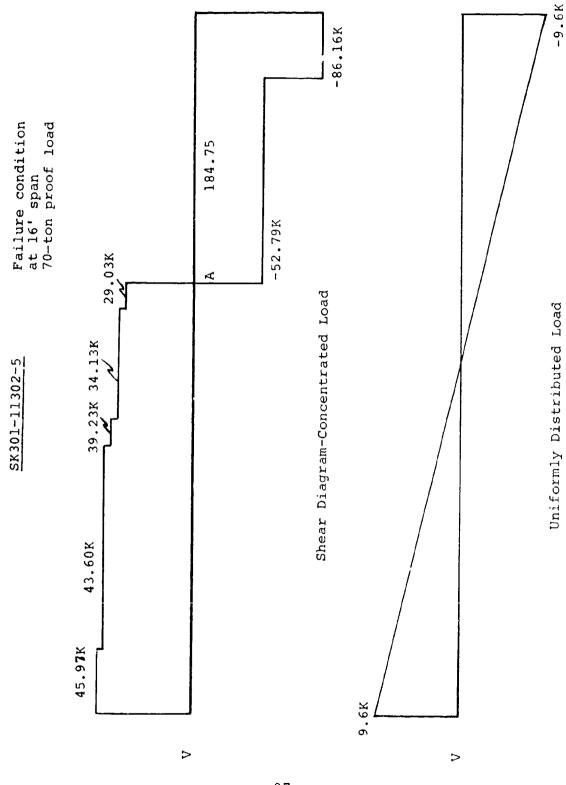
 $\mathbf{Z}_{R_A} = 19.20(240) + 2.97(43.25) + 4.27(184.75) + 5.1(201.75) + 5.1(278.25)$

+81.82(295.25) + 33.87(436.75)-480RB

= 4608 + 128.45 + 788.88 + 1028.93 + 1419.08 + 24157.36 + 14792.72

 $R_{\rm B} = \frac{46923.42}{480} = 97.76$ K

 Σ Fu = +19.20 + 2.97 + 4.27 + 5.1 + 5.1 + 81.82 + 33.87 -97.76-R_A R_A = 54.57



Failure condition at 16' span 70-ton proof load

30 WF172

Moment is Max. at PT. "A"

Solving for Max. Mom.

 $M_A = -86.16(184.75) - 9.6(184.75) + 33.87(141.5) - 400.45$

= 12498.6 In. K

 $f_b = \frac{12498.6}{528.2} = 23.7 \text{ KSI}$

30 WF172 A =50.72 In? Sx=528.2 In? SK301-11302-5 Failure condition at 26' position 70-ton proof load (DD Wt included)

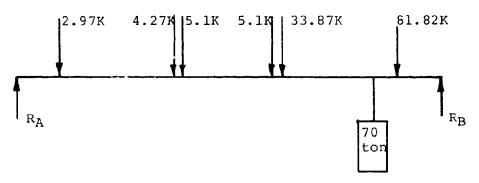


Figure A

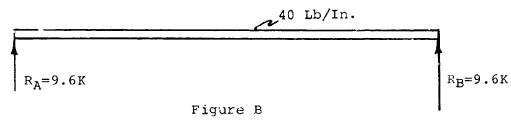


Figure A

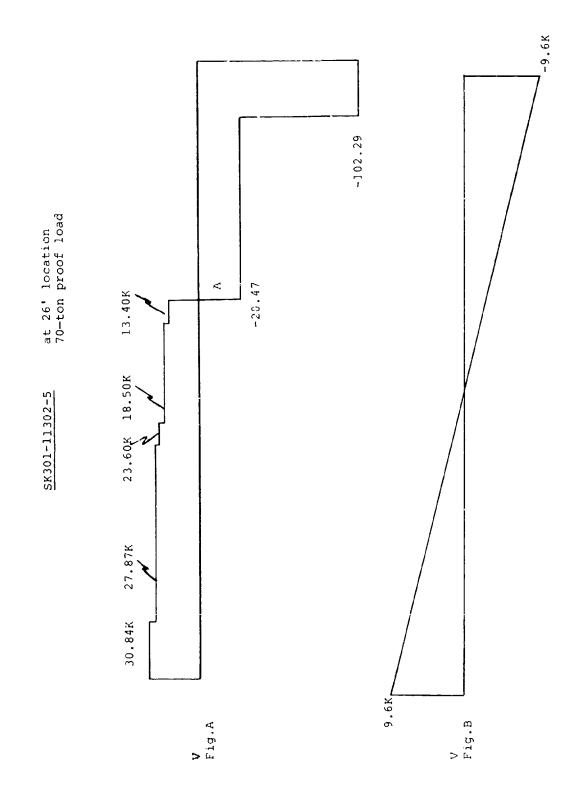
$$\Sigma$$
 M_{RA} = 128.45 + 788.88 + 1028.93 + 1419.08 + 33.87(295.25) (10000.12) + 81.82(436.75)-480R_B

(35734.89)

$$R_B = \frac{49,100.34}{480} = 102.29K$$

$$R_A = 30.84K$$

Shear and Moment Diagram on Page 100.



SK301-11302-5 Failure condition at 26' position 70-ton proof load

Moment is Max. at PT. "A"

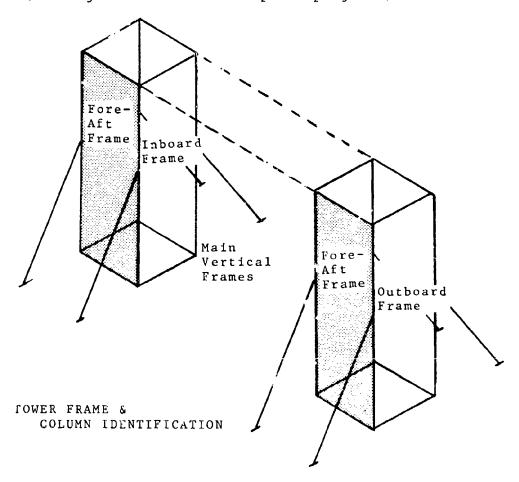
 $M_{MAX} = -102.29(184.75) - 9.6(184.75) + 81.82(141.5) + 400.45$

= 8693.72 = 16.45 KSI

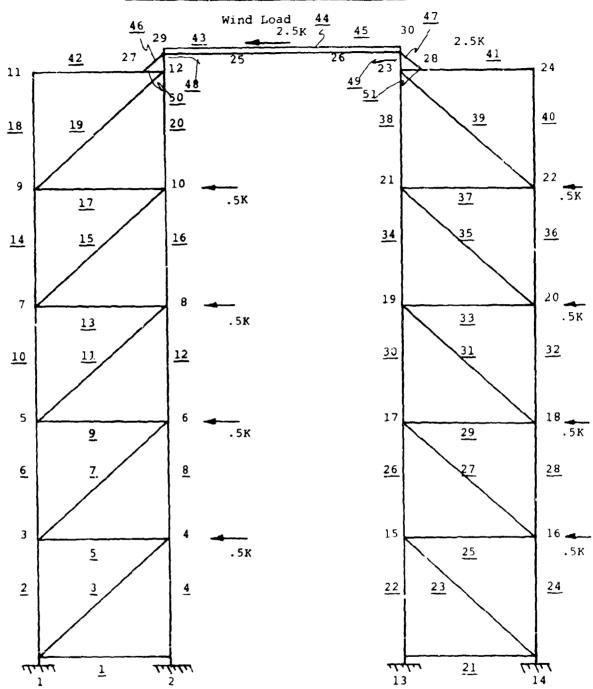
SK301-11304-1 Tower Assembly

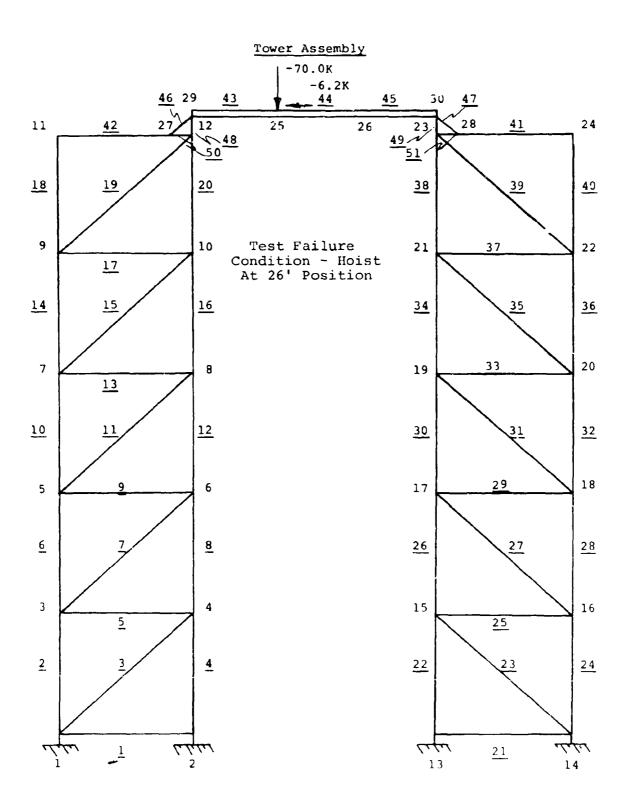
The tower assembly is analyzed using the plain frame computer program.* The fore-aft frame is analyzed using three separate loads: wind loads, test loads, and deadweight. The total load from the three combined loads and stress is computed for each member. The inboard-outboard frame is analyzed in the same way. To obtain the maximum stress in the main vertical columns it is resessary to add the stress found in the inboard-outboard frame to the stress of the fore-aft frame found as a result of the horizontal test loads only. The tower component identification is shown below.

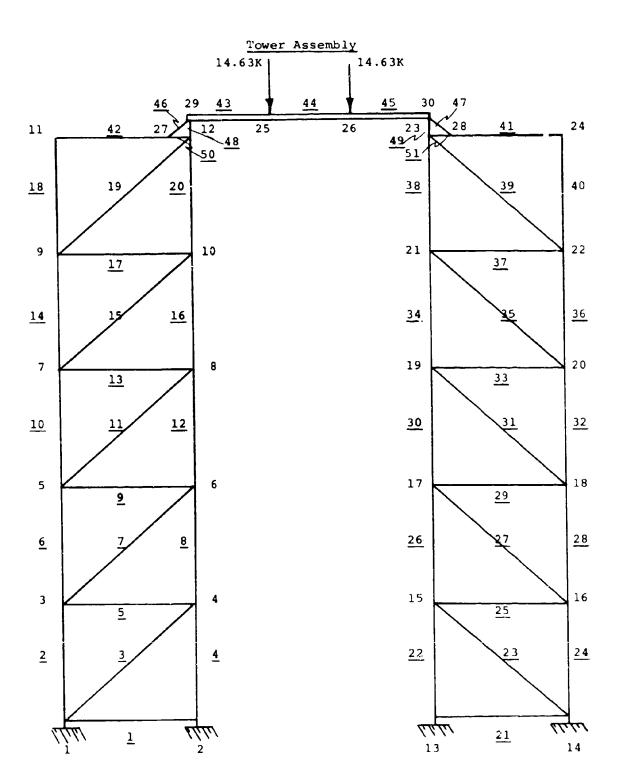
(*Boeing-Wichita Watfor computer program.)



Tower Assembly - Fore-Aft Frame



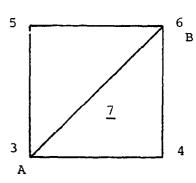




GLOSSARY - COMPUTER ANALYSIS

Coordinates of Joints - Location of joints of frame using X and Y coordinate axes, the X axis being the line formed by base of towers and the Y axis the left-hand column centerline.

Member End Action - Nodes 1 and 2 define coordinate axis system of each member (see example). Node 1 represents "A" end of member, Node 2 represents "B" end of member. The line 3-6 is the Y axis of the member. The X



axis is 90° to the Y axis.

Negative "X" action at the "A"
end, and positive "X" action at
the "B" end would indicate
tension (KIPS) in member.

"Y" action indicates shear (KIPS)
in member. MZ action indicates
moment (In.K.) about X axis cf
member.

Member Information:

Area - Cross-sectional area (In.2) of member.

Inertia - Moment of inertia (In.4) of member.

Modulus - Modulus of elasticity of member.

Length - Length in inches.

Actions applied at joints - load applied at joints. Support Reactions - foundation loads.

X Reaction - Shear

Y Reaction - Positive (down load) - Negative (up load)

	TABLE	VI. TOWER	R ASSEMBLY	- FORE-AFT	FRAME LOADS	S (KIPS).		-
Member	Size	Area	Moment	Wind Load	Test Load	Dead- weight	Horiz. Test Only	Total Load
1	(7r) 4x3-1/2x3/8	5.34	00.	00.	00.	00.	00.	00
2	8 WF 31	9.12	18.73	12.27	25.99	5.92		44.18
m	7F 4x3-1/2x3/8	5.34	49	5.92	9.13	2.08		17.13
4	8 WF 40	11.76	13.47	-16.24	25.63	7.28	14.56	16.67
Ŋ	7F 4x3-1/2x3/8	5.34	1.94	-3.93	-6.74	-1.54		-12.21
9	8 WF 31	9.12	1.82	8.63	19.82	4.52		33.07
7	7F 4x3-1/2x3/8	5.34	1.41	5.41	9.18	2.09		16.68
∞	8 WF 40	11.76	6.29	-12.26	31.77	89.8	11.75	28.19
6	7F 4x3-1/2x3/8	5.34	1.58	-3.53	-6.87	-1.57		-11.97
10	8 WF 31	9.12	6.50	5.45	13.61	3.10		22.16
11	7F 4x3-1/2x3/8	5.34	1.48	4.73	9.23	2.10		16.06
12	8 WF 40	11.76	4.35	-8.62	37.94	10.08	8.97	39.40
13	7F 4x3-1/2x3/8	5.34	1.95	-2.93	-6.64	-1.51		-11.08
14	8 WF 31	9.12	6.43	2,76	7.53	1.72		12.01
15	7F 4x3-1/2x3/8	5.34	. 44	4.00	9.05	2.06		15.11
16	8 WE 40	11.76	35.35	-5.44	44.15	11.50	6.16	50.21
17	75 4x3-1/2x3/8	5.34	1.42	-2.75	-7.51	-1.71		-11.97
18	8 WF 31	9.12	36.70	.36	86.	.22		1.56
19	75 4x3-1/2x3/8	5.34	6.71	3.55	9.72	2.22		15.49
0 C	8 WF 40	11.76	115.96	-2.75	50.22	12.88	3.39	60.35
21	7/ 4x3-1/2x3/8	5.34	00.	00.	00.	00.		00.

			TABLE VI.	Continued			,	
Member	Size	Area	Moment	Wind Load	Test Load	Dead- weight	Horiz. Test Only	Total Load
22	8 WF 40	11.76	-10.75	16.08	9.21	7.28		32.57
2.3	7F 4x3-1/2x3/8	5.34	28	-5.87	.88	2.08		-2.91
2.4	8 WF 31	9.12	-12.05	-12.15	2.45	5.92		-3.78
25	7F 4x3-1/2x3/8	5.34	1.05	4.39	~.64	-1.54		-2.21
26	8 WF 40	11.76	-2.48	12.13	9.80	89.8		30.61
27	75 4x3-1/2x3/8	5.34	67.	-5.36	98.	-2.09		-2.41
28	8 WF 31	9.12	1.60	-8.53	1.87	4.52		-6.66
29	75 4x3-1/2x3/8	5.34	. 75	3.99	65	-1.57		1.77
30	8 WF 40	11.76	1.62	8.52	10.38	10.08		28.98
31	7F 4x3-1/2x3/8	5.34	.45	-4.68	.87	2.10		-1.7i
32	8 WF 31	9.12	-2.31	-5.38	1.29	3.10		99
33	7F 4x3-1/2x3/8	5.34	.77	3,40	63	-1.51		1.26
3.4	8 WF 40	11.76	-7.14	5.37	10.96	11.50		27.83
35	7/ 4x3-1/2x3/8	5.34	.23	-3.96	.85	2.06		-1.05
36	8 WF 31	9.12	2.39	-2.72	.72	1.72		28
37	75 4x3-1/2x3/8	5.34	.73	3.21	71	-1.71		.79
38	8 WF 40	11.76	-14.64	2.71	11.53	12.88		27.12
39	75 4x3-1/2x3/8	5.34	1.83	-3.50	.93	2.22		35
4 0	8 WF 31	9.12	13.57	36	60.	.22		
41	8 WF 31	9.12	34.21	-,11	.02	90.		
42	8 WF 31	9.12	149.98	.12	.29	90.		.47

			TABLE VI.	Concluded	ŋ			
Member	Size	Area	Moment	Wind Load	Test Load	Dead-	Horiz. Test Only	Total Load
43	30 WF 172	50.65	See Page 99	66				
44	30 WF 172	29.05	See Page 99	66				
45	30 WF 172	50.65	See Page 99	66				
46	8 WF 31	9.12	-27.86	2.36	7.91	1.84		12.11
47	8 WF 31	9.12	-16.40	-2,33	.92	1.84		
25	8 WF 40	11.76	-34.44	-1.33	52.70	13.41		64.78
49	8 WF 40	11.76	-17.11	1.31	11.60	13.41		26.32
50	8 WF 31	9.12	116.95	-1.85	-5.81	-1.34		-9.00
51	8 WF 31	9.12	50.44	1.83	64	-1.34		

SK301-11304-1 Tower Assembly - Fore-Aft Frame

The following calculations are for stress in the members of the frame (see page 103 for member numbers).

General Equation:

$$f = \underbrace{Moment \ x \ 1.5}_{Section \ Modulus} + \underbrace{Total \ Axial \ Load \ x \ 1.5}_{Area}$$

Member 19

$$f = \frac{6.71 \times 1.5}{3.0} + \frac{15.49 \times 1.5}{5.34} = 3.36 + 4.36 = 7.72KSI$$

. .

$$\frac{L}{r} = \frac{226.5}{1.25} = 181$$
 Fa = 6.56

Member 20

$$f = \frac{115.96 \times 1.5}{35.5} + \frac{60.35 \times 1.5}{11.76} = 4.92 + 7.70=12.62KSI$$

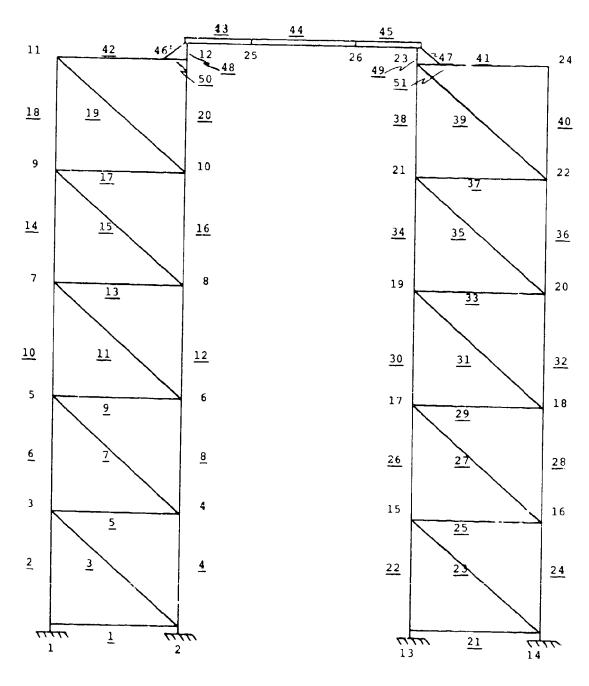
$$\frac{L}{r} = \frac{152}{2.04} = 74.5$$
 Fa = 14.89 KSI

Member 42

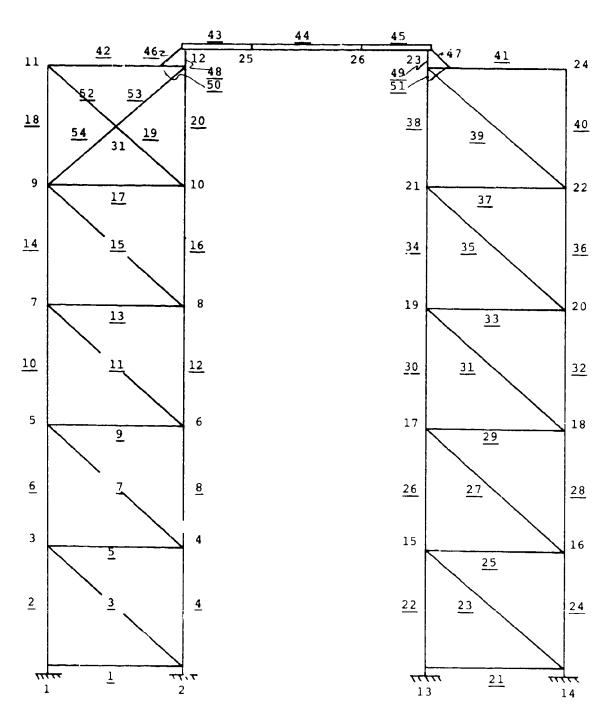
$$f = \frac{149.98 \times 1.5}{27.4} + \frac{.47 \times 1.5}{9.12} = 8.23 + .08 = 8.31KSI$$

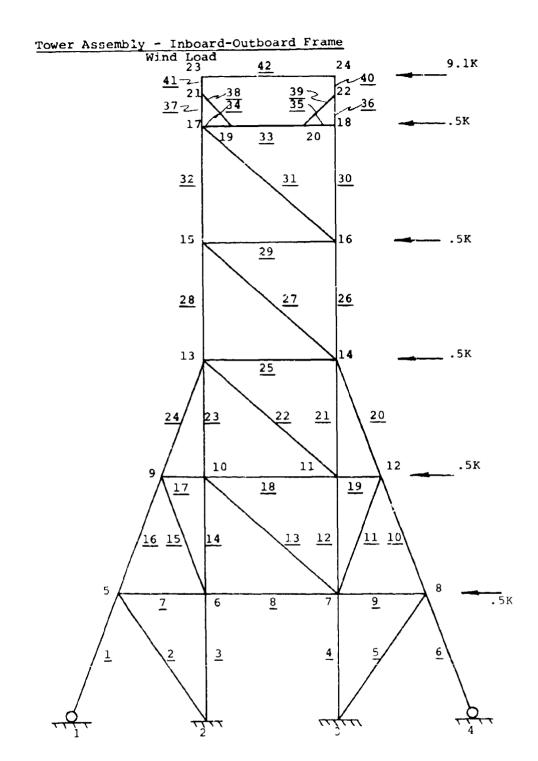
$$\frac{L}{r} = \frac{168}{2.01} = 83.5$$
 Fa = 14.03 KSI

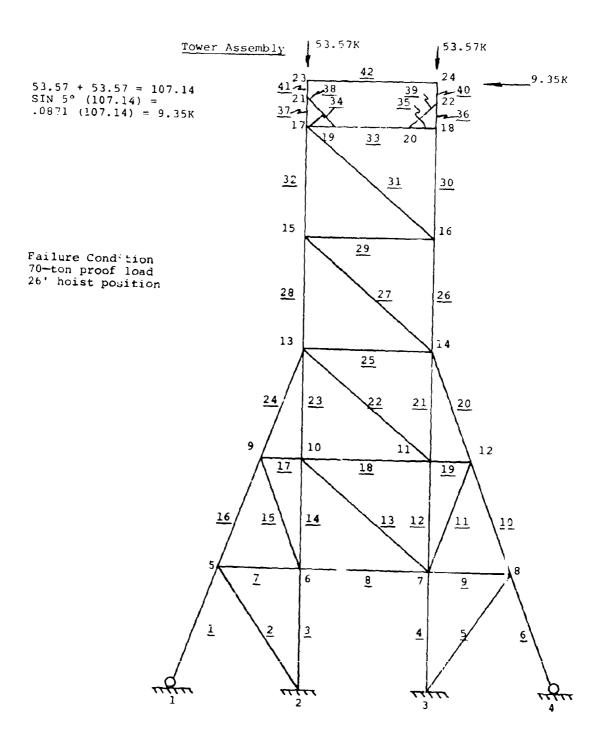
Tower Assembly (Optional Construction)



Tower Assembly (Optional Construction)







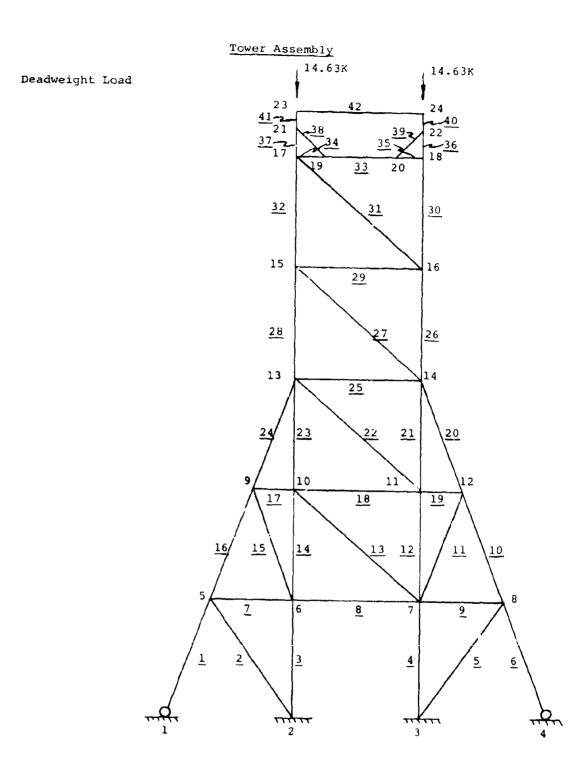


	TABLE	CE VII.	TOWER ASSEMBLY	۱ ا	INBOARD/OUTBOARD	ARD FRAME.		
Member		Area	Sect.Mod.	Moment	Wind Ld	Test Ld	Dead- weight	Total
1	10 WF 33	9.71	9.2	Very small	+14.94	31.03	4.85	50.86
2	7543-1/2x3/8	5.34	3.0	į	68	4.38	1.24	4.94
3	8 WF 40	11.76	12.1		+8.72	40.27	8.65	57.64
4	8 WF 40	11.76	12.1		-8.35	28.28	9.92	29.85
S	7/ 4×3-1/2×3/8	5.34	3.0		1.02	3.92	1.02	5.96
9	10 WF 33	9.71	9.2		-15.63	3.04	4.50	-8.09
7	7/ 4x3-1/2x3/8	5.34	0.		09.	-3.90	-1.10	-4.40
80	7/4×3-1/2×3/8	5.34	3.0		9.	-2.01	62	-1.98
6	75 4x3-1/2x3/8	5.34	3.0		06	-3.48	06*-	-4.78
10	10 WF 33	9.71	9.2		-14.75	6.40	5.37	-2.98
11	754x3-1/2x3/8	5.34	3.0		19	4.70	1.46	5.97
12	8 WF 40	11.76	12.1		-7.16	23.75	8.36	24.95
13	75 4x3-1/2x3/8	5.34	3.0		-1.52	.19	.03	-1.30
14	8 WF 40	11.76	12.1		8.61	35.08	7.34	51.03
15	75 4x3-1/2x3/8	5.34	3.0		.13	5.55	1.41	7.09
16	10 WF 33	9.71	9.2		14.35	34.77	5.95	55.07
17	75 4×3-1/2×3/8	5.34	3.0		07	-3.79	97	-4.83
18	7/4x3-1/2x3/8	5.34	3.0		1.09	-3.86	-1.17	-3.94
19	7/ 4×3-1/2×3/8	5.34	3.0		.62	-3.23	-1.00	-3.61
20	10 WF 33	9.71	9.2		-14.90	11.03	6.81	2.99
21	8 WF 40	11.76	12.1	•	-6.74	23.21	6.21	24.68

			TABLE	VII. Concl	Concluded.			
Member	Size	Area	Sect.Mod.	Moment	Wind Ld	Test Ld	Dead- weight	Total
22	T4x3-1/2x3/8	5.34	3.0	1.11	68	.84	.25	.41
23	8 WF 40	11.76	12.1	9.37	7.64	35.29	7.55	50.48
	10 WF 33	9.71	9.2	1.29	14.44	40.23	7.34	62.01
. 25	7/4x3-1/2x3/8	5.34	3.0	3.71	5.42	13.16	2.34	20.92
26	8 WF 40	11.76	12.1	12.29	-11.67	42.01	14.60	44.94
27	7/6×3-1/2×3/8	6.84	6.5	3.50	-13.47	-12.49	.01	-25.96
28	8 WF 40	11.76	12.1	90.6	20.71	73.58	14.60	108.89
59	774x3-1/2x3/8	5.34	3.0	2.87	10.10	9:36	01	19.45
30	8 WF 40	11.76	12.1	22.95	-2.77	50.68	14.58	63.49
31	76x3-1/2x3/8	6.84	6.5	8.17	-13.20	-12.86	.02	-26.04
	8 WF 40	11.76	12.1	12.05	11.66	65.16	14.60	91.42
33	8 WF 31.	9.12	27.4	246.90	5.15	4.98	٠٥5	10.18
34	WF	9.12	27.4	5.01	11.80	11.30	11	21.99
35	8 WF 31	9.12	27.4	205.44	-1.84	-2.96	15	4.95
36	8 WF 40	11.76	12.1	112.70	-6.03	46.58	14.38	54.93
3.7	8 WF 40	11.76	12.1	97.71	5.91	58.95	14.40	79.81
38	8 WF 31	9.12	27.4	24.51	-8.58	-7.96	.25	-16.29
39		9.12	27.4	35.46	8.92	10.27	.29	29.48
40	8 WF 40	11.76	12.1	143.68	49	53.09	14.60	67.20
41	8 WF 40	11.76	12.1	133.90	.49	54.11	14.60	69.20
42	8 WF 31	9.12	27.4	83.72	4.55	4.52	05	9.02

SK301-11304-1 Tower Assembly - Inboard/Outboard Frame

$$f = \frac{M \times 1.5}{S} + \frac{P \times 1.5}{A}$$

Member 1
$$f = \frac{50.86 \times 1.5}{9.71} = 7.86 \text{ KSI}$$

 $\frac{L}{r} = \frac{162.0}{1.94} = 83.5 \quad F_A = 14.03 \text{ KSI}$

Member 2
$$f = .90 \times 1.5 + 4.94 \times 1.5 = .45 + 1.39 = 1.84 \text{ KSI}$$

 $\frac{L}{r} = \frac{189}{1.25} = 151$ $F_A = 7.69$

Member 3
$$f = \frac{3.28 \times 1.5}{12.1} + \frac{57.64 \times 1.5}{11.76} = .41 + 7.36 = 7.77 \text{ KSI}$$

 $\frac{L}{r} = \frac{152}{2.04} = 74.5$ $F_A = 14.89$

Member 15
$$f = \frac{1.03 \times 1.5}{3} + \frac{7.09 \times 1.5}{5.34} = .52 + 1.99 = 2.51 \text{ KSI}$$

 $\frac{L}{r} = \frac{162}{1.25} = 129.6$ $F_A = 9.30 \text{ KSI}$

Member 23
$$f = 9.37 \times 1.5 + \frac{50.98 \times 1.5}{11.76} = 1.16 + 6.44 = 7.60 \text{ KSI}$$

 $\frac{L}{r} = 74.5 \quad F_A = 14.89 \text{ KSI}$

Member 24
$$f = \frac{1.29 \times 1.5}{9.2} + \frac{62.01 \times 1.5}{9.71} = .21 + 9.58 = 9.79$$

 $\frac{L}{r} = 83.5$ $F_A = 14.03$

Member 25
$$f = \frac{3.71 \times 1.5}{3.0}$$
 $\frac{20.92 \times 1.5}{5.34} = 1.85 + 5.88 = 7.73 KSI$ $\frac{L}{r} = \frac{168}{1.25} = 134$ $F_A = 8.94$

Member 28
$$f = \frac{9.06 \times 1.5}{12.1} + \frac{108.89 \times 1.5}{11.76} = 1.12 + 13.90=15.02KSI$$

 $\frac{L}{r} = \frac{152}{2.04} = 74.5$ $F_A = 15.90 \text{ KSI (A-36 steel)}$

Member 31
$$f = 8.17 \times 1.5 + 26.04 \times 1.5 = 1.88 + 5.50=7.38KSI$$

 $\frac{L}{r} = \frac{227}{1.39} = 163$ $F_A = 7.16 KSI$ Over

Member 32
$$f = \frac{12.05 \times 1.5}{12.1} + \frac{91.42 \times 1.5}{11.76} = 1.49 + 11.65-13.14KSI$$

 $\frac{L}{r} = 74.5$ $F_A = 15.90 \text{ KSI}$

Member 35
$$f = \frac{205.4 \times 1.5}{27.4} + \frac{4.95 \times 1.5}{9.12} = 11.24 + .81 = 12.05KSI$$

 $\frac{L}{r} = \frac{38.0}{1.61} = 23.6$ $F_A = 20.35$ KSI

Maximum Stress in Main Vertical Columns

f = f(Inbd-Outbd Frame) + f(Fwd-Aft Frame Horiz.Test Only)

Member 28 f = 15.02 +
$$\frac{6.16 \times 1.5}{11.76}$$
 = 15.80 KSI
 $\frac{L}{r} = \frac{152.0}{2.01} = 75.62$ F_A = 15.90 KSI

SK301-11304-1 Tower Assembly - Column Splice - Friction

Design splice load = $70.98 \times 1.5 = 106.4K$ (Max.load) 70.98 = 8.97 + 62.01

16 Ea. 3/4 Dia. Bolts/Splice

Friction Load/Bolt = 19.2K(.35) = 6.73K

 $16 \times 6.73K = 107.6K$

(2) $3/4 \times 9.0 \text{ Plate} \quad A = 13.5 \text{ In}^2$ $f = \frac{P}{A} = \frac{106.4}{13.5} = 7.88 \text{ KSI}$

32 In. of 3/8 Fillet Weld (3.6 K/In.) = 115K

Diagonal Strut Splice - Friction

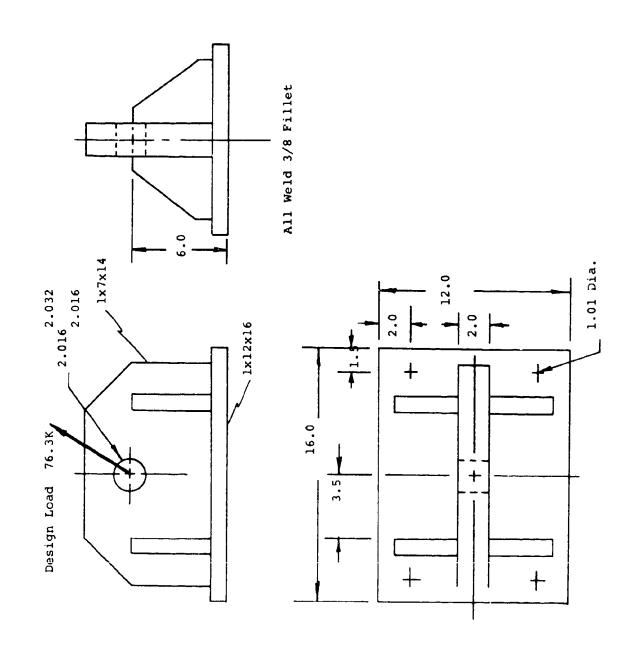
Design Splice Load = $16.06 \times 1.5 = 24.1K$ (Max.10ad)

(2) Ea. 3/4 Dia. Bolts/Splice - Double Friction

Friction Load/Bolt = $2 \times 6.73 = 13.46$ K

2 Bolts = 26.92 K

SK301-11304-10 Outrigger Base Plate Assembly



SK301-11304-10 Assembly -82 Assembly Outrigger Tiedown Assembly

-1C Assembly Shear Out

Shear Area - 1.5 x 2 x 2 = 6.0 In. 2

Shear Stress = $\frac{P}{A}$ $\frac{76.3}{6}$ = 12.7 KSI

Tension at 2.0 Diameter Hole -82 Assembly

Area = $2 \times 3 = 6.0 \text{ In.}^2$

$$f_f = \frac{P}{A} = \frac{76.3}{6.0} = 12.7 \text{ KSI}$$

Double Shear Pin 2.0 Diameter A307

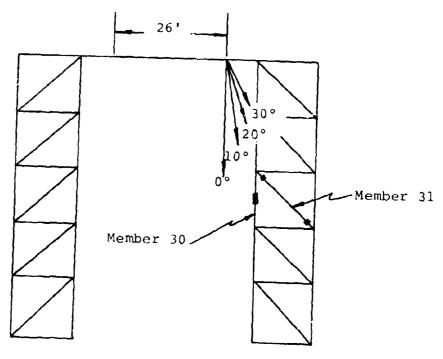
$$A = 2$$
 $r^2 = 6.28 \text{ In.}^2$

$$S = \frac{P}{A} = \frac{76.3}{6.28} = 12.14 \text{ KSI}$$

Inches of 3/8 Inch weld required.

$$\frac{76.3}{3.6}$$
 = 21.2 In.

Failure Loads @ 0°, 10°, 20° & 30° Single-Point - 26-Ft Hoist Position



Location of Failure

Failure Load 0° 537.0K	Member 30 Friction splice in main vertical column (8 WF 40)
Failure Load 10° 297.0k	Member 31 Friction splice in diagonal brace
Failure Load 20° 180.0K	Member 31 Friction splice in diagonal brace
Failure Load 30° 135.0K	Member 31 Friction splice in diagonal brace

Towers would not collapse from these failure conditions.

Failure Loads

$$F_{cr} = \frac{C \pi^{2}E}{\left(\frac{L}{r}\right)^{2}} = Critical stress$$

Use a value for c of 1.5 which corresponds to an effective length of .815L in the x-x axis.

K = 1.0 in the y-y axis

x-x Axis

$$F_{Cr} = \frac{286 \times 10^6}{\left[\frac{.815(226)}{1.25}\right]^2} = 13 \text{ KSI}$$

y-y Axis

Fcr =
$$\frac{286 \times 10^6}{1.56}$$
 = 13.6 KSI

x-x Axis is critical Fcr = 13.0 KSI for double angle struts

30° Load Angle

Moment in diagonal member - 9.10 In.K Axial load in diagonal member - 20.92K Above values are member loads as a result of a 50K load applied on fwd-aft frame.

fa =
$$\frac{P}{A} = \frac{20.92}{5.34} = 3.92 \text{ KSI}$$

fb = $\frac{M}{S} = \frac{9.10}{3.0} \text{ In }_3 \text{K} = 3.03 \text{ KSI}$
(3.03 + 3.92) x = 13.0 KSI
x = $\frac{13.0}{6.95} = 1.87$

50K(1.87) = 93.40K/Frame93.40K(2) = 186.80K total failure load

20° Load Angle

Member loads as a result of a 50K load applied on fore-aft frame:

$$M = 6.82 \text{ In. K}$$

 $P = 15.65 \text{ K}$

$$f_b = \frac{M}{S} = \frac{6.82}{3.0} = 2.27 \text{ KSI}$$

$$fa = \frac{P}{A} = \frac{15.65}{5.34} = 2.94 \text{ KSI}$$

$$(2.27 + 2.94)x = 13.0 KSI$$

$$x = \frac{13.0}{5.21} = 2.49$$

$$50K(2.49) = 125K/Frame$$

250K total failure load

10° Load Angle

Member loads as a result of a 50K load applied on fore-aft frame:

$$M = 4.12 \text{ In- K}$$

$$P = 9.5 \text{ In. } K$$

$$f_b = \frac{M}{S} = \frac{4.12}{3.0} = 1.37 \text{ KSI}$$

$$fa = P = 9.5 = 1.78 \text{ KSI}$$

$$(1.37 + 1.78)x = 13.0 KSI$$

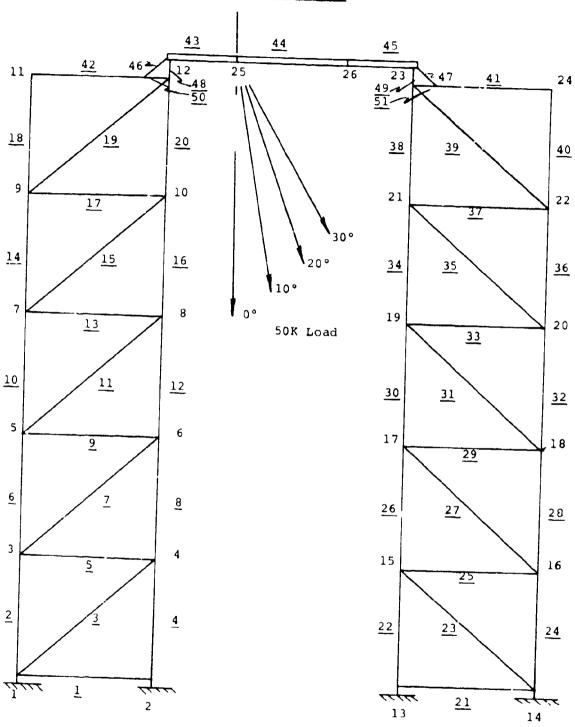
$$x = \frac{13.0}{3.15} = 4.12$$

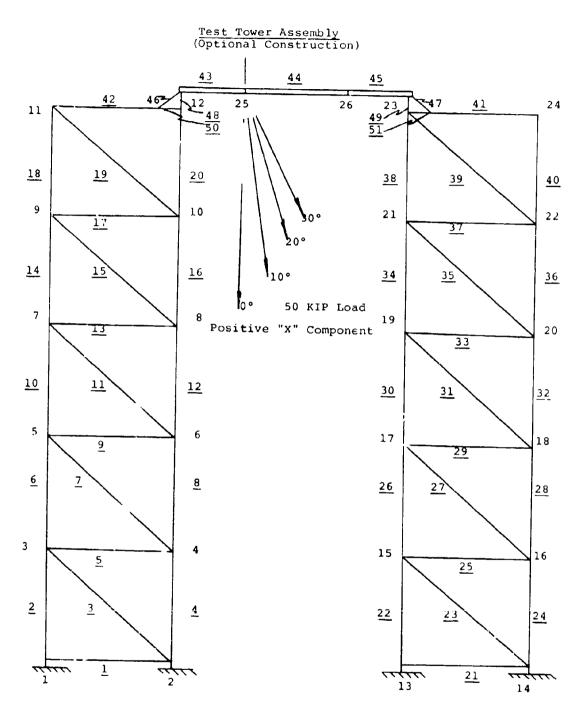
50K x 4.12 = 206 K/Frame 412K Total failure load

0° Load Angle

412K total failure load

Test Tower Assembly





Foundation Requirements

The following pages show foundation requirements in KIPS. In the load column, five separate load requirements are listed:

- Foundation requirements as a result of side wind loads.
- Foundation requirements as a result of test failure loads.
- 3. Foundation requirements as a result of positive or negative horizontal component of test load in the fore-aft direction.
- 4. Foundation requirements as a result of deadweight of overhead structure.
- Foundation requirements as a result of deadweight of tower.

All horizontal loads are considered to be reversible, and therefore maximum foundation requirements are symmetrical about \boldsymbol{c} in both X and Y planes.

Load Table VIII is the same as load Table IX except that the horizontal component of test load is reversed.

Table X shows foundation requirements as a result of the empty load container swinging into the tower at maximum possible height of 58 feet. This condition results in maximum uplift requirement.

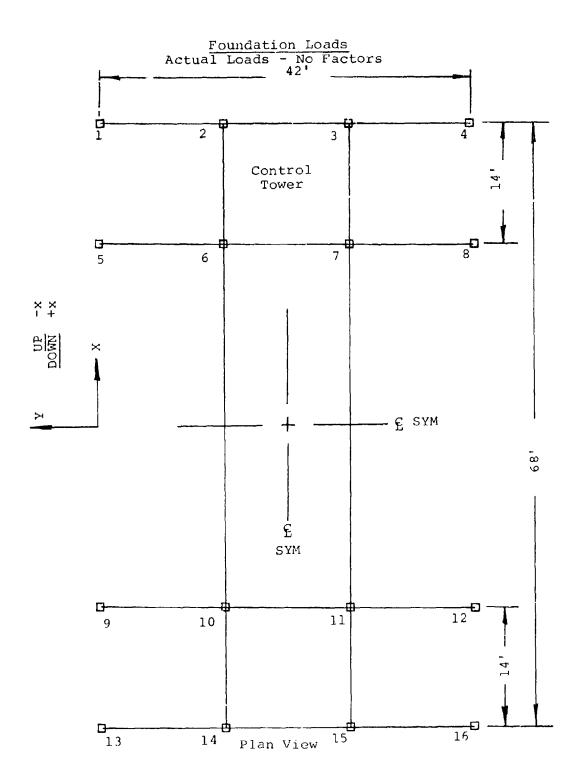


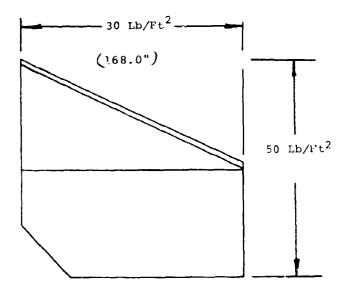
TABLE VIII. REAC	TION LO	ADS -	70-TON	FAILU	RE CON	DITION		
			Colum	n No.) (к	IPS)		
	(1		(2		(3		4	
	х	Z	Х	Z	х	2	х	Z
(1) Wind Load	2.57	6.99	.19	3.88	.24	-3.30	2.79	-7.57
(2) Test Load		4.47		14.01		10.05		3.22
(3) + Horiz.Comp.Test			-3.09	14.56	-3.09	14.56]
(4) Deadwt. Overhead				4.28		4.48		
(5) Deadwt. Tower				10.70		10.70		
TOTAL	2.57	11.46	-2.90	18.31	-2.85	7.37	2.79	-4.35
		5)	(((8	
(1)	5.17	14.02	. 42	8.18	.62	-7.53	5.40	14.67
(2)	10.73	29.12	-2.59	29.80	2.33	21.40	-1.05	2.85
(3)			.01	14.56	01	14.56		
(4)		2.85		7.12		7.48		2.99
(5)			-	10.70		10.70		- 00
TOTAL	15.90	45.99	-1.96	70.36	2.94	46.61	4.35	-8.83
TABLE IX. REACTI	ON LOA	DS - 7	0-TON 1	FAILUR	E COND	TION.		
		1)		2)		3)	(
(1) Wind Load	2.57	6.99	.19	3.88	.24	-3.30	2.79	-7.57
(2) Test Load		4.47		14.01		10.05		3.22
(3) -Horiz.Comp.Test			3.09	14.56	3.09	14.56		
(4) Deadwt. Overhead				4.28		4.48		
(5) Deadwt. lower				10.70		10.70		
TOTAL	2.57	11.46	3.28	47.43	3.33	36,49	2.79	4.35
	(<u>s</u>	(<u></u>	(<u> </u>	(3)
(1)	5.17	18.02	.42	8.18	.62	-7.53	5.40	14.67
(2)	10.73	29.12	-2.59	29.80	2.33	21.40	-1.05	2.85
(3)			.01	14.56	01	14.56		
(4)		2.85		7.12		7.48		2.99
(5)				10.70	<u> </u>	10.70		
TOTAL	15.90	45.99	-2.16	41.24	2.94	17.49	3.35	-8.83

			Col	umn No	O KI	PS		
		1)	(2		(3)	(4)
	х	Z	х	z	x	z	<u> </u>	z
(1) Impact Load		22.6		52.7		52.7	·	22.6
(2) Deadweight		15.5		11.0		11.0		5.5
TOTAL		28.1		63.7		63.7		28.
		5)	(6		(7)	(<u>B</u>
(1)		-22.6		-52.7		-52.7		-22.
(2)		5.5		11.0		11.0		5.
TOTAL		-17.1		-41.7		-41.7		-17.
		9)	((<u>1)</u>	(12)
(1)		22.6		52.7		52.7		22.
(2)		5.5		11.0		11.0		5.
TOTAL		28.1		63.7		63.7		28.
	6	3	(-	4)	Œ	3	Œ	<u></u>
(1)		-22.6		-57.7		-52.7		-22.
(2)		5.5		11.0		11.0		5.
TOTAL		-17.1		-41.7		-41.7		-17.

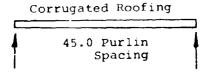
- (1) 8.0K weight moving horizontally strikes tower at a velocity of 20.6 Ft/Sec. Resulting reaction at base of tower is shown in KIPS.
- (2) Deadweight of overhead steel and deadweight of tower acting at base of tower in KIPS.

	TABLE XI DEA	DWEIGH	TS.	
Dwg. No.	Nomenclature	Qty.	Dead Wt/Assy.	Total Dead Wt.
SK301-11304-1	Tower Assy.	1	38723	3 8723
SK301-11304-2	Tower Assy.	1	42750	42750
SK301-11302-1	Hoist Module	2	2470	4940
-2	Davit Mount	1	6207	6207
-3	Module Support	2	6274	12548
-4	Platform Assy.	1	831	831
-6	Platform Assy.	1	831	831
SK301-11302-5	Beam Assy.	2	8081	16162
		_		
Test Hoist		2	1700	3400
Aux. Hoist		1	3365	3365
Mast		1	3965	3965
TOTAL				133,722#

Control Room Shelter



Control room shelter designed for a $50-Lb/Ft^2$ wind load on projected frontal area. Roof is designed for $30-Lb/Ft^2$ snow load on vertical projected area.



Standard 2-2/3 x 1/2 Galvanized Corr. Sheet

$$M = \frac{W1^2}{8} = \frac{2.5 \text{ Lb/In.}(45)^2}{8}$$

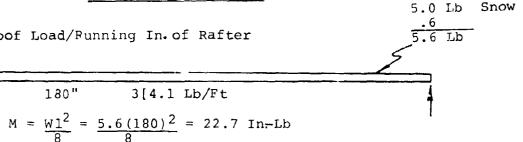
$$M = .634 \text{ In. K}$$

$$f = \frac{M}{S} = \frac{.634}{.0532} = 11.9 \text{ KSI}$$

 $S = .0532 \text{ In}^{3} \text{ per ft of width}$

Control Room Shelter

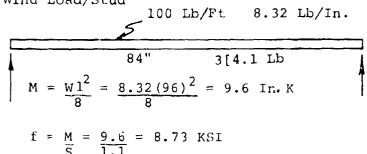
Roof Load/Punning In. of Rafter



$$f = M = \frac{22.7}{5}$$
 In. $K = 20.6$ KSI

180" 3[4.1 Lb/Ft

Wind Load/Stud

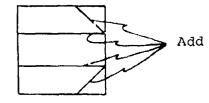


250 Lb/Ft. 20.8 Lb/Ft

$$168.0 3 \times 3 \times 3/16 \text{ Sq Tube}$$
 $M = \frac{\text{W}1^2}{8} = \frac{20.8(168)^2}{8} = 73.3 \text{ In. K}$

$$\bar{f} = \frac{M}{S} = \frac{73.3}{1.732} = 42.3 \text{ KSI}$$

Add Horizontal Braces @ 8-Ft Level

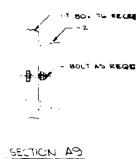


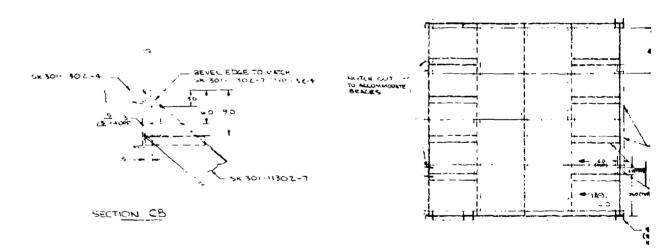
APPENDIX II DRAWINGS - DESIGN LAYOUTS

The integrated tesm rig is described by the following drawings which are provided as the listed figures.

Figure	Drawing No.	Title
49	SK301-11277	Structures Installation - HLH Hoist Test
50	SK301-11304	HLH Hoist Tower Assembly
51	SK301-11302	HLH Overhead Assembly
52	X72-002-AS-YD3/1	Site Plan - Paving and Utilities, Plan and Sections
53	X72-002-AS-YD3/2	Foundations - Plans, Sections and Details
54	X73-003-AS-YD3/2	Pneumatic Power Generator Shelter HLH/ATC Cargo Handling System
55	X73-003-AS-YD3/3	Control Room Plans, Elevations and Section
56	X73-003-AS-YD3/4	Control Room Section and Details
57	V73-C03-AS-YD3	Removable Handrail Details
58	X73-003-M-YD3/1	Piping Arrangement, Test Tower Top Section
59	%73-003-M-YD3/2	Piping Arrangement, Test Tower Base
60	X73-003-M-YD3/3	Fuel Piping Arrangement - PPG Unit
61	V73-001-E-YD3	Electrical Single Line Diagram
62	X73-003-E-YD3/1	Electrical HLH/ATC Cargo Handling Test Rig
63	X73-003-E-YD3/2	Control Room Electrical Layout
64	SK301-11564	Load Controlling Crewman Platform - Integrated Test Rig

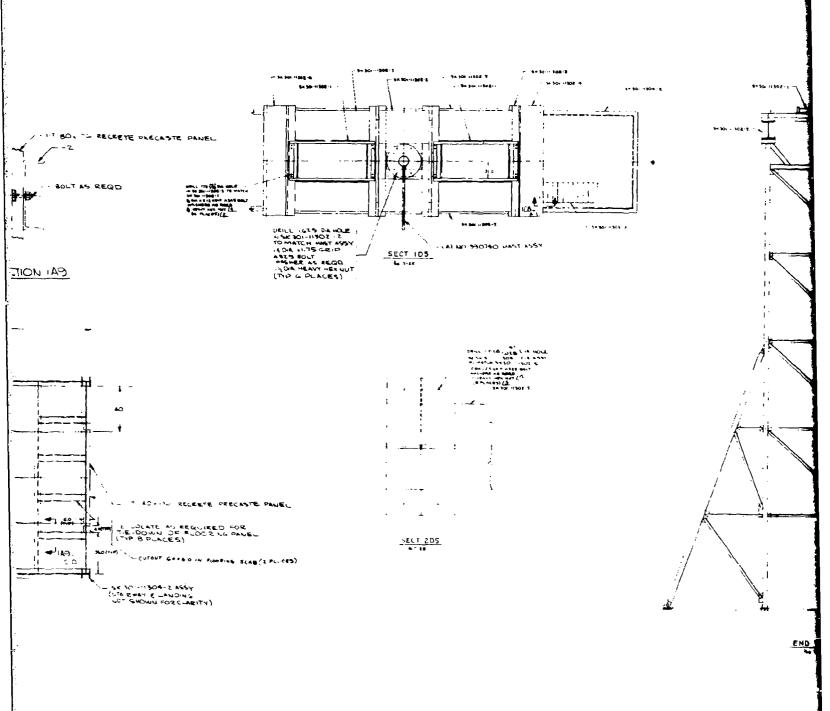
Figure	Drawing No.	<u>Title</u>
65	SK301-11676	System Test - Drawing Tree
66	ST40972	Lifting Sling, Hoist/Module
67	ST51273-1	Hoist Lifting Fixture
68	SK301-11694	Integrated Test Rig - System Wiring
69	ST30861	Instrumentation Drawing Tree





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Figure 49. Structures Installation - HLH Hoist Test.



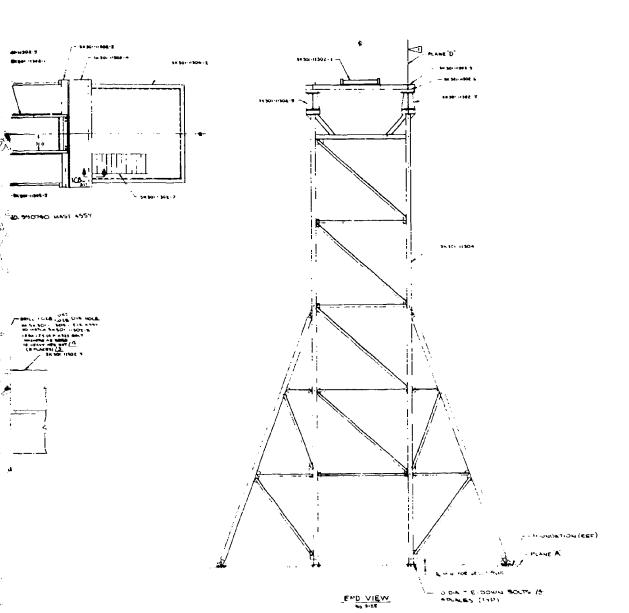
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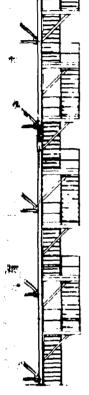


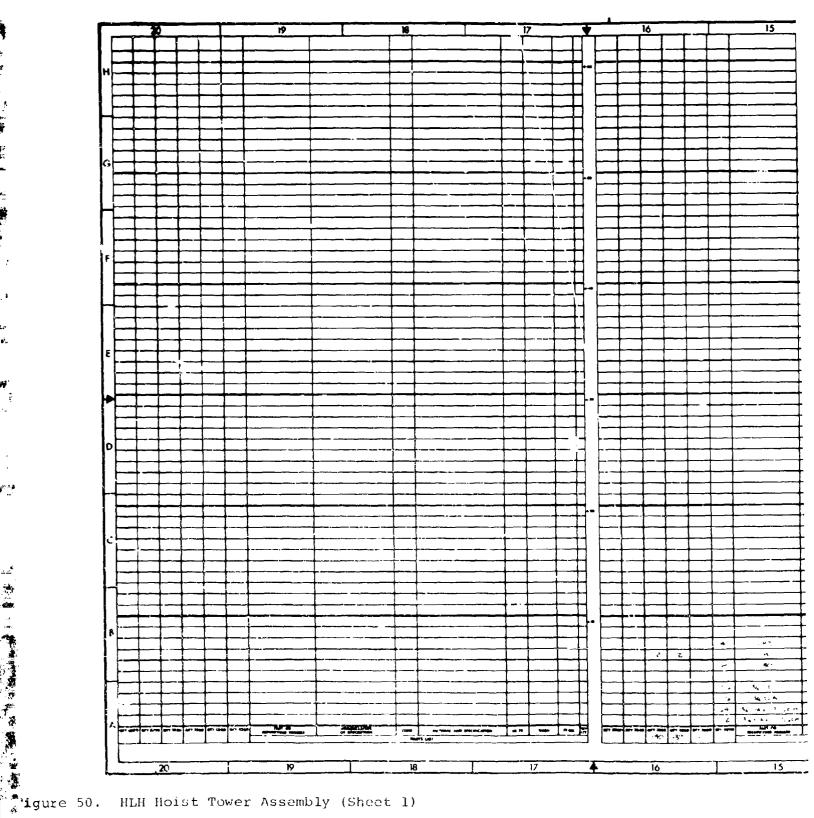
GENERAL NOTES:

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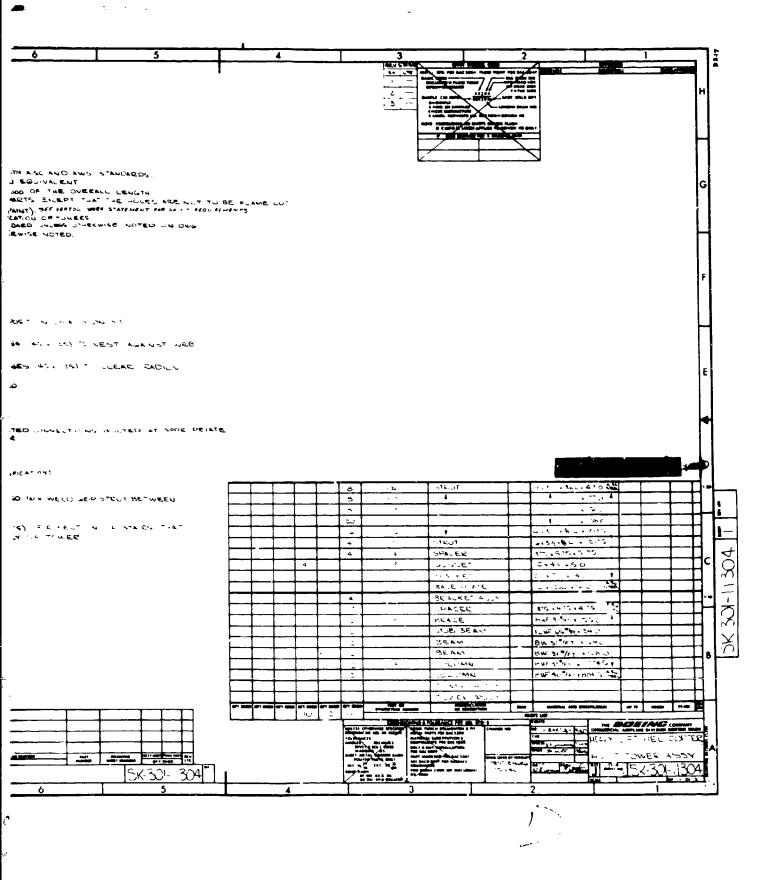
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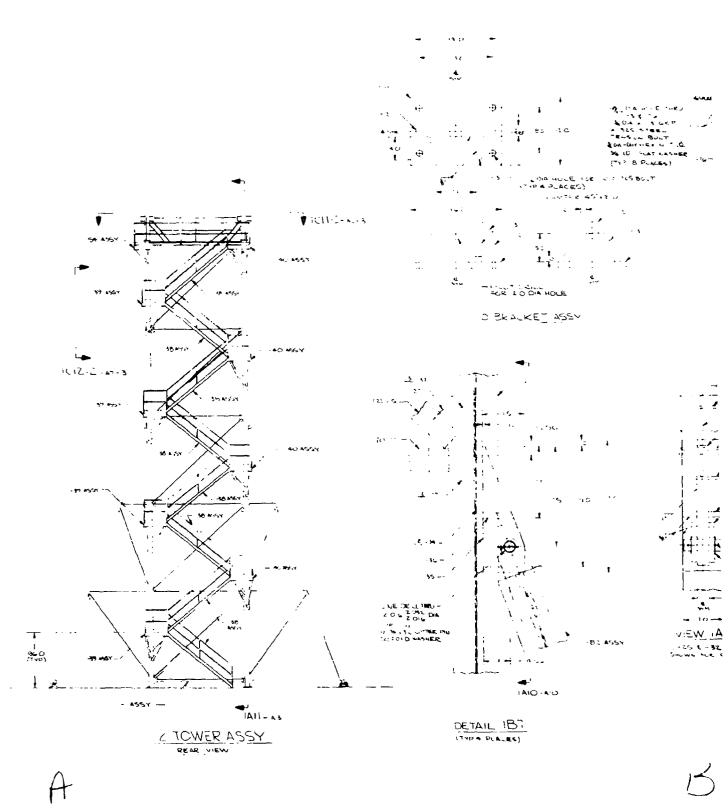
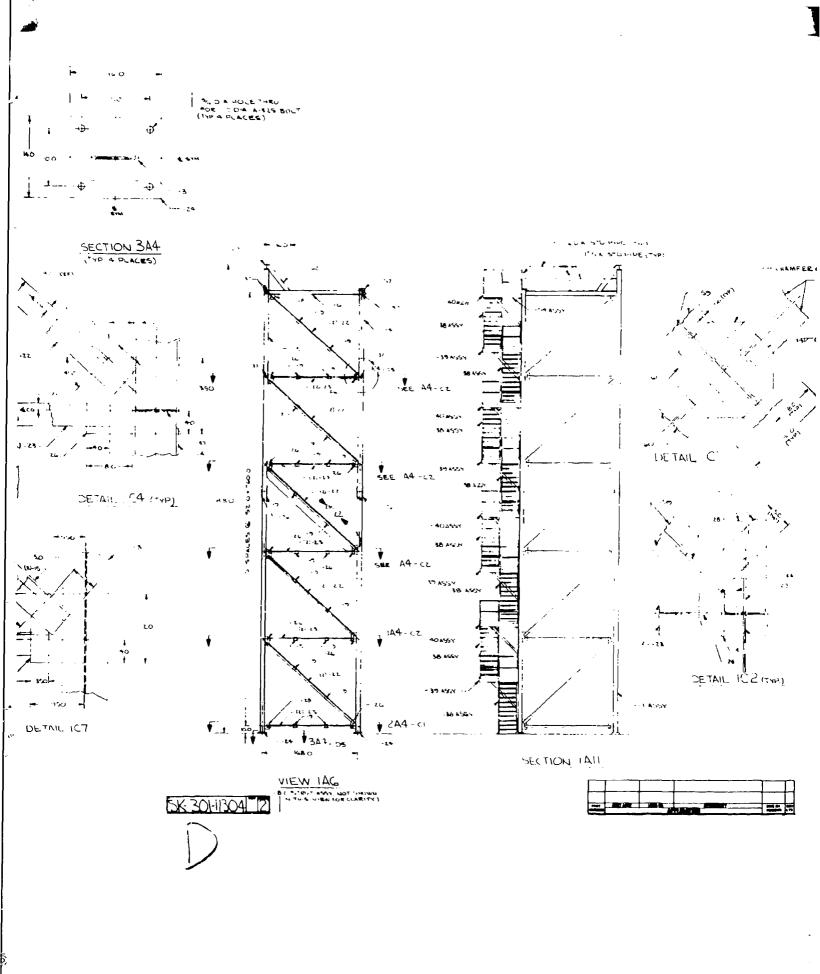
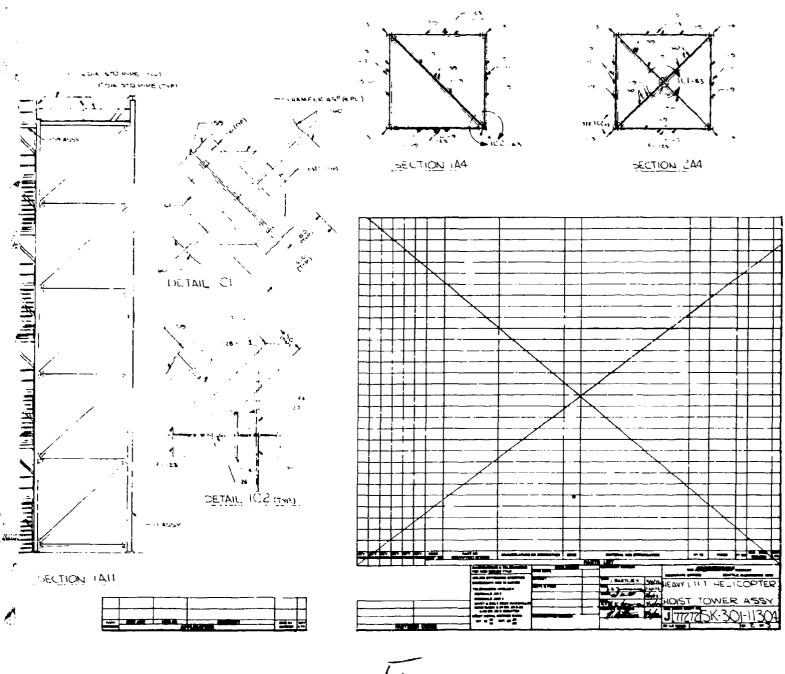


Figure 50. HLH Hoist Tower Assembly (Sheet 2).

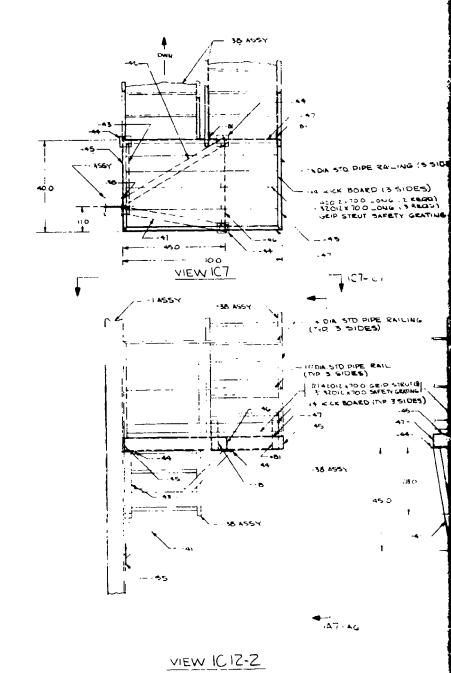
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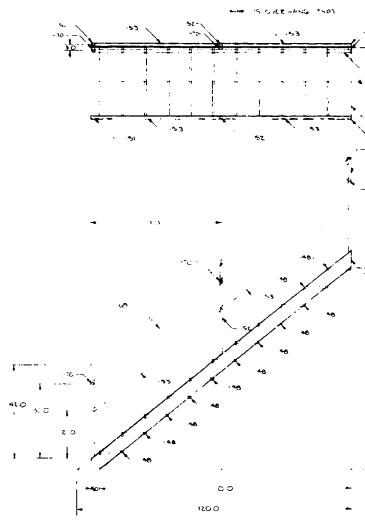
Figure 50. HLH Hoist Tower Assembly (Sheet 3).

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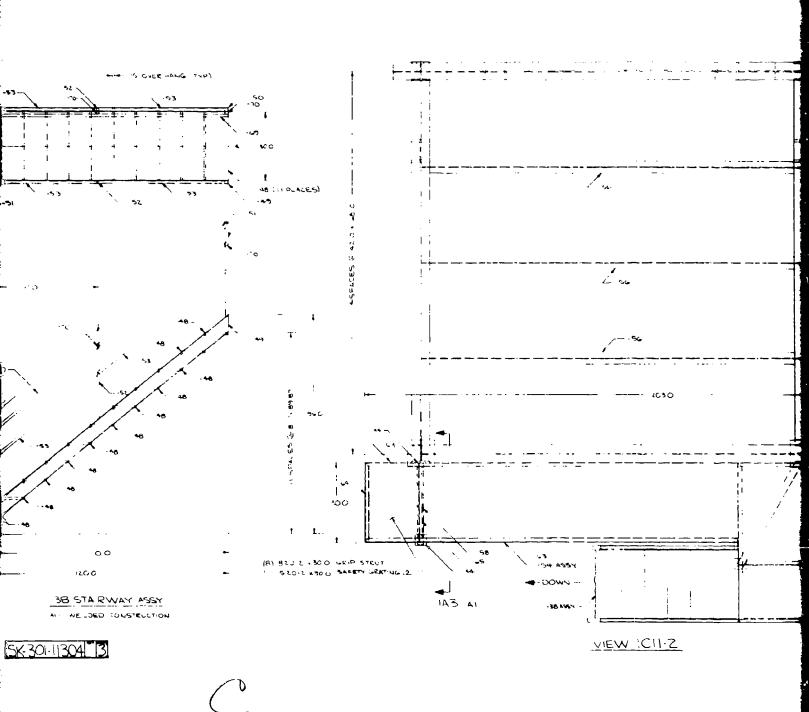
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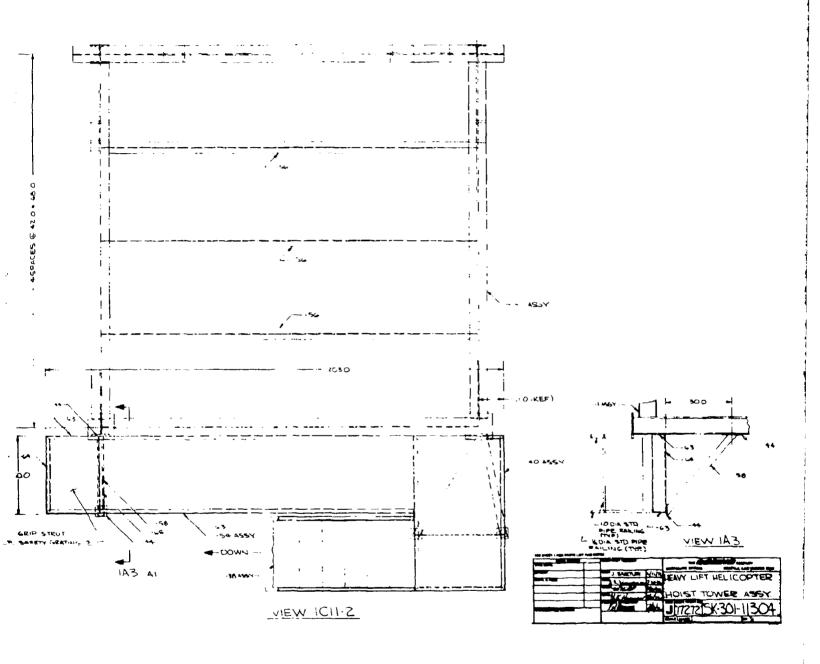


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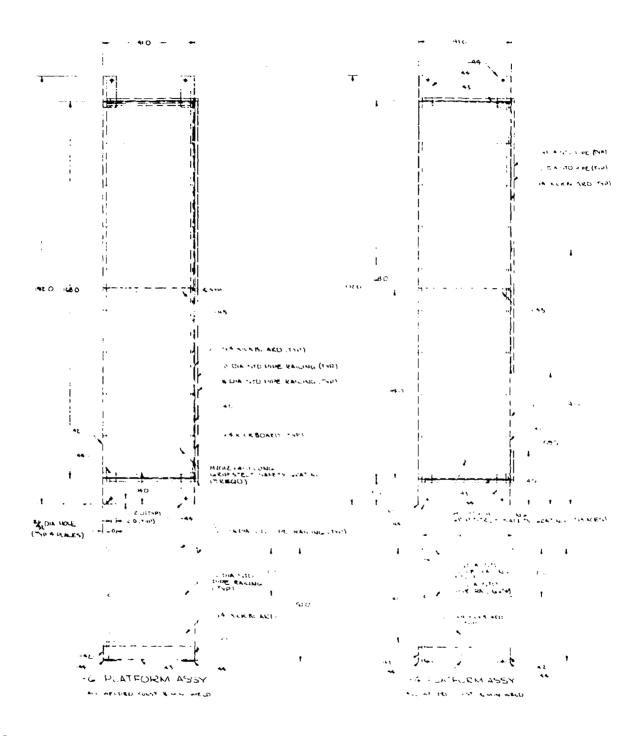
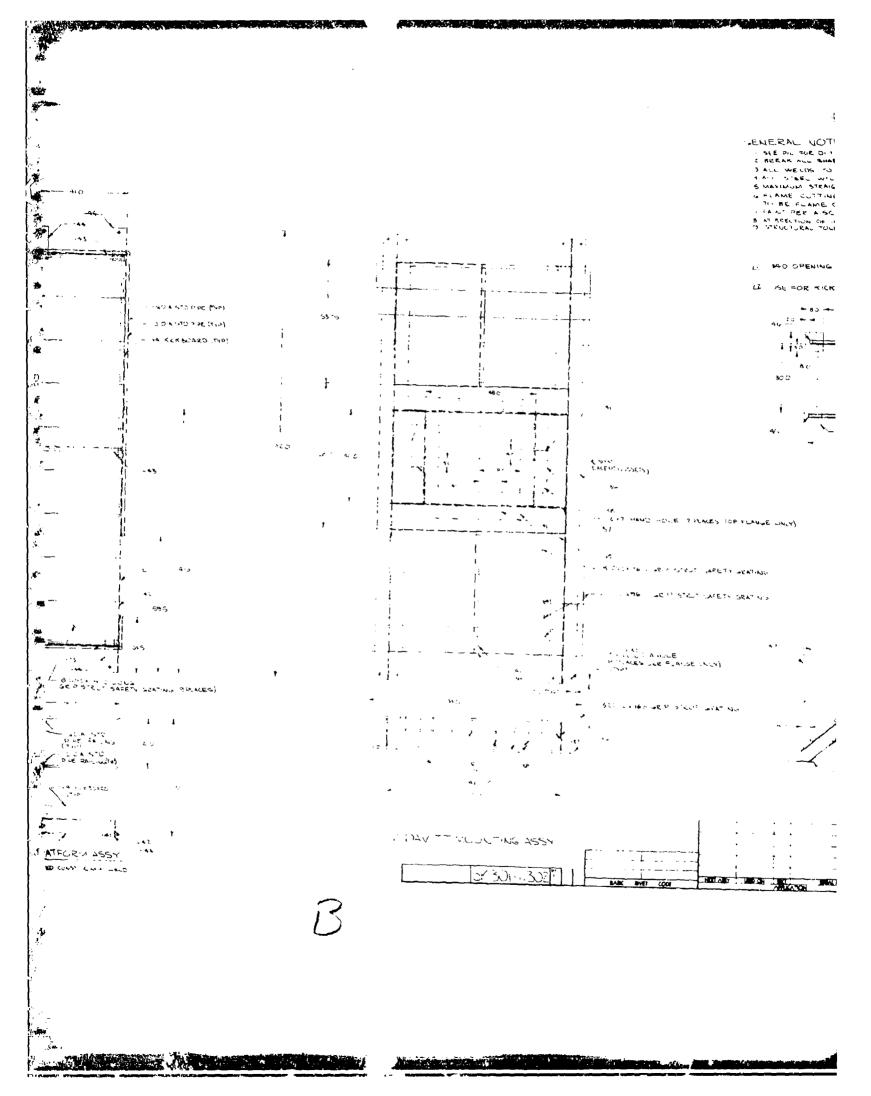
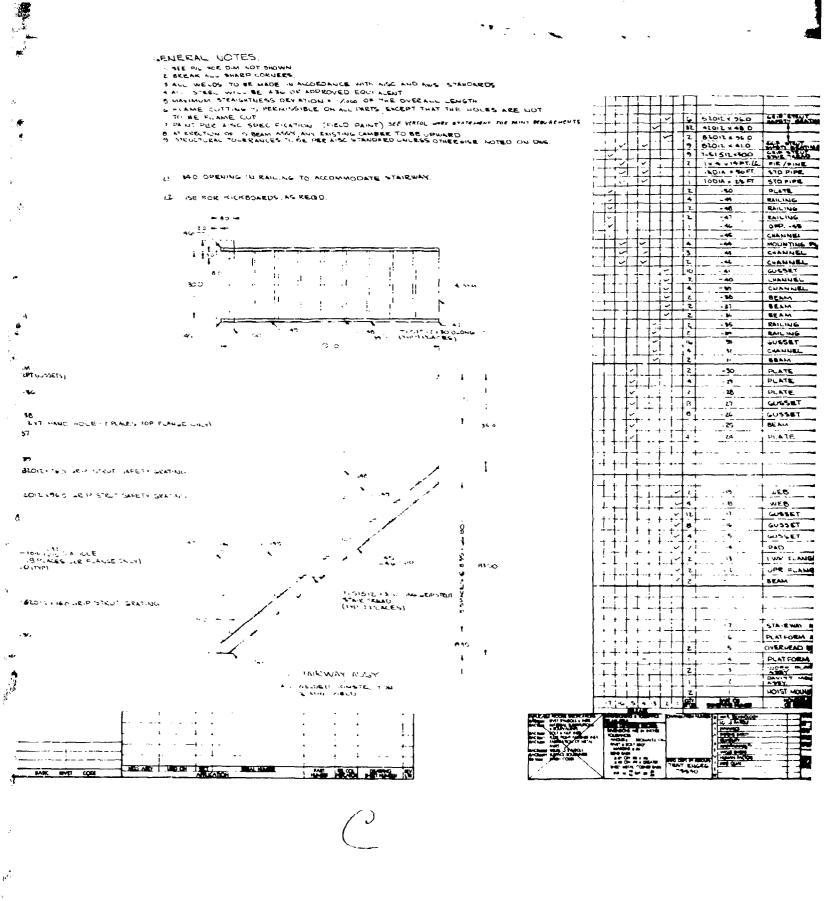


Figure 51. HLH Overhead Assembly.





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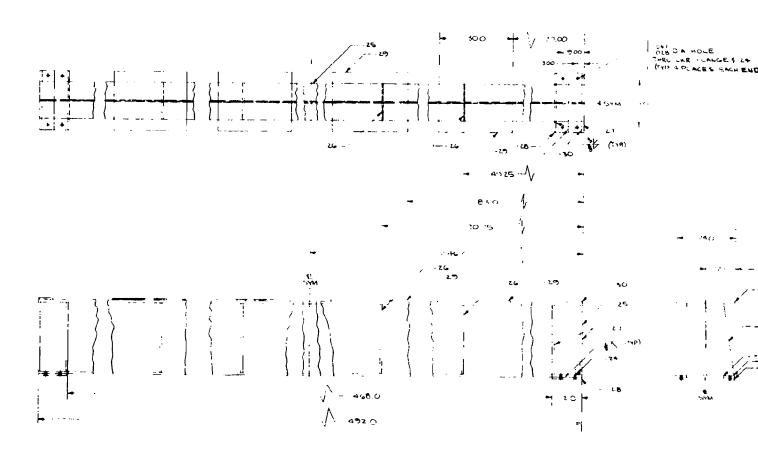
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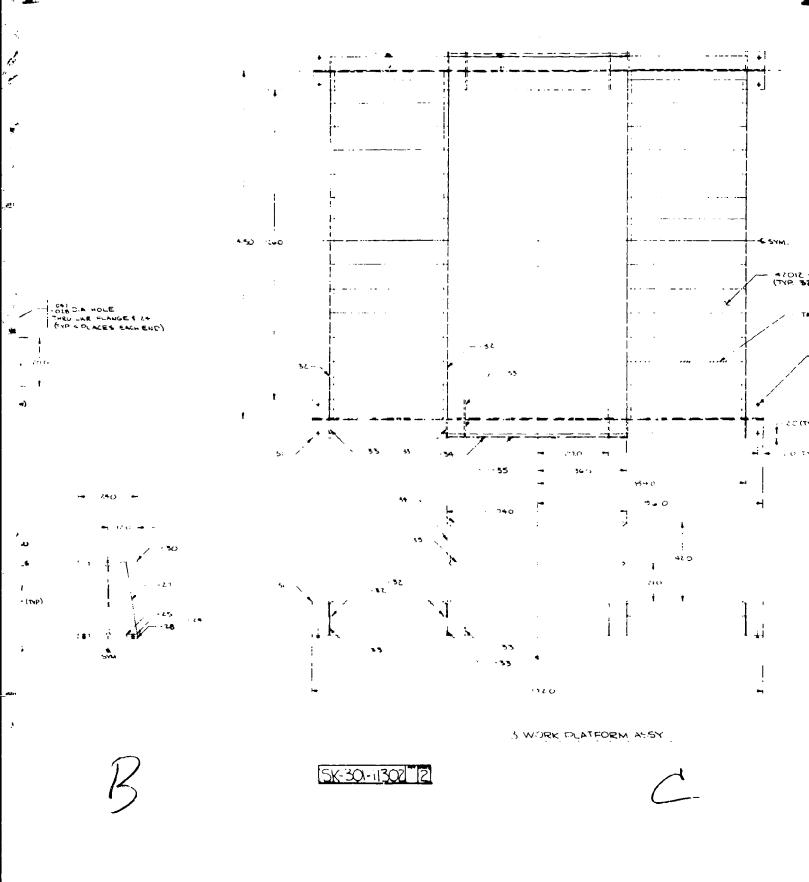


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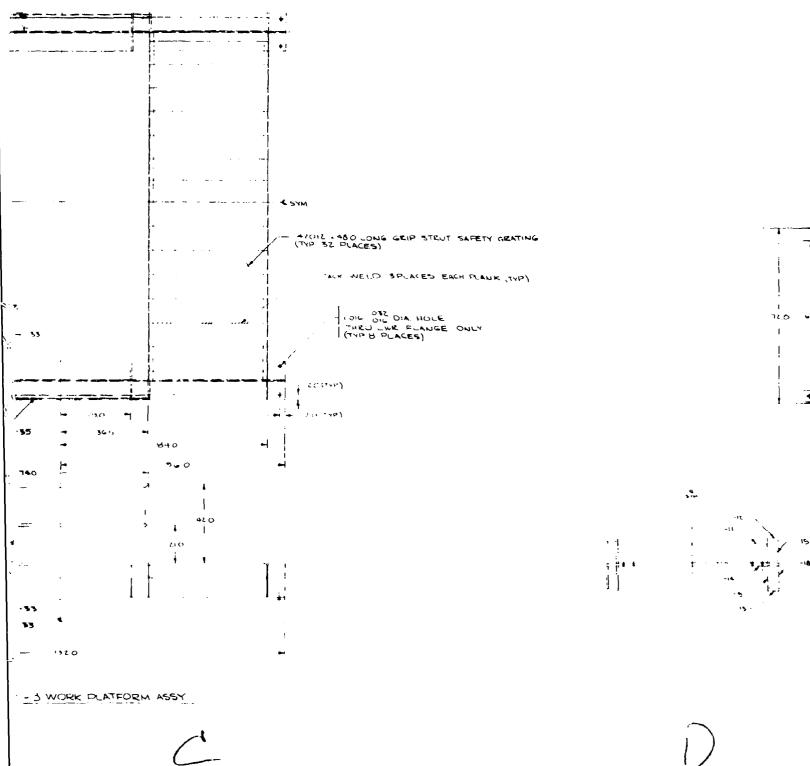
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Figure 51. HLH Overhead Assembly (Sheet 2).

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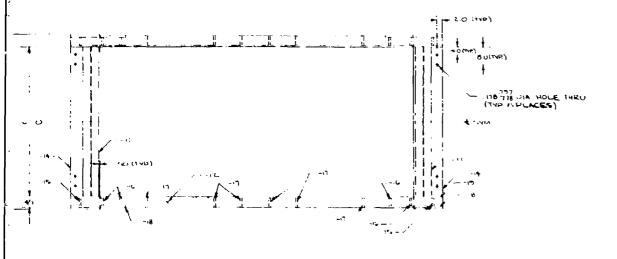


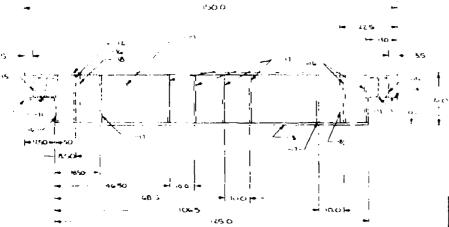
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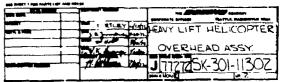
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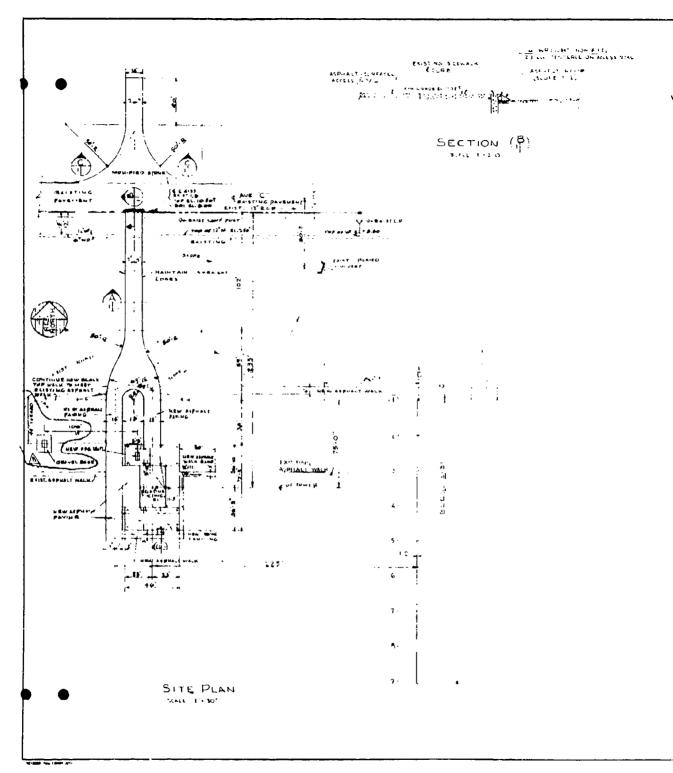
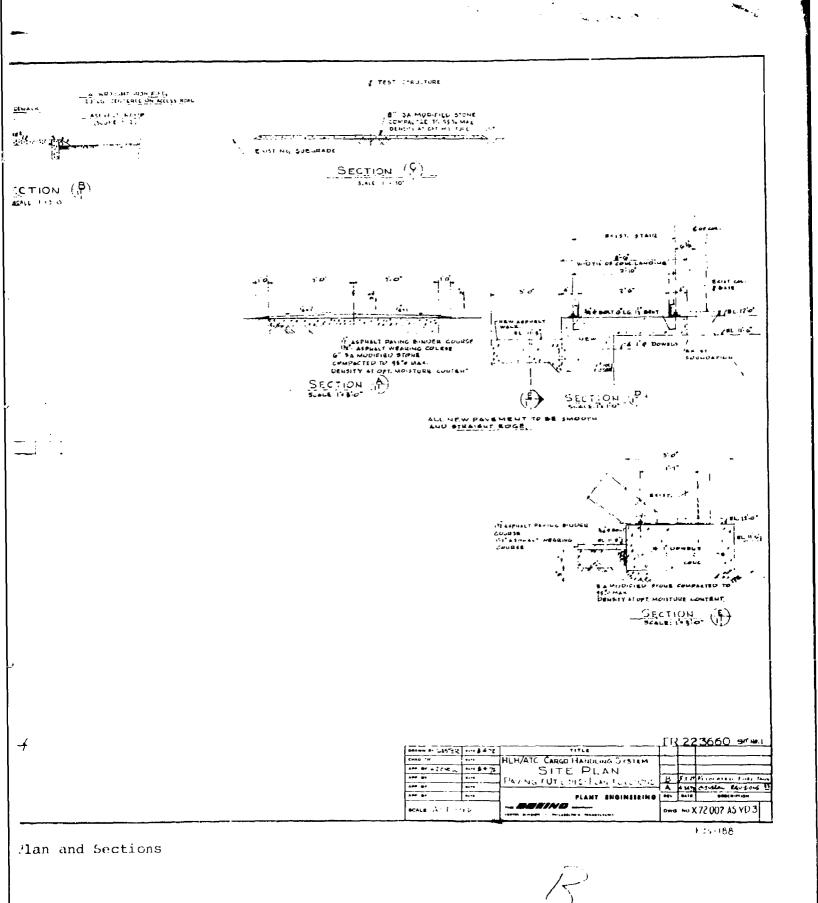


Figure 52. Site Plan - Paving and Utilities, Plan and Sections



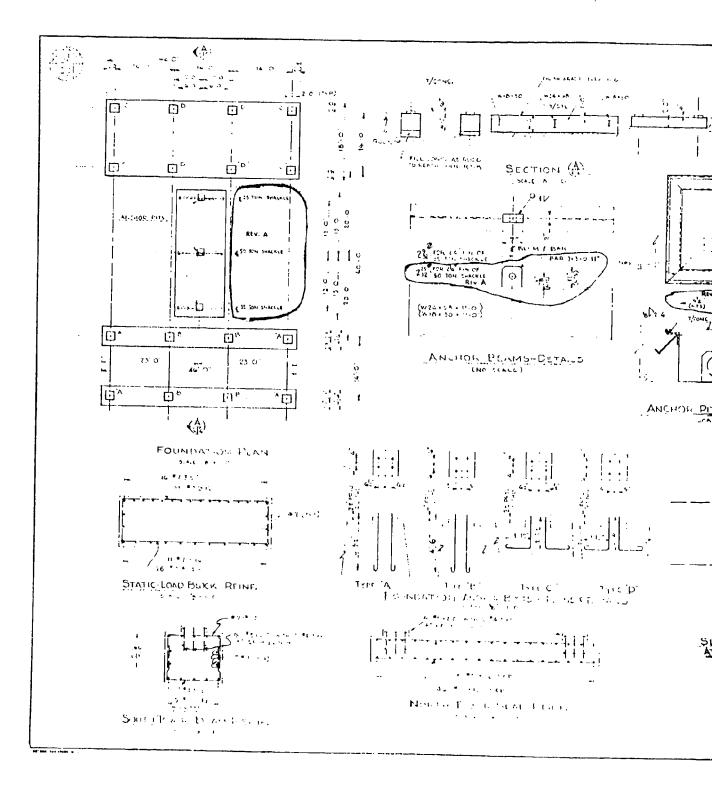
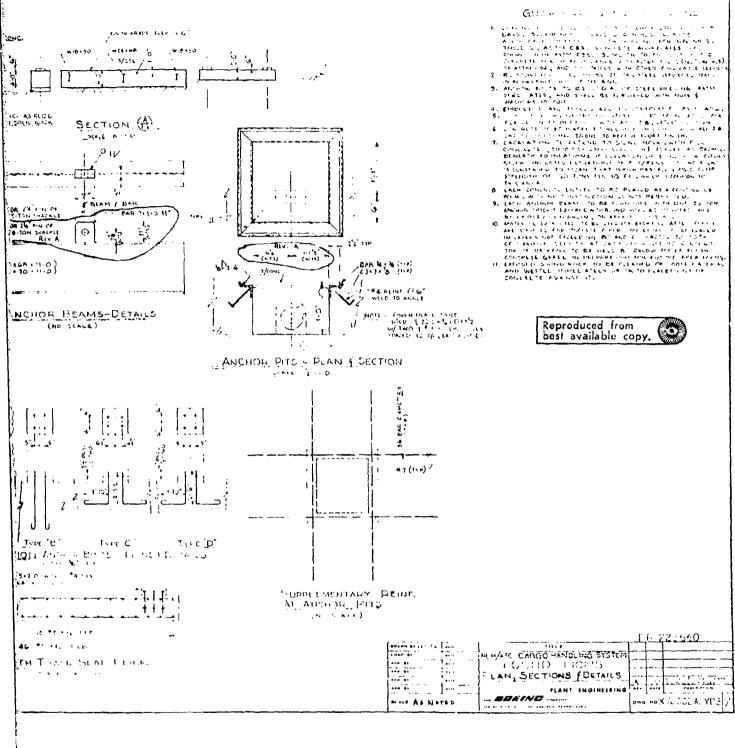


Figure 53. Foundations - Plans, Sections and Details.

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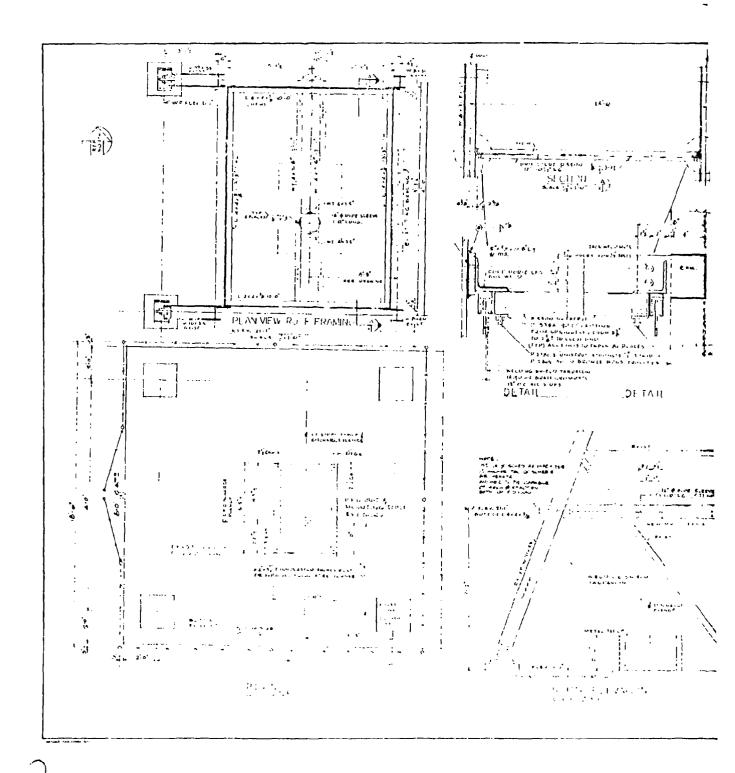
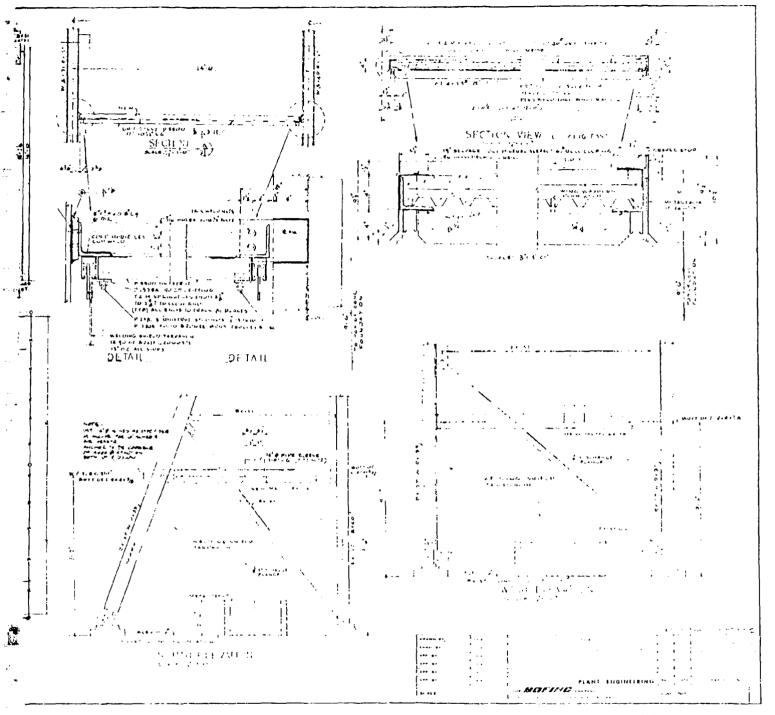


Figure 54. Pneumatic Power Generator Shelter, HLH/ATC Cargo Handling System.



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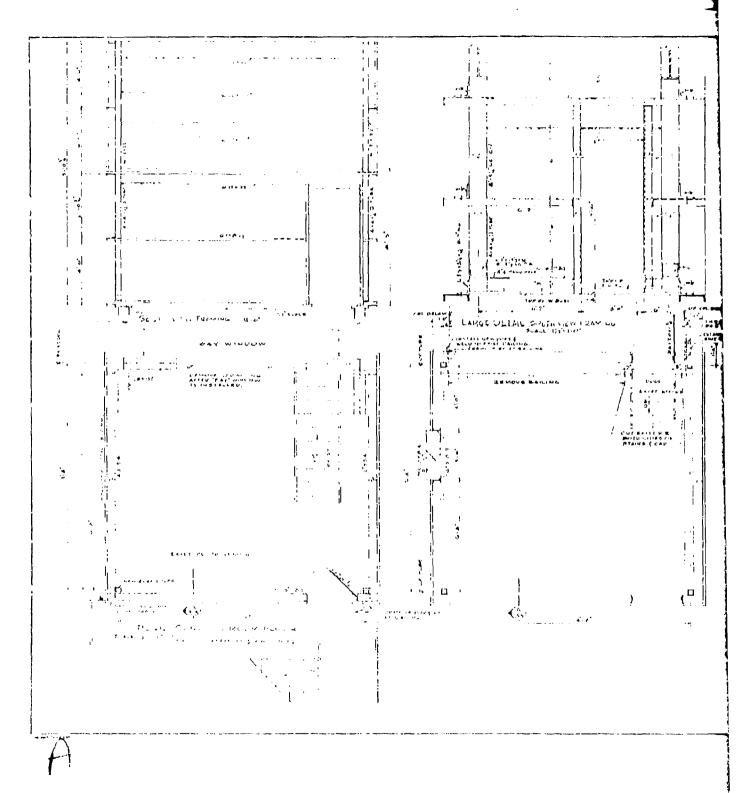
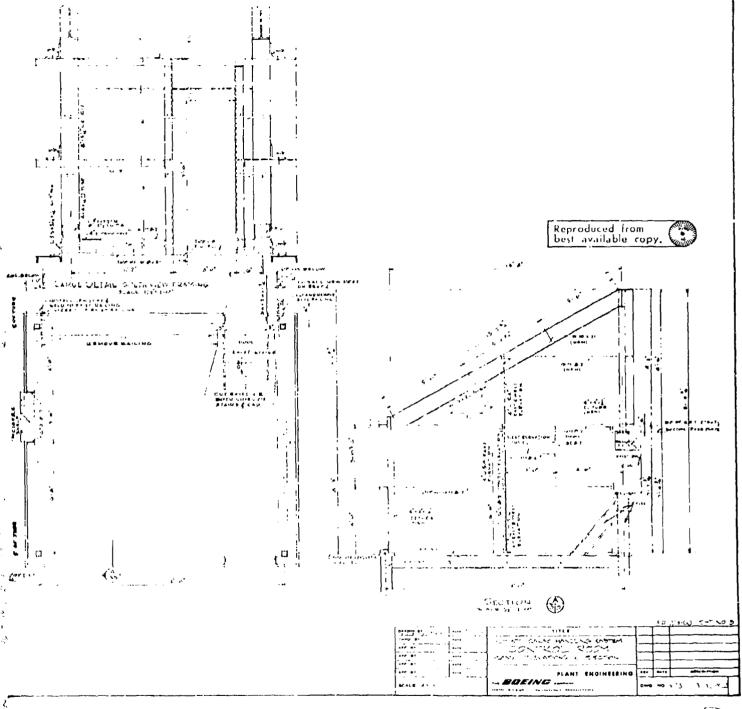


Figure 55. Control Room Plans, Elevations and Section.



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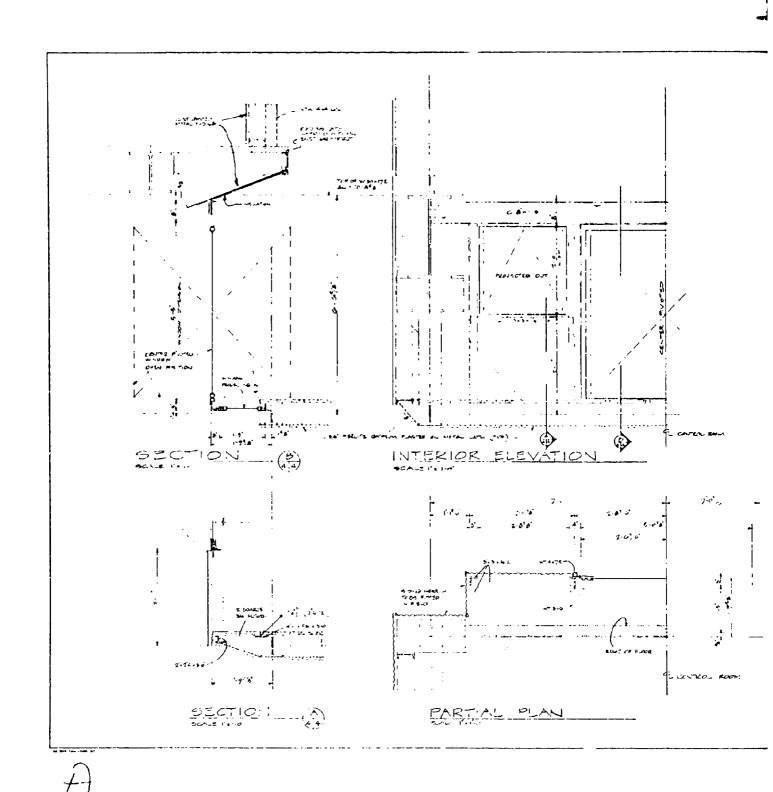
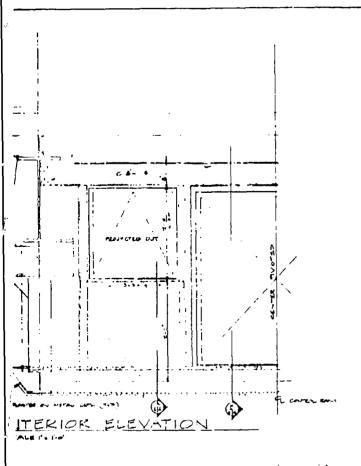


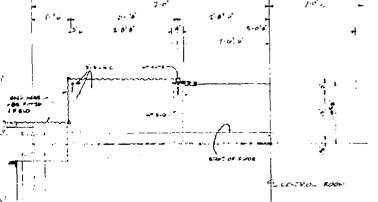
Figure 56. Control Room Section and Details.



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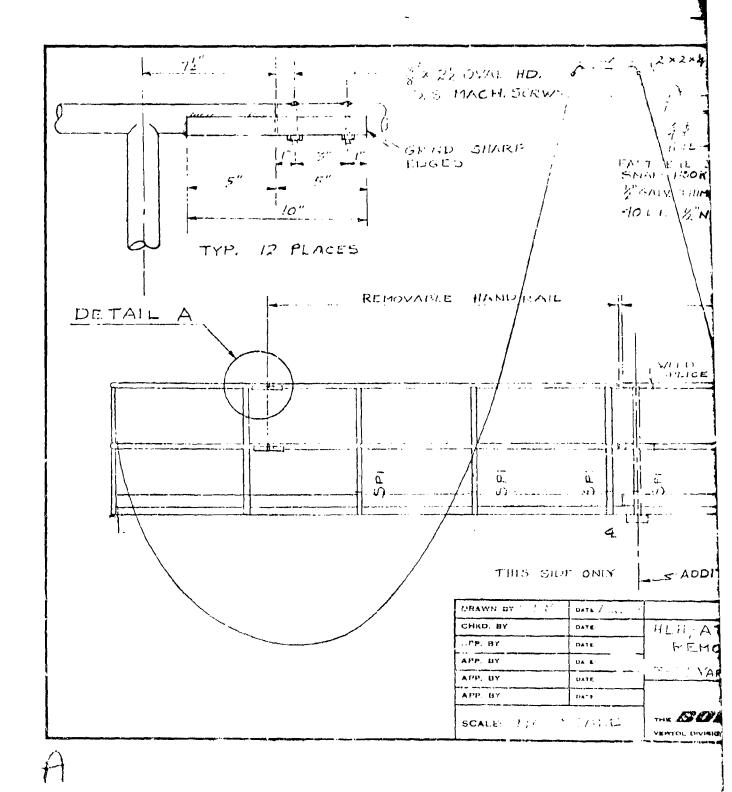


Figure 57. Removable Handrail Details.

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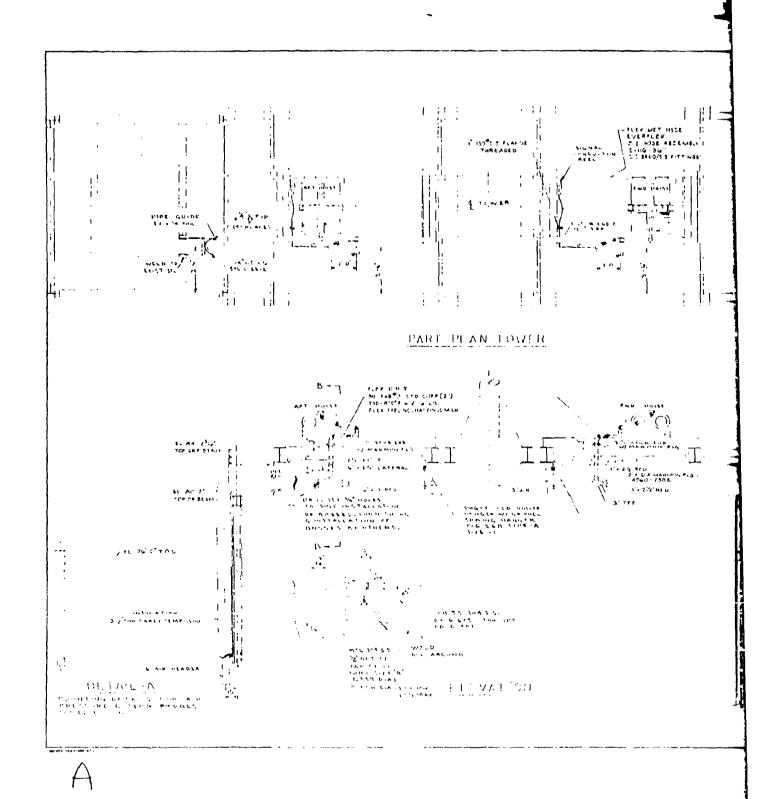
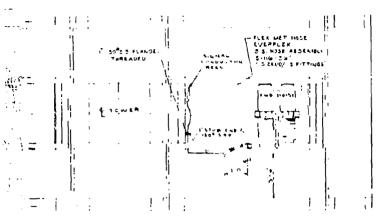


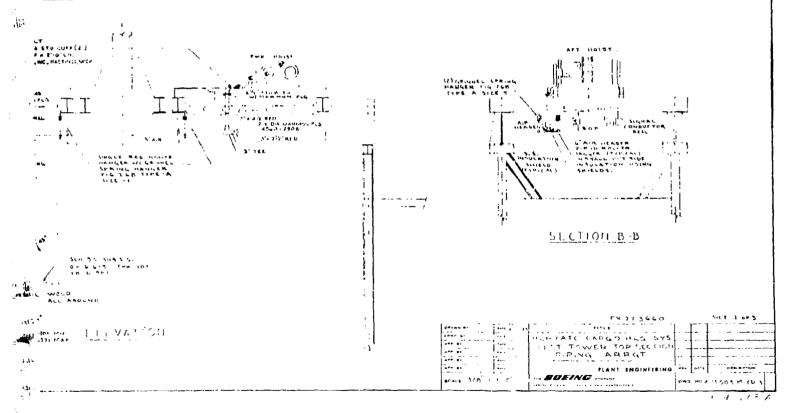
Figure 58. Piping Arrangement, Test Tower Top Section.



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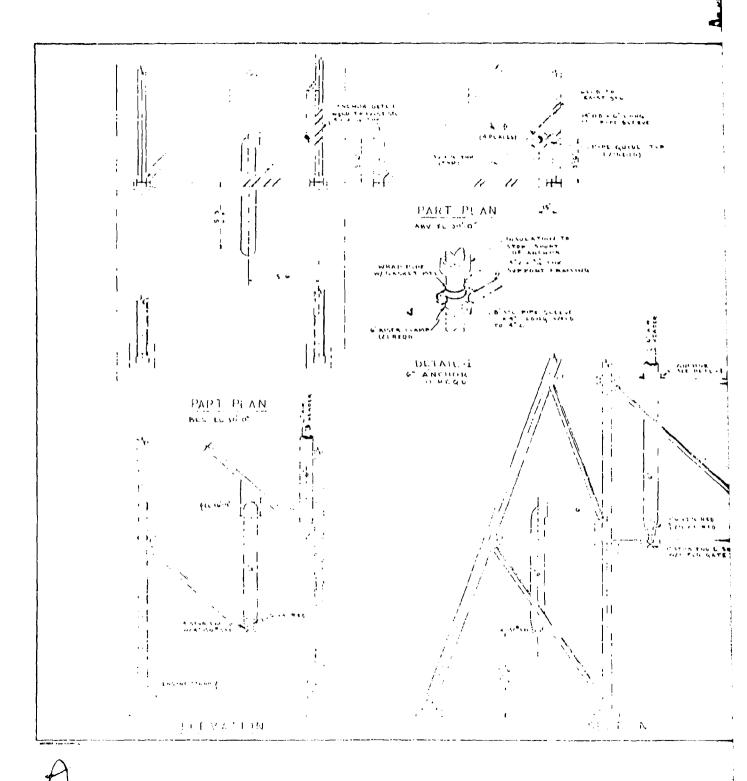
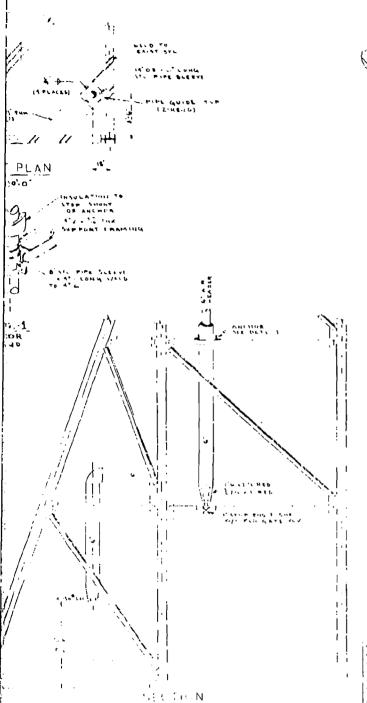


Figure 59. Piping Arrangement, Test Tower Base.



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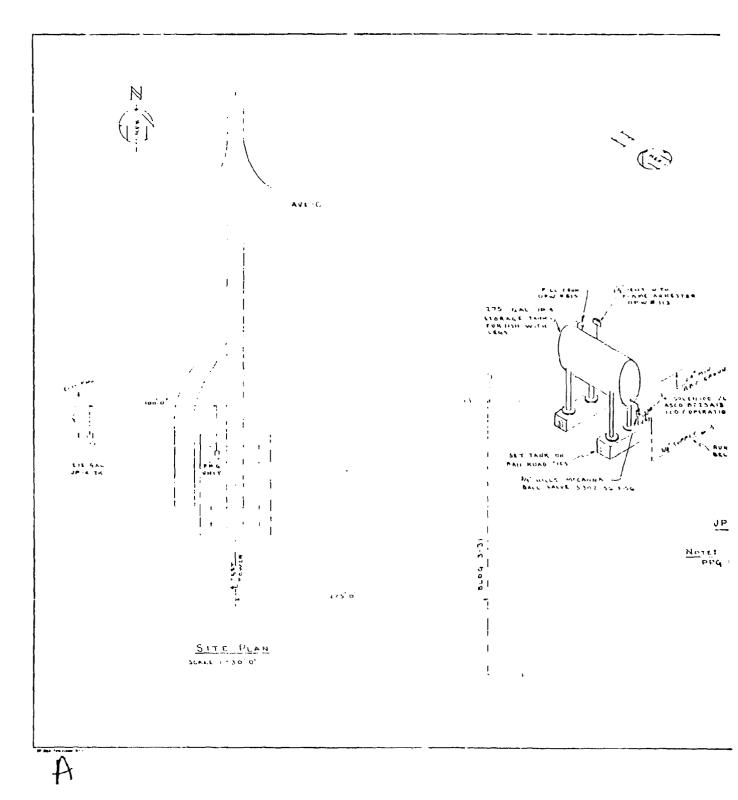
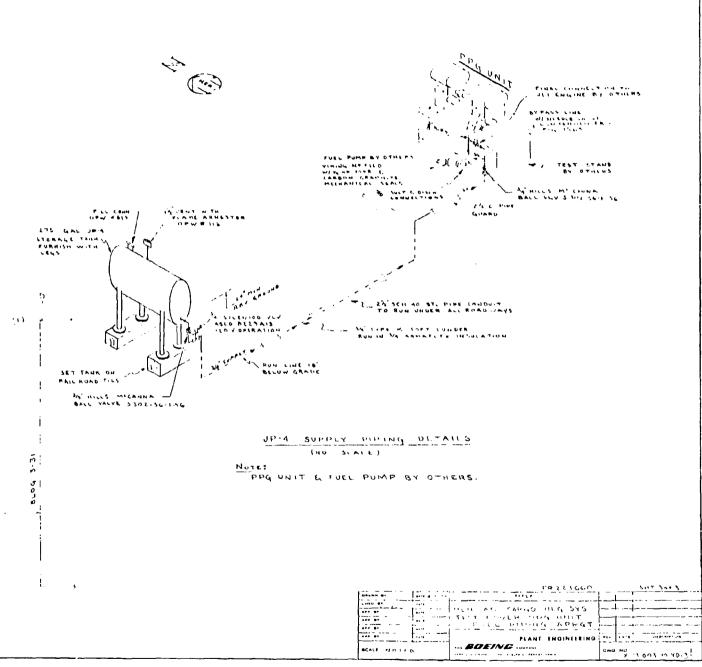
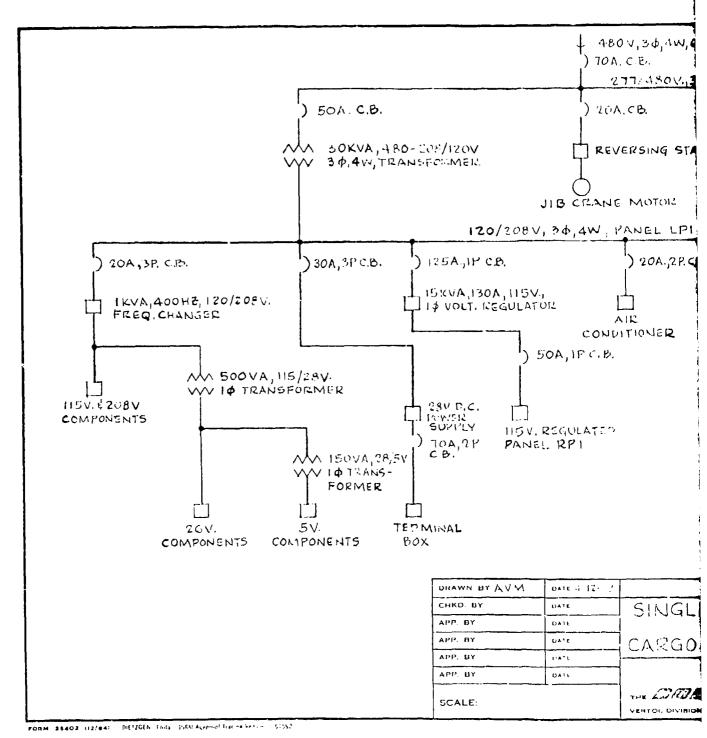


Figure 60. Fuel Piping Arrangement - PPG Unit.



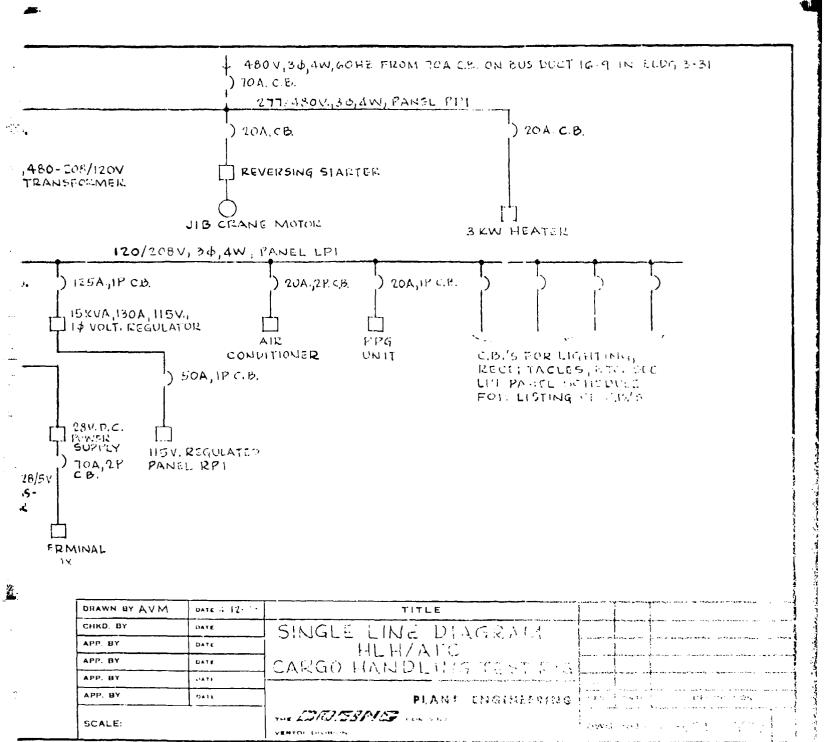
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Figure 61. Electrical Single Line Diagram.



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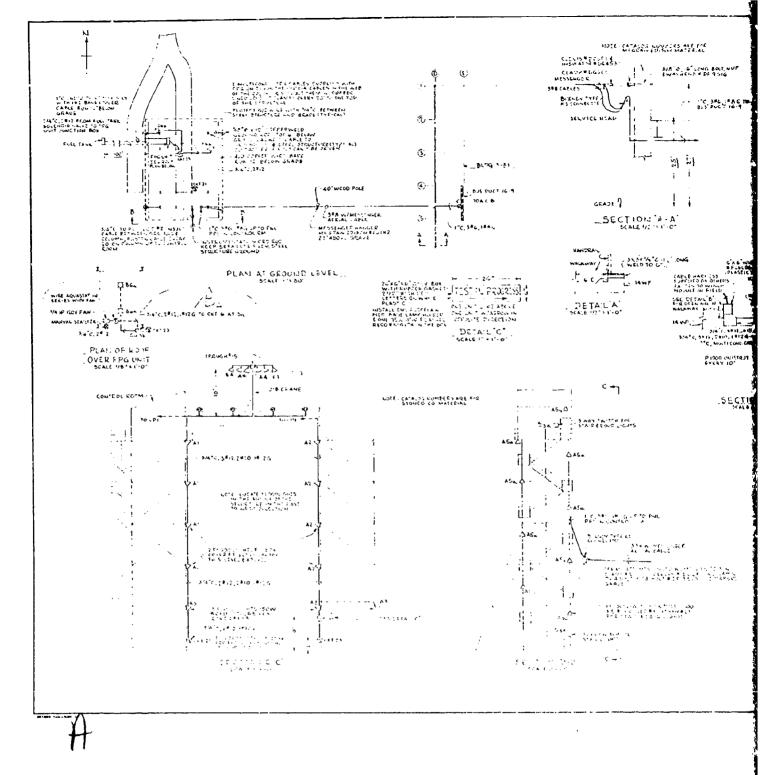
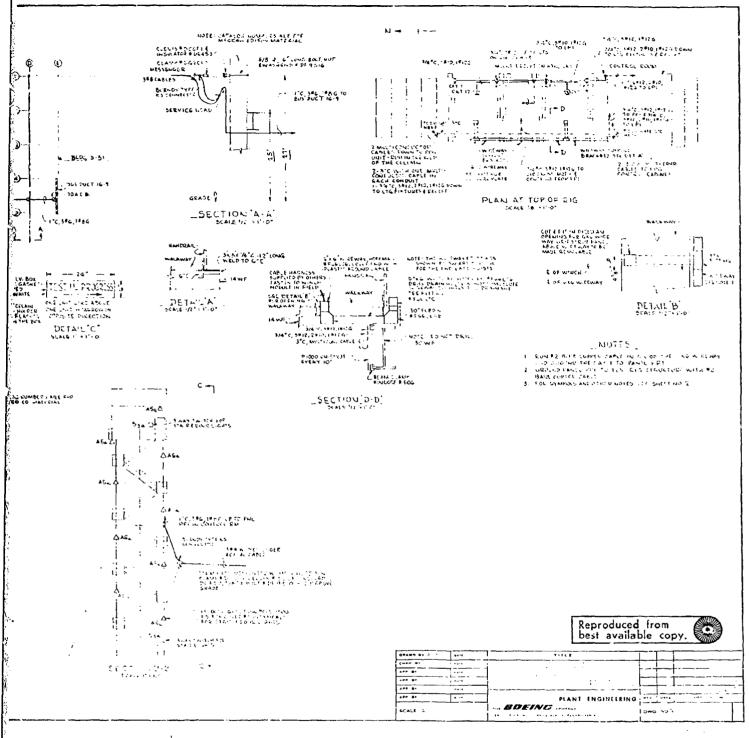


Figure 62. Electrical HLH/ATC Cargo Handling Test Rig.

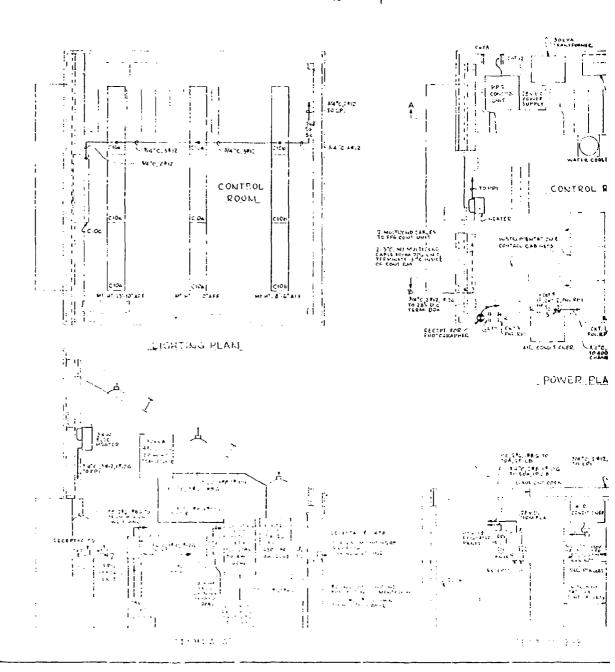
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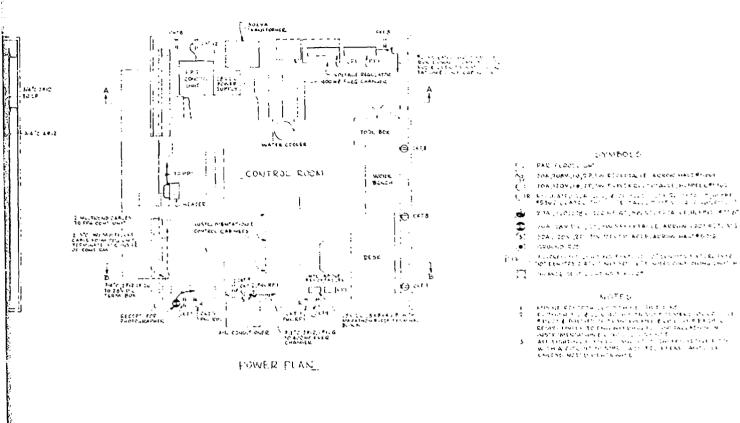




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Figure 63. Control Room Electrical Layout.

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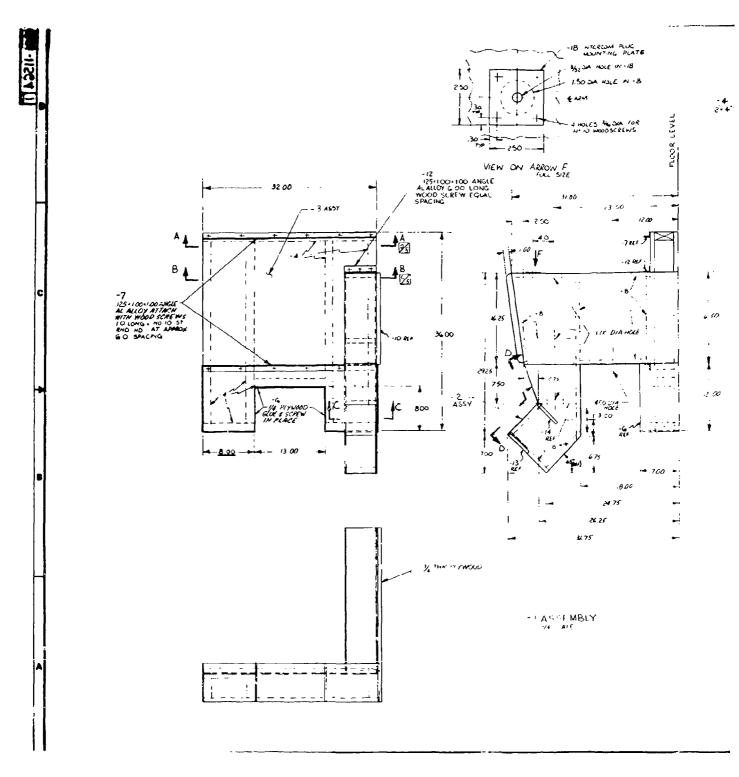
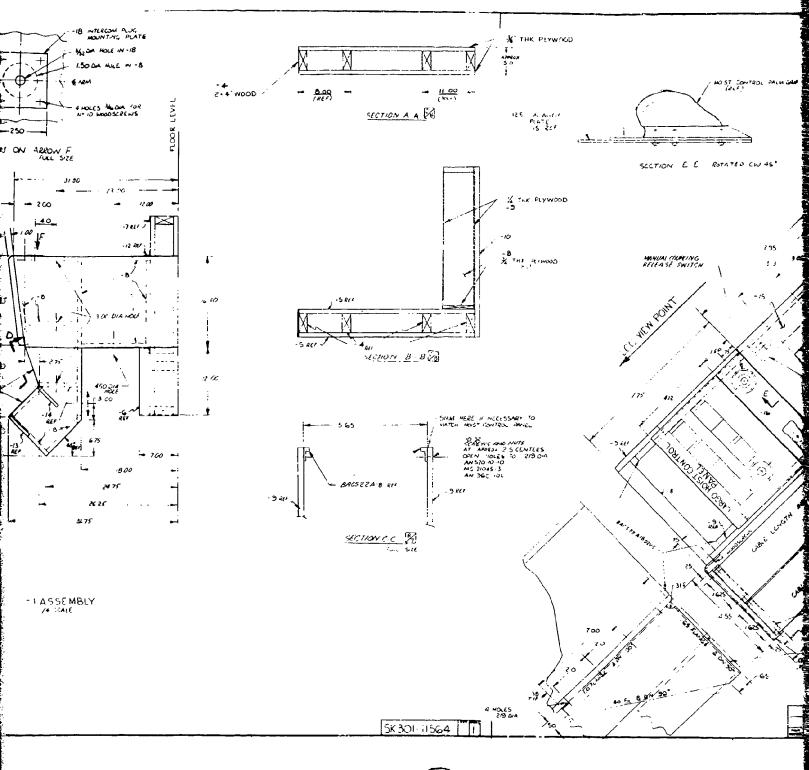


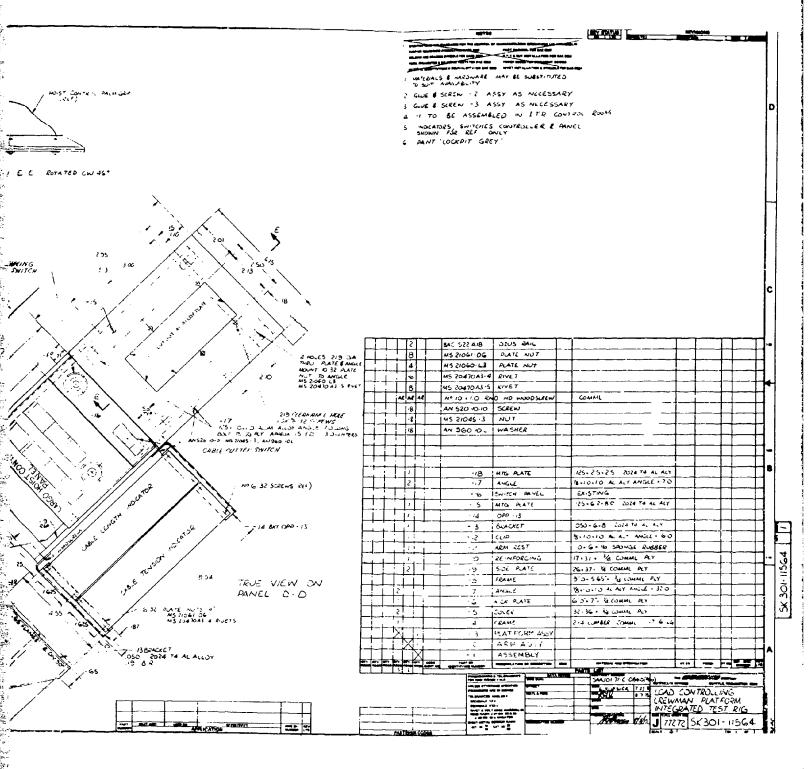
Figure 64. Load Controlling Crewman Platform - Integrated Test Rig. 173

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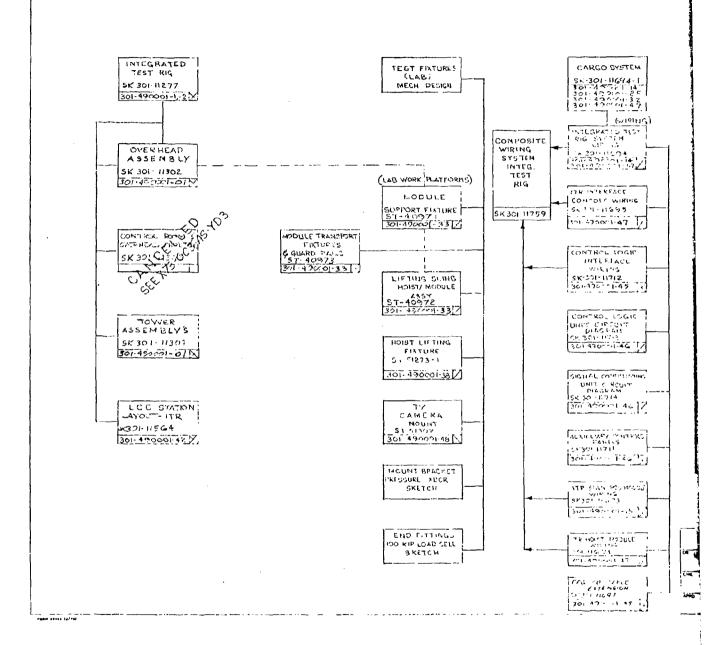
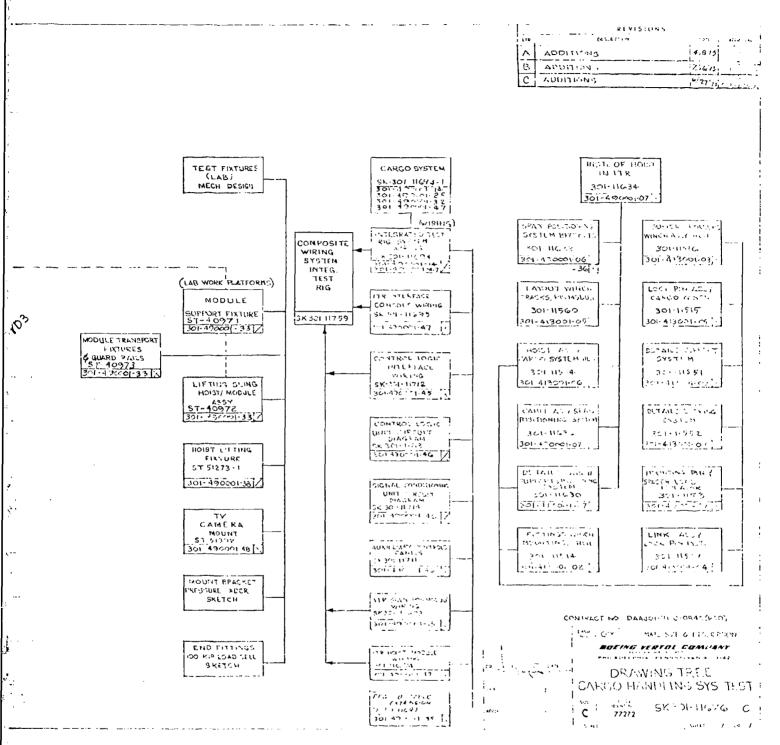


Figure 65. System Test - Drawing Tree.

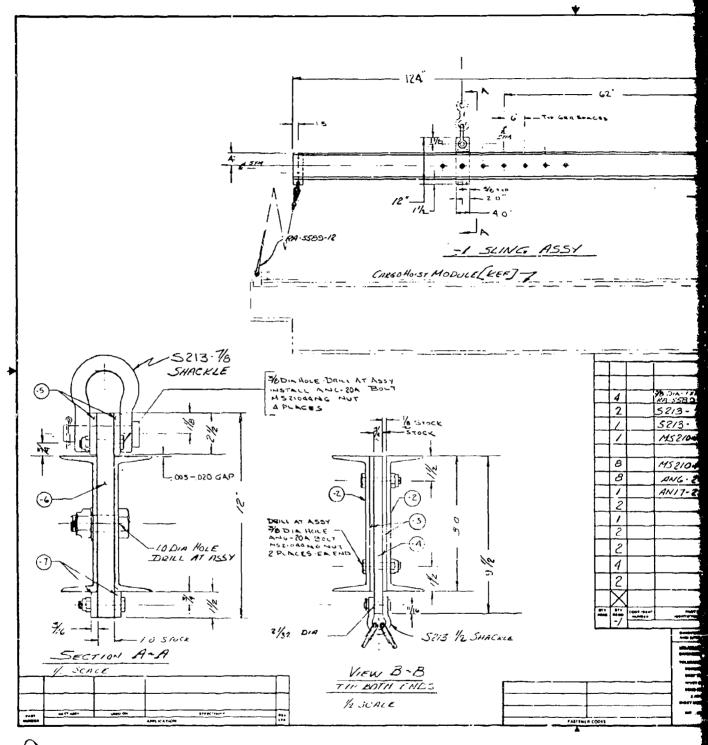
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Figure 66. Lifting Sling, Hoist/Module:

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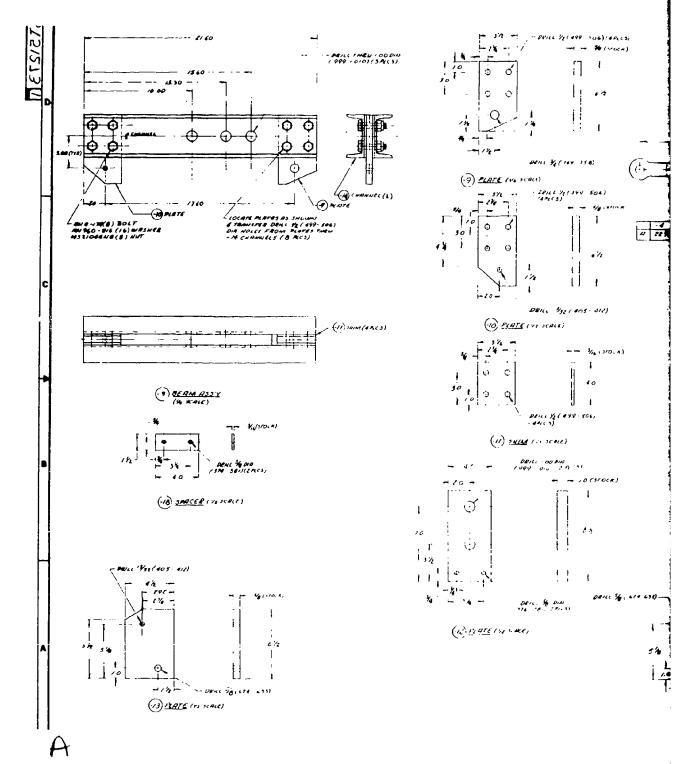
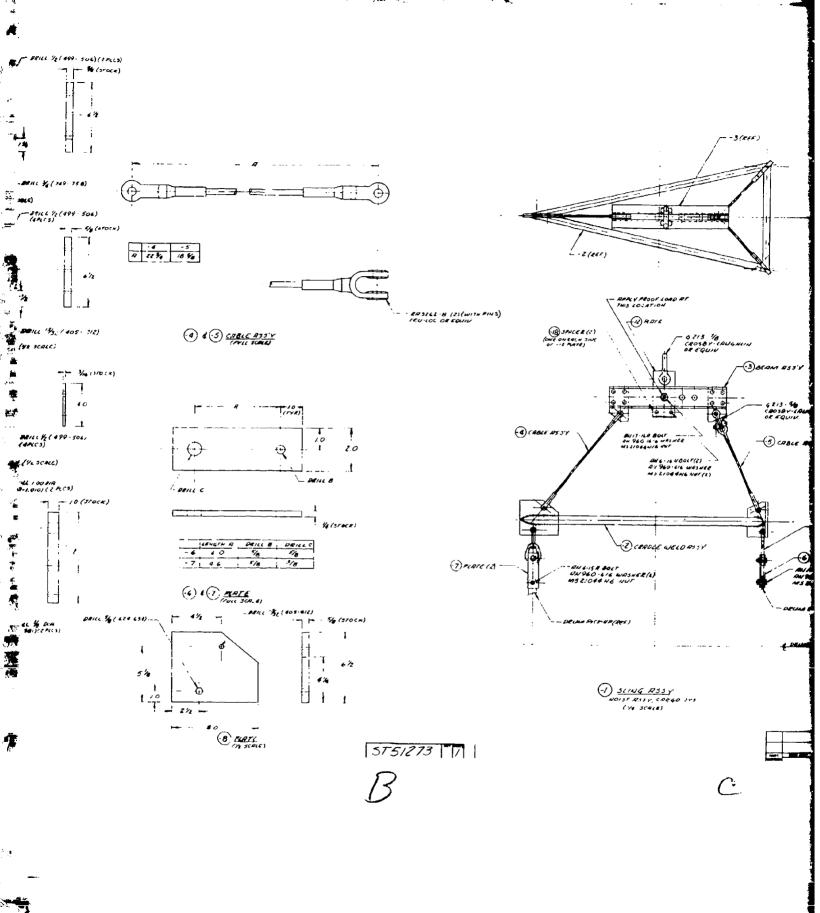
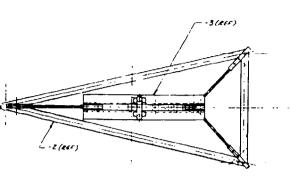
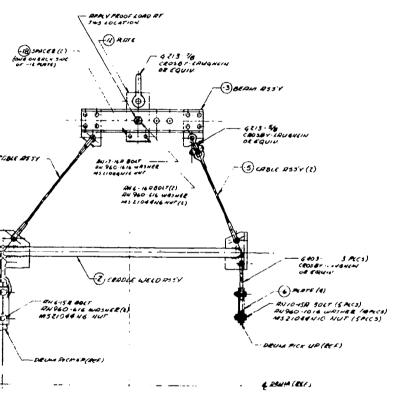


Figure 67. Hoist Lifting Fixture (Sheet 1).

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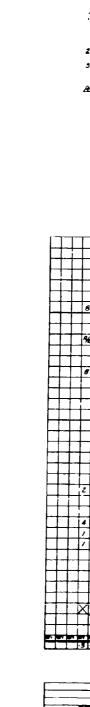


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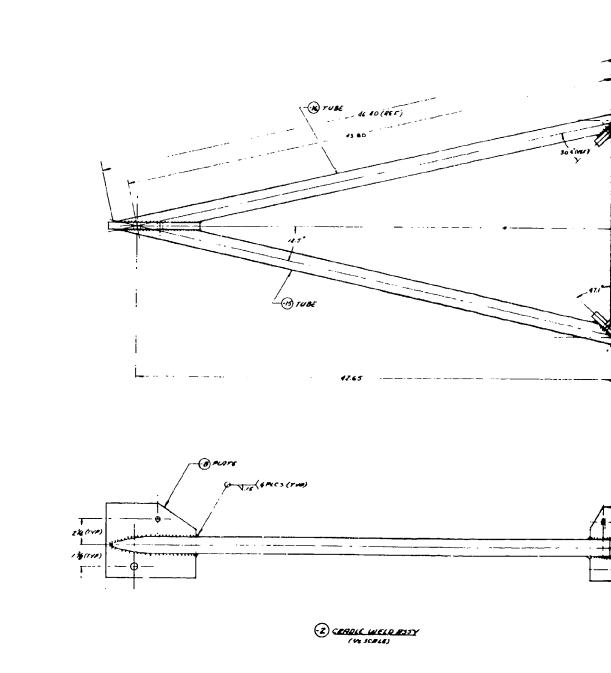
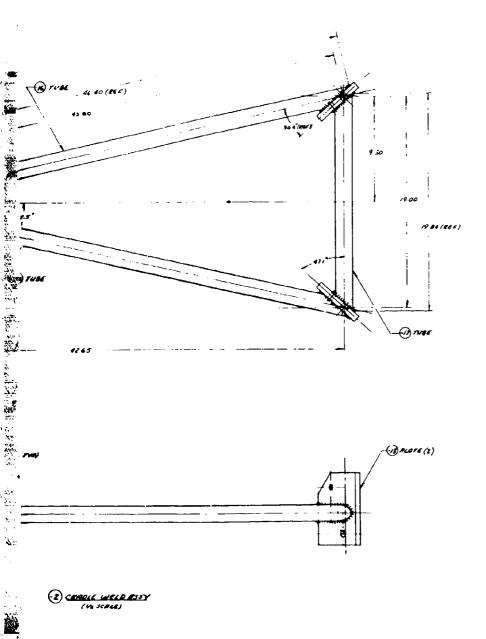


Figure 67. Hoist Lifting Fixture (Sheet 2).

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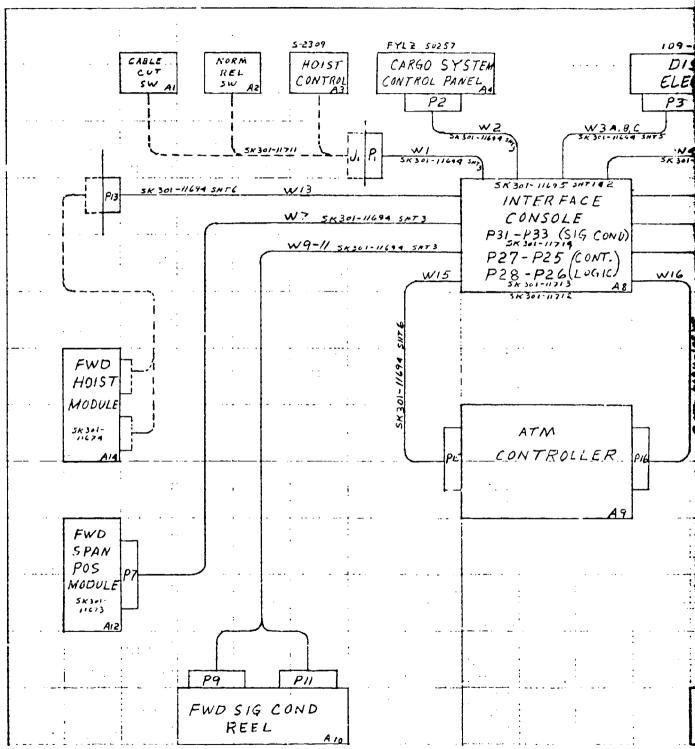
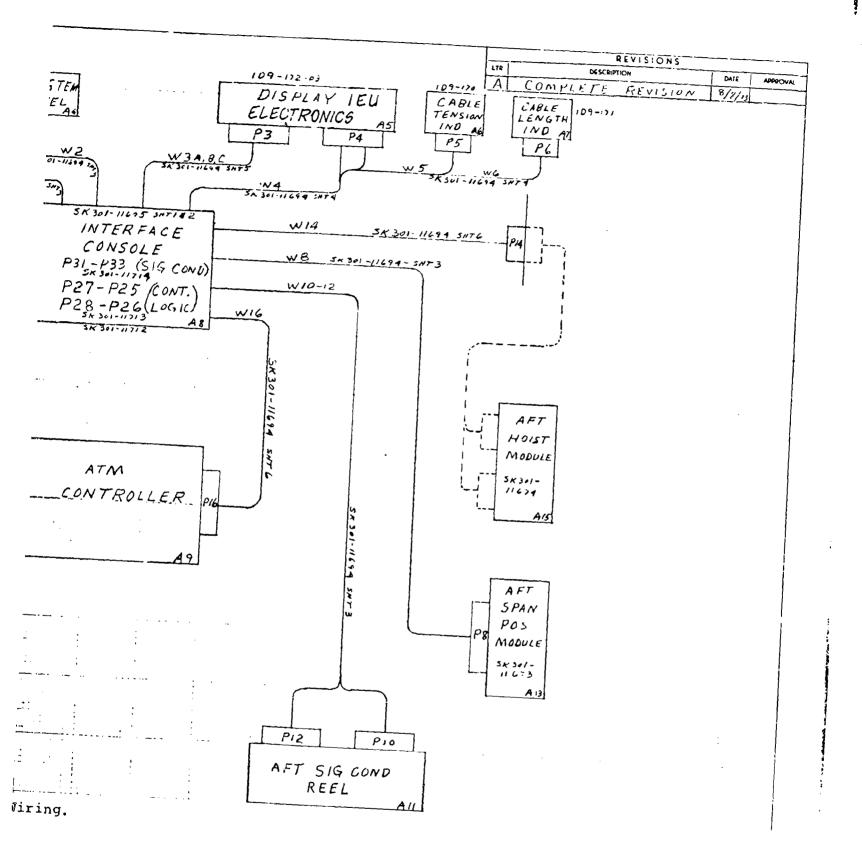


Figure 68. Integrated Test Rig - System Wiring.



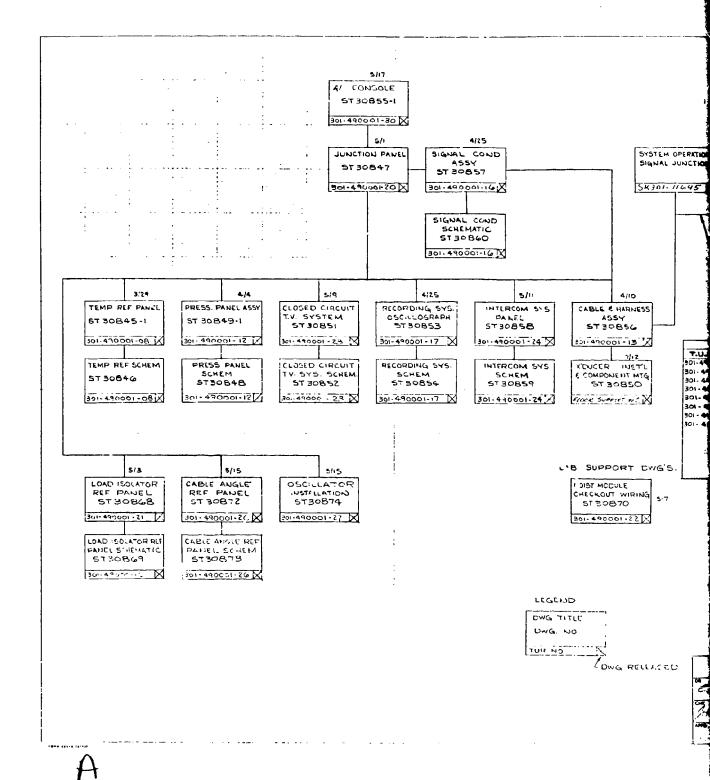
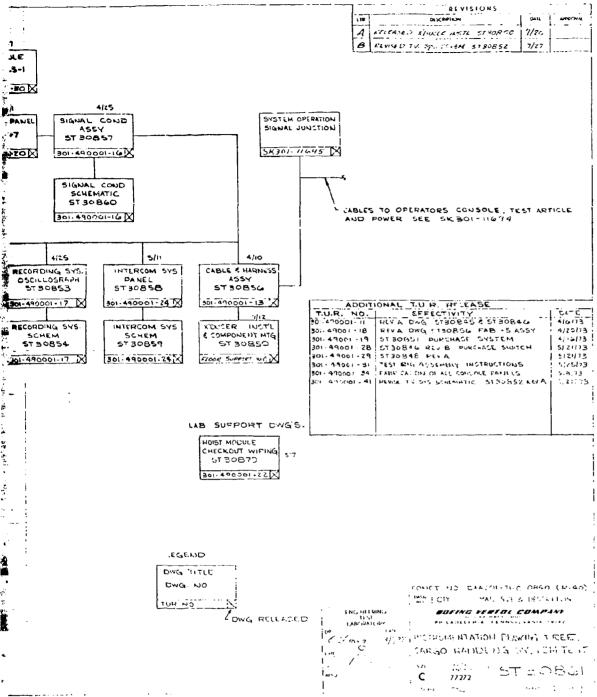


Figure 69. Instrumentation Drawing Tree.



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APPENDIX III PNEUMATIC POWER GENERATOR - STARTING AND OPERATING PROCEDURE

PRERUN CHECKOUT

- 1. Remove all covers from unit:
 - a. Unit covers
 - b. Exhaust cover
 - c. Bleed valve wooden plug
- 2. Turn on water for oil cooler.
- 3. Turn on hand shutoff valve in fuel line at 275-gal. tank.
- 4. Check all switches on control panel for "OFF" position.
- 5. Attach all electrical connectors:
 - a. 2 control lines at engine
 - b. 3 thermo couple leads at engine
 - c. 2 control lines at console
 - d. 3 thermo couple leads at console
 - e. 110-VAC line at console
 - f. 14-VDC connector from batteries to APU
 - q. 28-VDC connectors from batteries to APU

STARTING PROCEDURE - CONTROL CONSOLE

- 1. Turn power "ON".
- 2. Turn fuel valve "ON".
- 3. Turn fuel pump "ON" (requires 10 seconds to pressurize line).
- 4. Turn starter and ignition "ON".
- 5. When gas producer speed reaches 13-15%, increase throttle control with momentary jog. Observe engine starting limits.
- 6. When gas producer speed reaches 52-58%, turn "OFF" starter and ignition. Allow engine to warm up at idle for 1 minute.
- 7. Increase throttle control to "full speed" with momentary jog. Observe engine continuous run limits.

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STOP PROCEDURE

- Decrease throttle control to idle position with momentary jog. Run for 2 minutes at "idle" position.
- Decrease throttle control to "stop" position.
- 3. When gas producer speed is less than 15%, turn "OFF" fuel pump and fuel valve.
- 4. Turn "OFF" power.

POST-RUN SHUTTOWN

- 1. Turn "OFF" water to oil cooler.
- 2. Turn "OFF" hand fuel valve at 275-gal. tank.
- 3. Disconnect electrical connectors:
 - a. 110-VAC at console
 - b. 14-VDC from batteries to APU
 - c. 28-VDC from batteries to APU
- 4. Install covers after unit has cooled:
 - a. Exhaust covers
 - b Bleed valve wooden plug
 - c. Unit cover

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TABLE XII. N	ORMAL PPG OPERATING	LIMITS.
Parameter	Run Condition	Range or Limit*
T _{T5}	GI	700°-900°F
T _T 5	Max Continuous	1430°F
Nl	Start	12%-15%
Nl	GI	59%-65%
Nl	Max Continuous	104%
N2	GI	74.8%-104.7%
N2	Max Continuous	103.8%
P Turb.GB	GI	50-130 psi
P Turb.GB	Max Continuous	115-130 psi
T Turb.GB	GI-Max Continuous	130-225°F
P Compressor GB	GI	90 psi
P Compressor GB	Max Continuous	110 psi
T Compressor GB	Max Continuous	225°F

^{*}Values are for S.L. Standard operation. For information on transient limitations, consult the GMC Allison Div. 250-C20 Engine Operation and Maintenance Manual.

APPENDIX IV INSTRUMENTATION CALIBRATION PROCEDURE

The following represents the basic calibration steps carried out before and after daily hoist system operation.

TEMPERATURES

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STATE OF BUILDING

Operating Mode	Power Source	Osc A	Chanl. Selection
Cal. Normal	R.C. On	.25 ips	1 thru 7; then zero Zero
PRESSURES			

Operating Mode	Input	Osc A	Cal. Pushbutton
Cal. Normal	Depress Cal.	.25 ips	1-5 depress each

LOAD ISOLATORS

Operating Mode	Input	Indication	Adjust
Cal.	Depress sense	30.0	Adjust each pot

CABLE ANGLE

Operating Mode	Input	Read Voltage At	Adjust
Normal	.25V/DEC Fwd/Pitch Fwd/Roll Aft/Pitch Aft/Foll	CTB4 Pins 4&5	Adj.each pot for panel reading in accordance with CTB4 voltage reading

CABLE PAYOUT SPEED

Operating Mode	Input	Apply To	Adjust
Normal	5.0VDC=120 FPM	CTB3 Pins 1&2	Adj.Al gain for 2.0" trace deflection
		CTB3 Pins 4&5	Adj.A2 gain for 2.0" trace deflection

CABLE PAYOUT LENGTH

Operating Mode	Input	Apply To	Adjust
Normal	5.0VDC=100 Ft	CTB4 Pins 14&15	Adj.A5 gain for 2.0" deflection
		CTB4 Pins 18&19	Adj.A6 gain for 2.0" deflection

SPEED COMMAND CAL.

Operating Mode	Input	Indication	Adjust
Normal	6.0V=100%	2.0" trace deflec	Press fwd then aft control grip thumb switches.

MOD. VALVE CURRENT CAL.

Operating <u>Mode</u>	Input	Apply At			Adjust.
Normal	1.5VDC-100%	СТВ3	Pins	24&25	A3 gain for deflection
		СТВ3	Pins	9&10	A7 gain for deflection

* 7 S. GOVERNMENT PRINTING OFFICE 1979 BRIGHTERS